EXPERIMENTAL EVALUATION OF EFFECTIVE FRICTION COEFFICIENT FOR LIQUID RING SEALS

by

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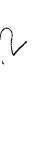
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(ABSTRACT)

Rotor dynamic analysis of liquid ring seals depends upon the correct specification of seal dynamic stiffness and damping characteristics. These are in turn dependent upon several parameters, including the friction holding force between the sealing face and the mating retaining ring. Designers currently assume a value for effective friction coefficient in order to utilize methods for prediction of response and stability. This thesis presents the results of testing on twelve actual seal rings of varying configuration at pressures of 689, 1378, 2068, and 2757 kPa in a static seal test rig to experimentally determine values of effective friction coefficient. The results are presented in graphical form as average effective friction coefficient versus eccentricity ratio for forward and backward motion of the rings.

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TABLE OF CONTENTS

INTRODUCTION	1
BACKGROUND	3
2.1 Application and Operation	3
2.2 Analysis	5
TEST FACILITY	1
3.1 Oil Console and Test Rig 1	1
3.2 Seal Rings	7
3.3 Initial Testing	1
PROCEDURE	3
4.1 Set-up	3
4.2 Data Acquisition	4
4.3 Calculation of Effective Friction Coefficient	5

RESULTS
5.1 Friction Coefficient Results
5.2 Pressure Data
CONCLUSIONS
RECOMMENDATIONS 58
REFERENCES
APPENDIX A: Calculation of β_F
APPENDIX B: Friction Coefficient Data for Individual Rings 63
APPENDIX C: Pressure Data for Individual Rings
APPENDIX D: Instrumentation Calibrations
APPENDIX E: Sample Calculation of Reynolds' Numbers
VITA
v

LIST OF FIGURES

1.	Cross Section of Multi-Ring Oil Seal 4
2.	Forces on Inner Seal Ring 6
3.	Forces on Outer Seal Ring 6
4.	Static Seal Test Rig and Oil Console
5.	Static Seal Test Rig, Cover Removed
6.	Static Seal Test Rig with Cover Removed, Top View
7.	Seal Ring Mounted in Test Rig
8.	Seal Ring
9.	Seal Ring
10.	Effective Friction Coefficient versus Eccentricity Ratio for Rings 1,2, and 3 at 689 kPa
11.	Effective Friction Coefficient versus Eccentricity Ratio for Rings 4,11, and 12 at 689 kPa
12.	Effective Friction Coefficient versus Eccentricity Ratio for Rings 5,6, and 7 at 689 kPa
13.	Effective Friction Coefficient versus Eccentricity Ratio for Rings 8,9, and 10 at 689 kPa
14.	Effective Friction Coefficient versus Eccentricity Ratio for Ring 1 at 1378, 2068, and 2757 kPa
15.	Effective Friction Coefficient versus Eccentricity Ratio for Ring 2 at 1378, 2068, and 2757 kPa

16.	Effective Friction Coefficient versus Eccentricity Ratio for Ring 3 at 1378, 2068, and 2757 kPa
17.	Effective Friction Coefficient versus Eccentricity Ratio for Ring 4 at 1378, 2068, and 2757 kPa
18.	Effective Friction Coefficient versus Eccentricity Ratio for Ring 5 at 1378, 2068, and 2757 kPa
19.	Effective Friction Coefficient versus Eccentricity Ratio for Ring 6 at 1378, 2068, and 2757 kPa
20.	Effective Friction Coefficient versus Eccentricity Ratio for Ring 7 at 1378, 2068, and 2757 kPa
21.	Effective Friction Coefficient versus Eccentricity Ratio for Ring 8 at 689, 1378, and 2068 kPa
22.	Effective Friction Coefficient versus Eccentricity Ratio for Ring 9 at 1378, 2068, and 2757 kPa
23.	Effective Friction Coefficient versus Eccentricity Ratio for Ring 10 at 1378, 2068, and 2757 kPa
24.	Effective Friction Coefficient versus Eccentricity Ratio for Ring 11 at 1378, 2068, and 2757 kPa
25.	Effective Friction Coefficient versus Eccentricity Ratio for Ring 12 at 1378, 2068, and 2757 kPa
26.	Cross Section of Ideal Seal Ring Orientation
27.	Cross Section of Seal Ring with Non-perpendicularity or Shaft Slope
28.	Pressure Data Averages for Ring 1 at Zero Eccentricity Ratio
29.	Pressure Data Averages for Ring 2 at Zero Eccentricity Ratio

30.	Pressure Data Averages for Ring 4 at Zero Eccentricity Ratio
31.	Pressure Data Averages for Ring 6 at Zero Eccentricity Ratio
32.	Pressure Data Averages for Ring 8 at Zero Eccentricity Ratio
33.	Pressure Data Averages for Ring 9 at Zero Eccentricity Ratio
34.	Pressure Data Averages for Ring 10 at Zero Eccentricity Ratio
35.	Pressure Data Averages for Ring 11 at Zero Eccentricity Ratio
36.	Diagram of Seal Ring Indicating Pressure Probe Locations

LIST OF TABLES

1.	Details of Ring Seal Geometry
2.	Ring Bore-to-Sealing Face Non-perpendicularity
3.	Calculated Axial Loads
4 - 50.	Friction Coefficient Averages and Standard Deviations for Rings at 689, 1378, 2068, and 2757 kPa
51 - 85.	Pressure Data for Rings at 689, 1378, 2068, and 2757 kPa
86.	Pressure Gauge Calibration
87.	Pressure Probe Calibrations
88.	Load Cell Calibration (+X Direction)
89.	Load Cell Calibration (-X Direction)

Nomenclature

В	= exponent for pressure influence on viscosity, cm^2/N
с	= radial clearance, cm
D _i	= seal inside lip diameter, cm
D _o	= seal outside lip diameter, cm
Ds	= seal bore diameter, cm
\vec{F}_N	= axial thrust load, N
F _s	= spring force, N
F _x ,F _y	= external seal ring loading, N
F _μ	= friction load, N
g	= acceleration due to gravity = 981 cm/sec^2
h	= film thickness, cm
Ms	= seal mass, kg
Po	= drain pressure, N/cm^2
P _s	= suction pressure, N/cm^2
ΔP	= pressure drop, N/cm^2
R	= shaft radius, cm
Rez	= axial Reynolds' number = $\frac{2VC}{v}$, dim
Re _θ	= circumferential Reynolds' number = $\frac{R\omega c}{v}$, dim
v	= axial velocity, cm/sec

β = seal lappe	face loading	factor, dim
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- ϵ = eccentricity ratio, dim
- μ = oil viscosity, N-sec/cm²
- $\mu_{\rm f}$ = coefficient of friction, dim
- v = kinematic viscosity, cm²/sec
- ω = rotor speed, sec⁻¹

Subscripts

i	= inner seal
0	= outer seal
R	= rotor
S	= seal
x	= in x direction
у	= in y direction
Z	= in axial direction

SI/English Engineering Equivalents:

Note: The data for this research was measured in English Engineering Units and then converted to SI Units to conform to the current policies for thesis writing. The following is a list of the equivalent converted measurements.

689 kPa	= 100 psi
725 kPa	= 105 psi
1378 kPa	= 200 psi
2068 kPa	= 300 psi
2757 kPa	= 400 psi
7600 kPa	= 1100 psi
52 °C	= 125 °F

Chapter 1

INTRODUCTION

Prediction of turbomachinery response and stability is an essential element of turbomachinery design and for improving the operation of existing equipment. Typical rotor analysis includes modeling of the rotor, consideration of bearing characteristics, and incorporation of the influence of the support structure. For turbomachinery that require floating ring oil seals, additional items of concern are how to correctly model these seals and the determination of their influence on response and stability. It has been noted [1] that "some of the major potential sources of destabilizing forces (in turbomachinery) are the seals in the machinery that have been responsible for the advances in ... efficiency."

Analysis of liquid ring seals depends upon the correct specification of seal dynamic stiffness and damping. These characteristics are functions of several parameters including the effective coefficient of friction between the sealing surfaces. The actual friction coefficient in operating machinery is not known. In order to utilize current methods for prediction of response and stability, designers assume a value of effective friction coefficient between 0.10 and 0.15. This range corresponds to typical dynamic

friction factors for steel on steel. "The proper evaluation of the frictional force magnitude, direction, and sense for given dynamic conditions must be made to calculate the stick-slip motion of the ring seal"[2]. This thesis presents the results of testing on actual seal rings to experimentally determine values of effective friction coefficients for liquid ring seals under operating conditions. It is hoped that the results of this research will allow the turbomachinery designer to better evaluate radial loads on liquid ring seals for prediction of seal performance.

Chapter 2

BACKGROUND

This chapter describes the general operation of liquid ring seals and provides a brief discussion of the analysis of the seal rings as it pertains to the determination of effective friction coefficient from forces on the rings.

2.1 Application and Operation

Liquid ring seals are used in many industrial pump and compressor applications where a positive control of the process gas is required with minimal leakage of the sealing liquid into the process gas. A typical arrangement consists of one inner seal ring and one, or more, outer seal rings stacked axially on a turbomachine shaft. Springs are used to aid in assembly and initial pressurization of the seal. The seal is created by forcing a liquid, generally oil, through the clearance between the ring bore and the shaft. Figure 1 shows a cross section of a typical double breakdown oil seal with one inner and two outer seal rings. The oil seal pressure is maintained at suction (P_s) plus an increment

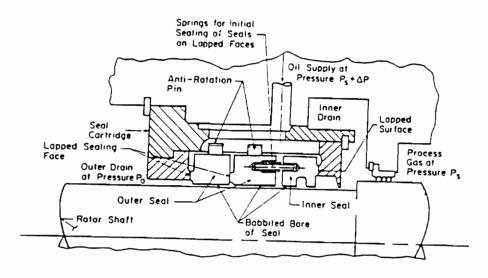


Figure 1: Cross Section of Multi-ring Oil Seal

 ΔP which is typically generated by overhead oil supply tanks. The inner seal thus has a drop ΔP , whereas the outer seal has a drop from $P_s + \Delta P$ to the outer drain pressure, P_o [3]. The rings are floating bearing elements and generate full hydrodynamic fluid film reaction when they lock up against the mating retaining plate. Rotation of the outer rings is generally prevented by an anti-rotation pin. The loading on the ring due to the pin is much smaller in magnitude compared to other forces and is usually neglected for analysis purposes.

2.2 Analysis

It has been shown in the literature [3,4,5] that seal dynamic stiffness and damping can be calculated if the operating eccentricity, average film temperature, and pressure distribution are known. More recent analysis considers the influence of temperature gradients on the seal dynamic characteristics [6]. An assumed value for effective friction coefficient is used for this analysis, because the actual friction coefficient is not known. Improved calculations require experimentally determined values of friction coefficient.

As shown by Kirk [5], the friction holding force is related to the axial load on a ring by means of a force balance. Figures 2 and 3 show the forces on the inner and outer seal rings considered in this analysis, which is repeated here:

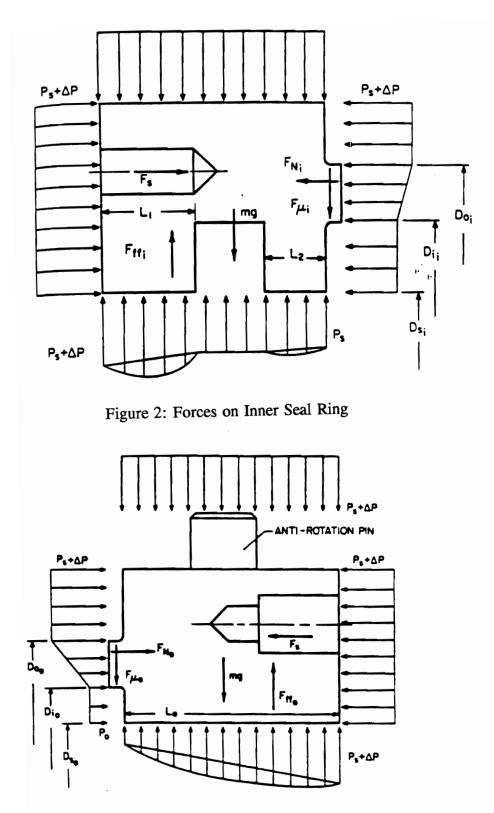


Figure 3: Forces on Outer Seal Ring

The pressure drop per outer seal ring is typically on the order of 130-260 N/cm^2 with the incremental pressure ΔP adjusted to be approximately 4.8 N/cm^2 . Neglecting entrance effects on the axial flow, the global Reynolds number typically indicates no turbulence corrections are required. For a 260 N/cm^2 drop across a typical seal element, the Reynolds numbers for axial and circumferential flows are:

$$Re_z = \frac{2VC}{v} = 144.3$$
 (1)

$$Re_{\theta} = \frac{RC\omega}{v} = 524.6 \tag{2}$$

The method of solution lends itself to local correction on viscosity for turbulence effects when required. The correction may be incorporated into the solution in terms of a modified local viscosity. The local Reynolds number is expressed as

$$Re_{L} = \frac{\rho Rh\omega}{\mu}$$
(3)

The modified local viscosity may then be expressed as

$$\mu_{eff} = j\mu = 0.0139 \, Re^{0.657} \mu \tag{4}$$

with the restriction that $j \ge 1.0$.

The influence of high pressure on the viscosity of the oil must also be accounted for when the pressures are in excess of 345 N/cm². Fuller [7] expresses this factor by the following equation:

$$|\boldsymbol{\mu}_{eff}|_{p} = \boldsymbol{\mu}_{eff} e^{Bp}$$
(5)

For light turbine oil, the value of B is given as $1.37 \times 10^4 \text{ in}^2/\text{lb}$. This value of B gives a 1.07 correction factor for a pressure drop of 345 N/cm² (500 psi).

Considering the inner seal and referring to Figure 2, the force balance can be expressed in terms of the seal geometry and the oil supply pressure level. The relatively small pressure drop across the inner seal produces a low normal loading on that seal ring in the axial direction. A linear pressure gradient $\frac{\partial P}{\partial r}$ is usually assumed across the seal lip interface. The inner seal ring axial thrust load equation, including the spring load, may be expressed as follows:

$$F_{N_i} = F_s + \frac{\pi}{4} (D_{o_i}^2 - D_{i_i}^2) \Delta P \beta_{F_i} + \frac{\pi}{4} (D_{i_i}^2 - D_{s_i}^2) \Delta P$$
(6)

8

The parameter β_F is a function of the condition of the lapped sealing surface. The calculation of β_F for an assumed linear pressure drop across the lapped face is given in the Appendix. The axial thrust load is multiplied by the appropriate coefficient of friction which yields the resisting friction force.

$$F_{\mu_i} = \mu_i F_{N_i} \tag{7}$$

The coefficient of friction is a function of surface finish, pressure loading, and the boundary lubrication conditions at the lapped seal lip-retainer interface. The resultant external seal radial loading is the vector sum of the friction load, which acts in a direction opposite that of the seal velocity, and the inner seal weight. This is expressed as:

$$F_{x_i} = \vec{F}_{\mu_i} \bullet \vec{\eta}_{x_i} \tag{8}$$

$$F_{y_{i}} = \vec{F}_{\mu_{i}} \bullet \vec{\eta}_{y_{i}} - M_{s_{i}}g$$
(9)

where

$$(\vec{\eta}_x, \vec{\eta}_y) = (\frac{-\dot{X}}{\sqrt{\dot{X}^2 + \dot{Y}^2}}, \frac{-\dot{Y}}{\sqrt{\dot{X}^2 + \dot{Y}^2}})$$
 (10)

9

A similar analysis can be performed for the outer seal ring. The pressure drop across this ring is large which gives rise to a high normal loading. The mating surfaces of the seal ring and the retaining ring are finely lapped surfaces which are assumed to give a nearly perfect seal. The pressure between the seal ring lip and the stationary retaining ring is assumed to be linear based on current knowledge of seal dynamic forces. The actual pressure is not known due to the thermal and mechanical distortion of the end plate at design conditions. A factor β_{F_o} , ranging from 0 to 1.0 is therefore specified for a particular design. A force balance yields (see Fig. 3).

$$F_{N_o} = F_s + \left\{ \frac{\pi}{4} \left(D_{o_o}^2 - D_{i_o}^2 \right) \beta_{F_o} + \frac{\pi}{4} \left(D_{i_o}^2 - D_{S_o}^2 \right) \right\} \left(P_s + \Delta P - P_o \right)$$
(11)

The outer seal ring axial thrust load is multiplied by the coefficient of friction which yields the resisting friction force. The resultant external outer seal loading is expressed in component form as follows:

$$F_{x_o} = \vec{F}_{\mu_o} \bullet \vec{\eta}_{x_o}$$
(12)

$$F_{y_o} = \vec{F}_{\mu_o} \bullet \vec{\eta}_{y_o} - M_{s_o}g \tag{13}$$

See Appendix E for a sample calculation for the test rig used in this research.

Chapter 3

TEST FACILITY

The apparatus and instrumentation used in the experiments for this thesis are described. Photographs of the actual facility accompany the description. At the end of this chapter is a discussion of the results of initial testing with the apparatus and how it affected the experiments used for this thesis.

3.1 Oil Console and Test Rig

The facility at Virginia Polytechnic Institute and State University that was used for the research presented in this thesis consists of an industrial high pressure lube-seal oil console, a static seal test rig, twelve outer seal rings, and the instrumentation necessary to measure the parameters of interest. The lube-seal oil console supplies oil to the test rig at pressures from 725 kPa to 7600 kPa. With the exception that the shaft is non-rotating, the static seal test rig models a centrifugal compressor and was designed from actual compressor drawings to conform to industrial specifications on roundness,

runout, and perpendicularity. The rig was designed with a non-rotating shaft in order to eliminate the added complexity of shaft dynamics so that proper operation of the test rig could be verified. Once confidence is gained in the results obtained from the static rig, shaft dynamics can be incorporated for future research. Oil is supplied to the rig by means of a single line. One through-flow line and one by-pass line allow adjustment of rig pressure. Figure 4 shows the test rig with the lube seal oil console in the background. Figure 5 shows the rig with the shaft/end cap removed. Just inside the rig's inner wall is a cylinder with several flow holes to provide an even distribution of the supplied oil. This cylinder can be seen in Figure 6, which shows a top view of the rig with the end cap removed. At the bottom of the rig is the retaining ring that the seal rings mate against during test. A 70.231 mm diameter hole in the center of the retaining ring allows the shaft, which is an integral part of the end cap, to pass through. The rings seal against a raised lapped surface which borders the hole. The rings remain in contact with this surface throughout motion around the clearance circle. The diametral clearance between the shaft and seal ring bores is four mils. Four rods are positioned at ninety degree intervals on the rig's circumference, just above the retaining ring. The rods are threaded through the wall of the rig to allow the seal rings to be "pushed" across the clearance circle during test. Load cells (Omega model # LCG-1K) positioned at the end of the rods measure the force required to move the ring to a given position. Figure 7 shows a seal ring mounted between the push rods inside the rig with load cells in place.

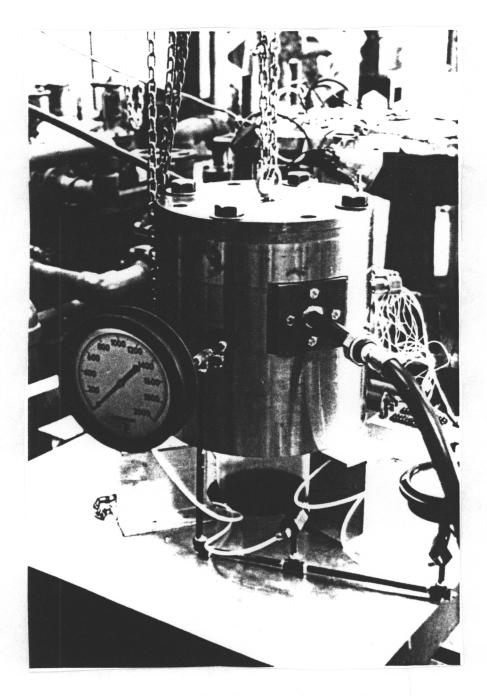


Figure 4: Static Seal Test Rig and Oil Console

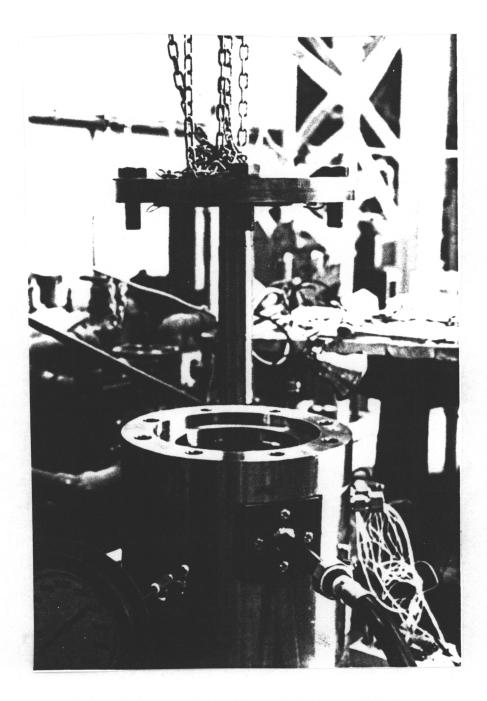


Figure 5: Static Seal Test Rig, Cover Removed

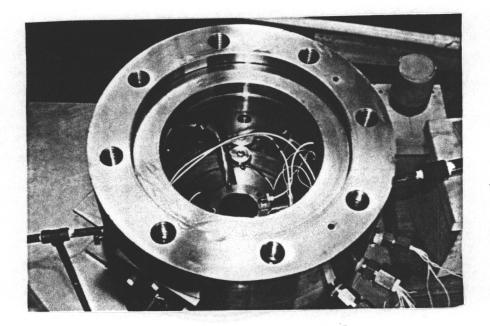


Figure 6: Static Seal Test Rig with Cover Removed, Top View

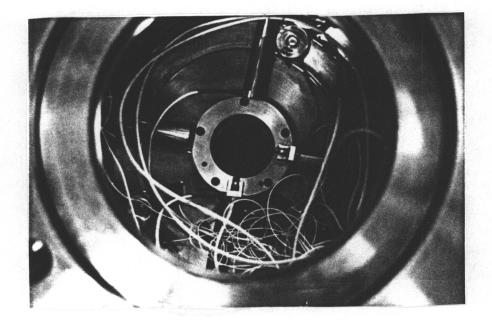


Figure 7: Seal Ring Mounted in Test Rig

3.2 Seal Rings

The seal rings were also made to industrial specifications and were further modified to accommodate the instrumentation required for this research. Twelve outer rings of varying configuration were tested. Table 1 lists the seal rings and indicates the sealing face width and groove configuration for each ring. Tapped holes were added to the seal rings to allow mounting of pressure probes and non-contact displacement probes. The two non-contact displacement probes (Bently-Nevada) are mounted ninety degrees apart on the top of the seal ring, opposite the sealing face. They measure the ring to shaft relative displacement in two directions, defined here as "X" and "Y". Twelve pressure probes (Kulite model # XT-140-500 A) are mounted through the ring wall. The probes are positioned in groups of three, at ninety degree intervals around the ring circumference. In each group, the first probe is located at one quarter of the axial length along the ring. The second probe is located at half of the axial length and the third probe is located at three quarters of the axial length. Within each group, the probes are offset by 18 degrees to allow clearance for the spotface (see Figures 7 and 8). This arrangement allows measurement of pressure in both the axial and circumferential Figure 8 shows a seal ring with a wide sealing face width and no directions. circumferential grooves. The tapped holes for mounting the pressure probes are visible on the sides of the ring. Figure 9 shows a seal ring with a narrow sealing face width and two circumferential grooves.

		Groove Configuration			
Ring No.	Sealing Face OD _F (mm)	Square		Round	
		Deep (mm)	Wide (mm)	Radius (mm)	No. Grooves
1	80.82				0
2	84.35				0
3	78.00			0.787	1
4	78.00	3.175	1.575		1
5	80.82	1.270	3.175		1
6	80.82	1.270	1.575		1
7	80.82	3.175	1.575		1
8	84.35		1.575		1
9	84.35		1.575		1
10	84.35		1.575		1
11	78.00	3.175	1.575		2*
12	78.00		0.787		2*

TABLE 1:Details of Ring Seal Geometry

Overall dimensions of rings (mm): ID = 70.05 - 0.0, +0.0254 OD = 111.125 - 0.102, +0.0 L = 30.15 + 0.0, -0.102Tolerance on sealing face $OD_F = +0.0, -0.127$

*The two grooves are spaced equally over the width of the ring



Figure 8: Seal Ring with No Circumferential Grooves

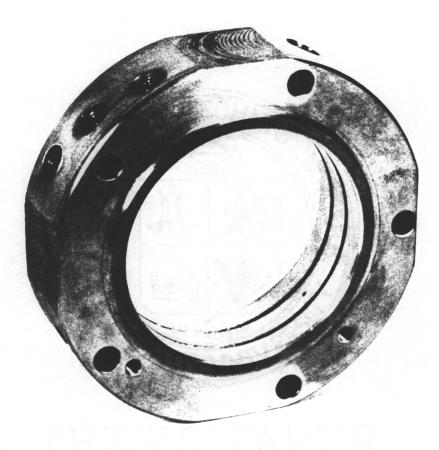


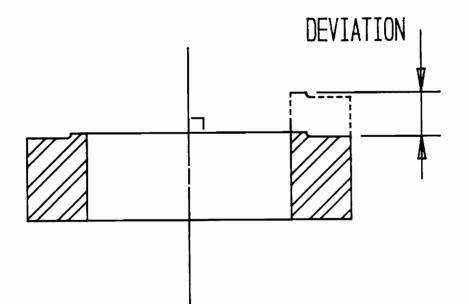
Figure 9: Seal Ring with Two Circumferential Grooves

3.3 Initial Testing

Initial testing with the rig [8,9] indicated that the radial forces on the ring were manufactured though to standard symmetric, even the rig was not turbomachinerytolerances. These unsymmetric forces may be due to minor shaft slope or ring bore-to-sealing face non-perpendicularity. Conventional analysis of these rings neglects the influence of these parameters because of their small length to diameter ratio, which is less than 0.5. A worst case shaft slope of 0.3 mm/m was confirmed by measuring the float clearance variation as a function of shaft clocking relative to the ring. The bore-to-sealing face non-perpendicularity of each ring was measured on a three-axis inspection machine. The maximum deviation from perfect perpendicularity for each ring is given in Table 2. In an effort to minimize the effects of shaft slope and to ensure repeatable testing conditions, results for all twelve rings at 516.7 kPa and different cover angular positions were compared. The end cover was then doweled in a position that showed the least influence of shaft slope in the X-direction for a majority of the rings. Testing of the rings therefore involves measurement of the radial forces in the Xdirection only.

RING NUMBER	NON-PERPENDICULARITY (mils)
1	-0.3
2	0.7
3	0.0
4	-0.4
5	0.3
6	0.3
7	-0.6
8	-0.5
9	0.4
10	-0.2
11	-0.2
12	-0.6

TABLE 2:Ring Bore-to-Sealing Face Non-perpendicularity



22

Chapter 4

PROCEDURE

This chapter describes the procedure used to take data with the test apparatus described in Chapter 3. The types of measurements that were taken are discussed as well as how they were used to determine effective friction coefficients.

4.1 Set-up

Once a ring had been fitted with pressure probes and the non-contact displacement probes, it was placed on the retaining ring inside the test rig. The rig has the capacity to incorporate spring preload, however, it was found that the ring will seal under its own weight in the vertical position and thus no preload was used for these experiments. The shaft and end cap were lowered into position, the end cap was doweled, and the cap bolts were tightened. The rig was then pressurized to 689 kPa with oil from the lube-seal oil console. The oil was forced through the clearance between the shaft and the seal ring bore to a reservoir at atmospheric pressure. Once the oil in the rig reached 52 °C testing

was begun. A solenoid valve was used to control the flow of cooling water and maintain the oil at this temperature. Measurements were taken to locate the centered and extreme eccentric positions of the ring relative to the shaft. The positions of the ring that correspond to ± 0.25 , ± 0.50 , and ± 0.75 eccentricity ratios were then determined.

4.2 Data Acquisition

During test, the ring was moved with the push rods to positions that correspond to the various ratios of eccentricity. One pass consisted of motion from one extreme eccentric position to the opposite extreme eccentric position and back again. The force required to move the ring was recorded at ± 0.25 , ± 0.50 , ± 0.75 eccentricity ratio and at the centered position for each pass. Three passes per a given pressure were recorded to produce the radial load data for each ring. Pressure data was taken at each eccentricity ratio by moving the ring to the required position with the push rods and recording the readings from the various probes. The test procedure was repeated at pressures of 1378, 2068, and 2757 kPa.

4.3 Calculation of Effective Friction Coefficient

Given the seal geometry and the pressure inside the rig, the axial load on a ring can be calculated by the force balance given by Eq. 11. A computer code by Kirk and Reedy [10] that incorporates this force balance was used to evaluate the axial load for each case of sealing face width and pressure tested. Table 3 gives the results of these calculations. Values of effective friction coefficient were calculated from the radial load data by dividing the radial force required to move a ring by the calculated axial load on the ring.

	PRESSURE (kPa)			
Sealing Face Width	689	1378	2068	2757
(mm)	AXIAL LOAD (N)			
3.175 (narrow)	373.1	735.5	1097.7	1459.9
4.760 (medium)	492.2	973.9	1455.2	1936.6
6.350 (wide)	647.0	1283.5	1919.7	2555.9

TABLE 3: Calculated Axial Loads

Chapter 5

RESULTS

This chapter presents the results of the testing done for this research. The friction coefficient results are presented in graphical form as well as tabulated in the appendix. The pressure data was not complete, but is included in the appendix in tabular form. Plots of pressure averages through the seal rings at zero eccentricity ratio are given.

5.1 Friction Coefficient Results

Figures 10 through 25, located at the end of this chapter, present the results of this research. The figures are plots of effective friction coefficient versus eccentricity ratio, for "forward" and "backward" motion of the rings, at pressures of 689, 1378, 2068, and 2757 kPa. Forward motion is defined as motion of the ring such that the displacement sensor, mounted on the ring, is moving toward the shaft. Backward motion is that in which the displacement sensor is moving away from the shaft. The test results show characteristics that are apparent for each ring regardless of configuration or

pressure. The most noticeable characteristic is that there is some force required to center the rings, and once centered, there is an increasing force required to further decenter the rings. The most important part of the results for each ring appears to be the values of effective friction coefficient from a high eccentricity ratio to the centered position for each direction of motion. Since the friction coefficient in the direction of decreasing eccentricity is generally lower than in the direction of increasing eccentricity, a symmetrical external force on a ring would move the ring toward the centered position. The results also indicate a tendency for the rings to exhibit a lower effective friction coefficient in one direction. As stated earlier, this tendency may be due to the nonperpendicularity of the ring bore or to minor shaft slope inherent in the rig itself. Figures 26 and 27 illustrate how minor shaft slope or bore-to-sealing face nonperpendicularity can influence the radial loads on a ring. Figure 26 shows a sketch of the ideal situation of a perfectly centered ring with no shaft slope or nonperpendicularity. In this arrangement the pressures around the ring should be symmetric and thus the ring would see an equal resistance to motion in all directions. Figure 27, however, shows a situation where shaft slope or ring non-perpendicularity is present (exaggerated for clarity). Side A of the ring has more surface area exposed to high pressure. The radial pressure force on this side would facilitate motion in the direction of the arrow, while motion in the opposite direction would be more difficult. This unsymmetrical pressure force will be called the bias force in this thesis.

Since ring 3 has been determined to have perfect perpendicularity (with regard to the inspection machine) the results indicate that the shaft slope is in a direction that facilitates forward motion of the rings. If the effects of non-perpendicularity do not cancel the effects of shaft slope, some bias force will be apparent in the results, indicating the direction of motion that the combined influence favors. This thesis defines the magnitude of this bias force as the difference between forward and backward friction coefficients at zero eccentricity ratio.

Figure 10 shows the results for rings 1,2, and 3 at 689 kPa. These results are presented together to provide some comparison of the effects of different ring configuration. Rings 1,2, and 3 have medium, wide, and narrow sealing face widths, respectively. Rings 1 and 2 have no circumferential grooves, while ring 3 has one circumferential groove. This figure shows that ring 3 has the lowest friction coefficients of the three for forward motion of the ring and for backward motion greater than approximately 0.2 eccentricity ratio.

Figure 11 presents the results for rings 4,11, and 12, all of which have narrow sealing face widths, at 689 kPa. Also note that rings 11 and 12 are the only rings with two circumferential grooves. The results for rings with medium sealing face widths at 689 kPa are given in Figure 12. Rings 5 and 6 are similar in all respects; each has one circumferential groove and a maximum non-perpendicularity of 0.3 mils. As shown in Figure 12 these rings exhibit similar friction coefficient trends. Ring 5, however, shows a lower effective friction coefficient over most of the forward motion, whereas ring 6

shows a lower coefficient over backward motion. This is an example of how the nonperpendicularity has a directional influence on friction coefficient. Even though rings 5 and 6 have the same non-perpendicularity, the directions of the non-perpendicularities are not known. Figure 12 indicates that the non-perpendicularity of ring 5 is in a direction that promotes forward motion of the ring, whereas ring 6 has a nonperpendicularity in a direction that promotes backward motion.

Figure 13 presents the results for rings with wide sealing face widths at 689 kPa. The results for individual rings at 1378, 2068, and 2757 kPa are given in Figures 14 through 25. Tables 4 through 50, in Appendix B show the averaged friction coefficients that were used to create Figures 10 through 25. These tables also show the maximum and minimum friction coefficients from the tests as well as the standard deviations at each eccentricity.

5.2 Pressure Data

Some problems were encountered with the pressure probes during test that could not be resolved in time to be included in this thesis. The data that was obtained with the probes available is given in Appendix C. Included in this appendix is a diagram indicating the circumferential and axial location of the probes. Figures 28 through 35 show plots of averaged pressure data at zero eccentricity ratio for some of the rings. These figures indicate substantial pressure losses at both the entrance and exit sides of the seal rings.



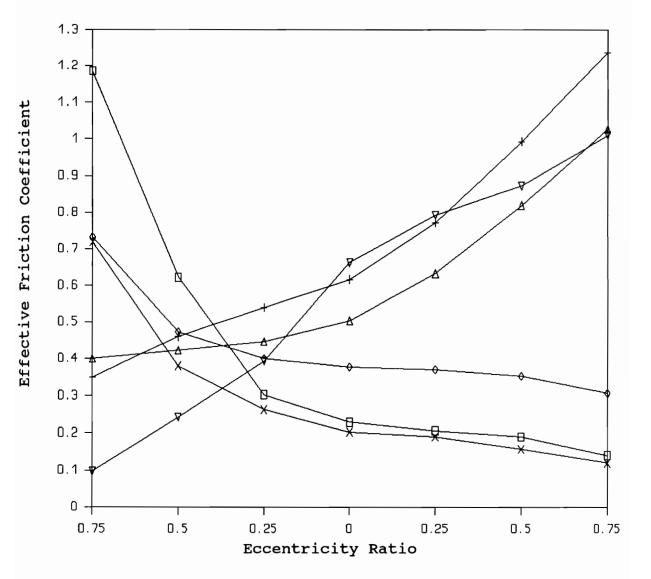


Figure 10: Average Effective Friction Coefficients for Rings 1, 2, and 3 at 689.48 kPa.

LEGEND:				
RING NO.	4	11	12	
FORWARD	D	\diamond	×	
BACKWARD	+	Δ	Δ	

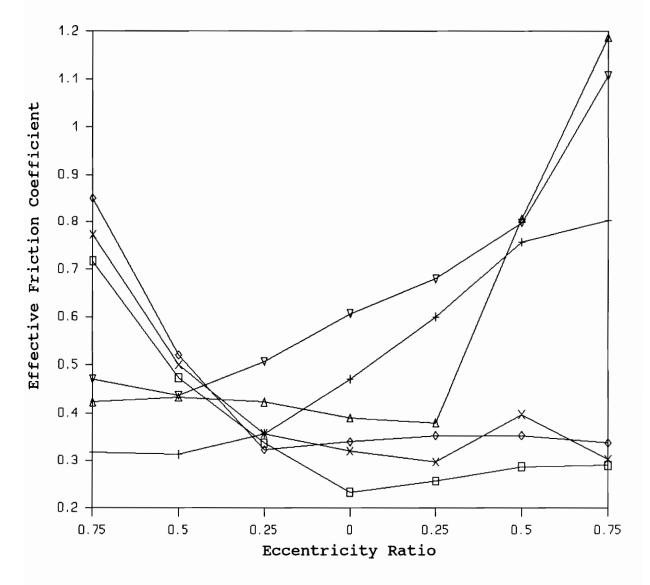


Figure 11: Average Effective Friction Coefficients for Rings with Narrow Sealing Face Widths at 689 kPa.

LEGEND:			
RING NO.	5	6	7
FORWARD	D	\diamond	×
BACKWARD	+	۵	▼

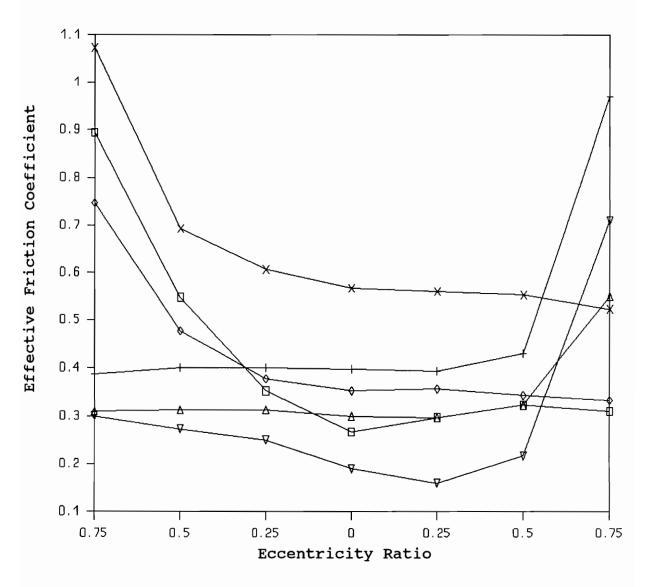


Figure 12: Average Effective Friction Coefficients for Rings with Medium Sealing Face Widths at 689 kPa.

LEGEND:			
RING NO.	8	9	10
FORWARD		\diamond	×
BACKWARD	+	Δ	v

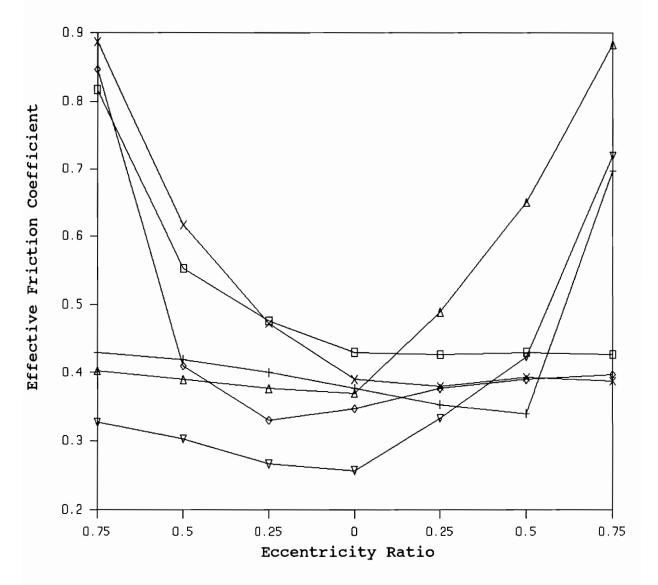


Figure 13: Average Effective Friction Coefficients for Rings with Wide Sealing Face Widths at 689 kPa.

LEGEND:	Ring No.	1	
PRESS. (kPa)	1378	2068	2757
FORWARD		\diamond	×
BACKWARD	+	Δ	♥

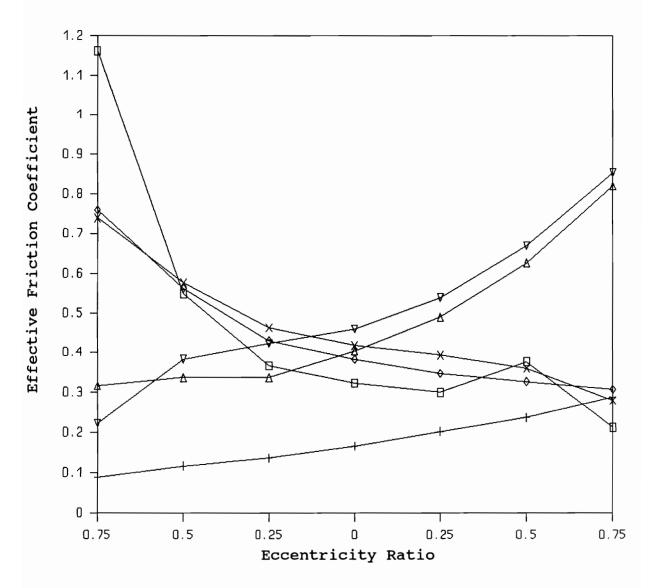
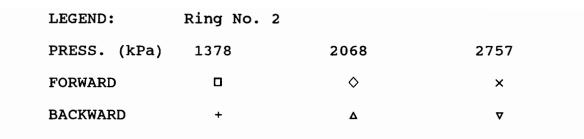


Figure 14: Average Effective Friction Coefficients for Ring 1 at 1378, 2068, and 2757 kPa.



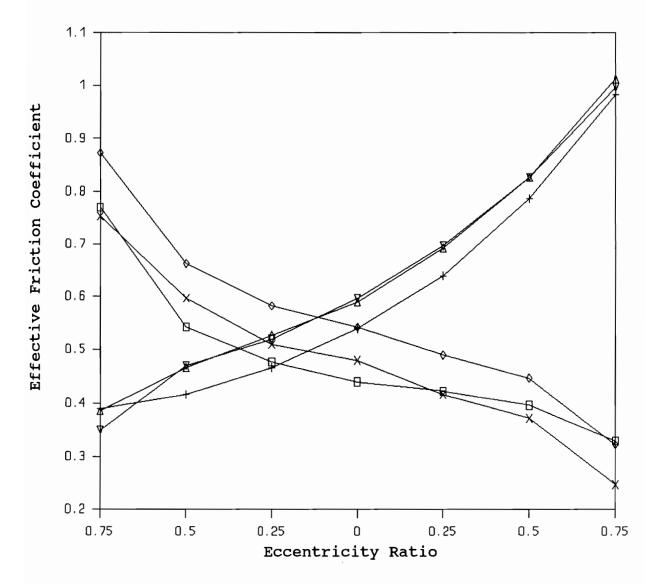


Figure 15: Average Effective Friction Coefficients for Ring 2 at 1378, 2068, and 2757 kPa.

LEGEND:	Ring No.	3	
PRESS. (kPa)	1378	2068	2757
FORWARD		\diamond	×
BACKWARD	+	Δ	♥

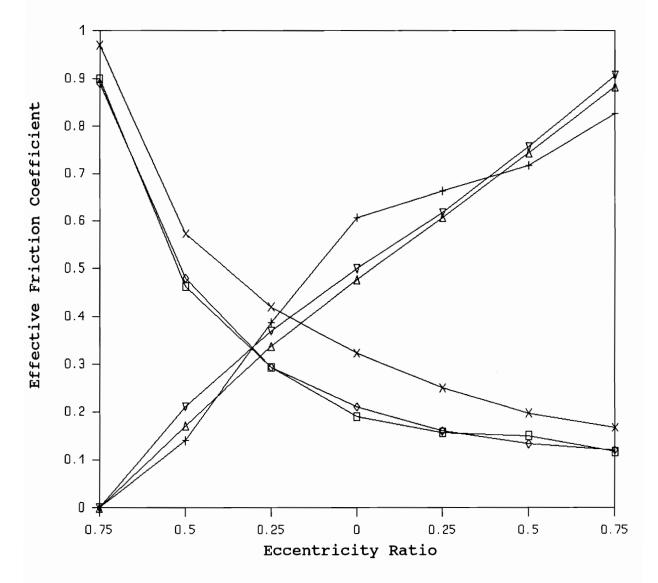


Figure 16: Average Effective Friction Coefficients for Ring 3 at 1378, 2068, and 2757 kPa.

LEGEND:	Ring No. 4		
PRESS. (kPa)	1378	2068	2757
FORWARD		\diamond	×
BACKWARD	+	Δ	▼

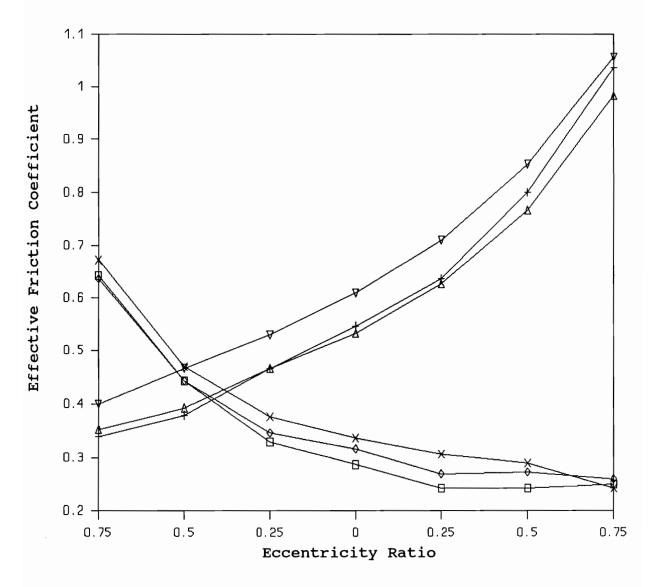
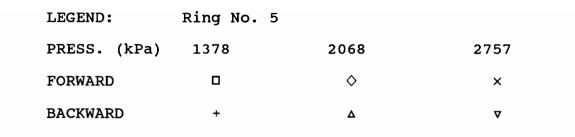


Figure 17: Average Effective Friction Coefficients for Ring 4 at 1378, 2068, and 2757 kPa.



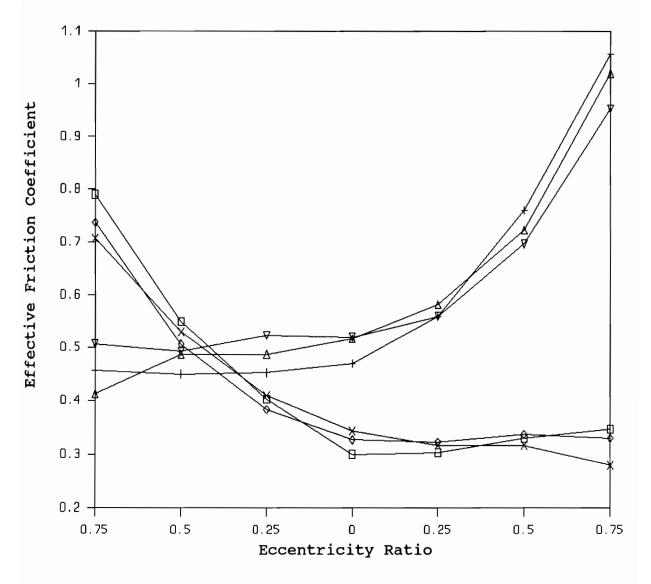


Figure 18: Average Effective Friction Coefficients for Ring 5 at 1378, 2068, and 2757 kPa.

LEGEND:	Ring No. 6	i	
PRESS. (kPa)	1378	2068	2757
FORWARD		\diamond	×
BACKWARD	+	Δ	▼

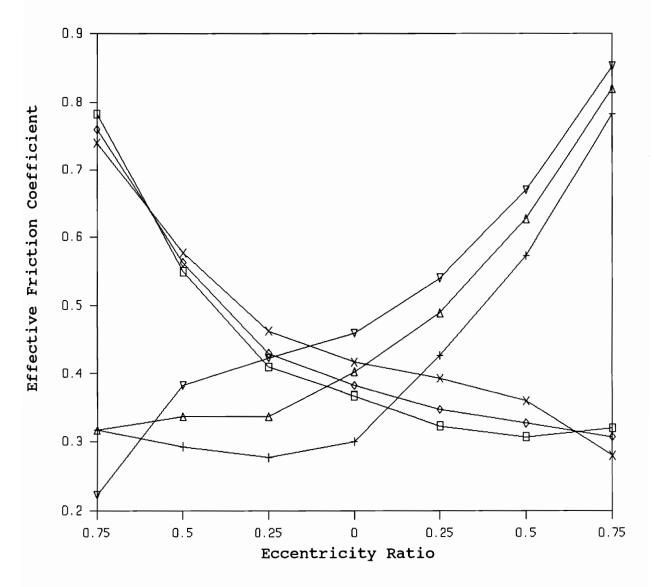
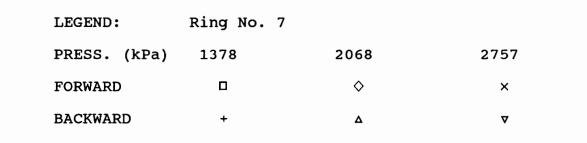


Figure 19: Average Effective Friction Coefficients for Ring 6 at 1378, 2068, and 2757 kPa.



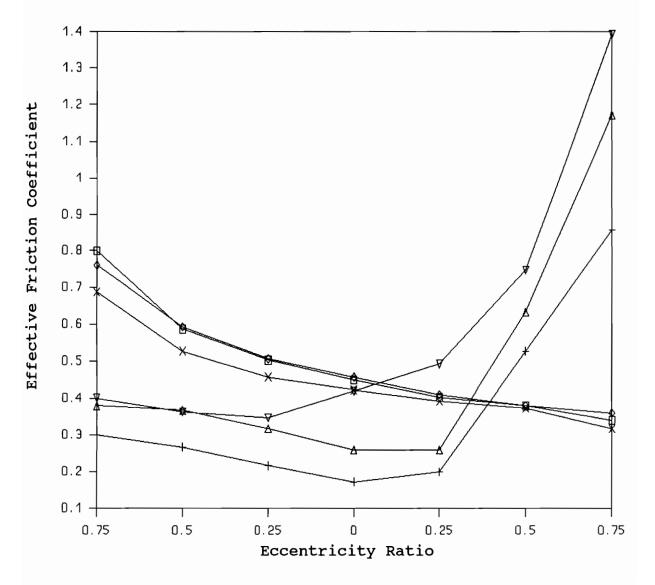


Figure 20: Average Effective Friction Coefficients for Ring 7 at 1378, 2068, and 2757 kPa.

Ring No. 8	В	
689	1378	2068
	\diamond	×
+	Δ	▼
	689	

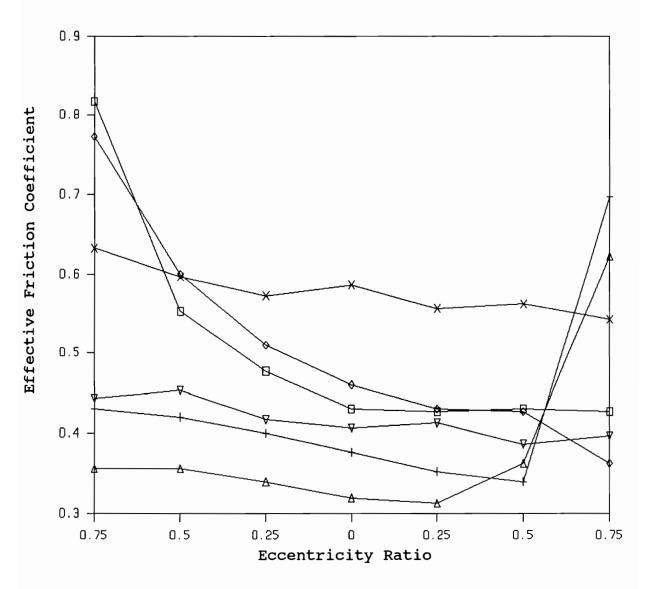


Figure 21: Average Effective Friction Coefficients for Ring 8 at 689, 1378, and 2068 kPa.

LEGEND:	Ring No.	9	
PRESS. (kPa)	1378	2068	2757
FORWARD		\diamond	×
BACKWARD	+	Δ	▼

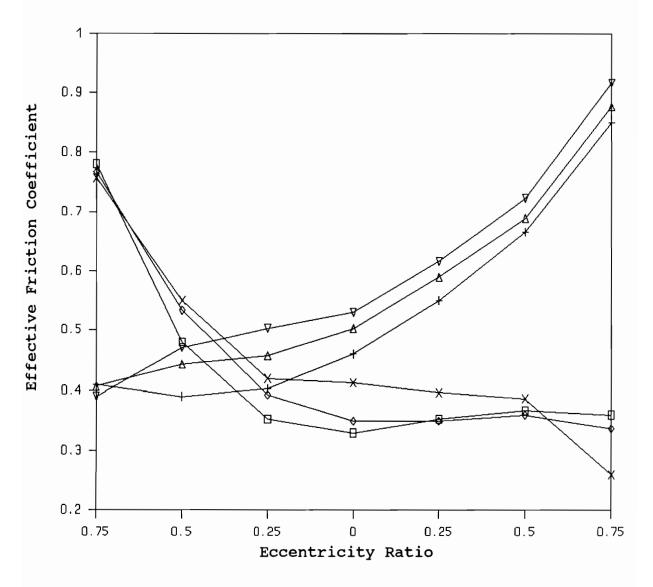


Figure 22: Average Effective Friction Coefficients for Ring 9 at 1378, 2068, and 2757 kPa.

LEGEND:	Ring No. 10			
PRESS. (kPa)	1378	2068	2757	
FORWARD		\diamond	×	
BACKWARD	+	Δ	▼	

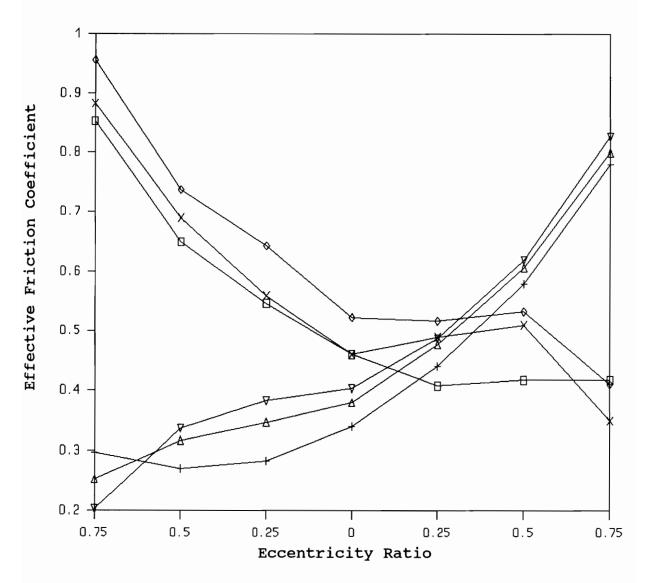


Figure 23: Average Effective Friction Coefficients for Ring 10 at 1378, 2068, and 2757 kPa.

LEGEND:	Ring No.	11	
PRESS. (kPa)	1378	2068	2757
FORWARD		\diamond	×
BACKWARD	+	Δ	▼

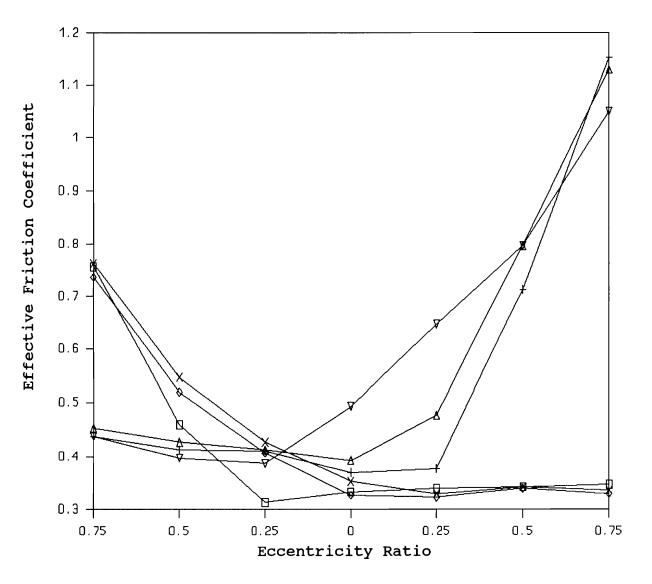


Figure 24: Average Effective Friction Coefficients for Ring 11 at 1378, 2068, and 2757 kPa.

LEGEND:	Ring No.	12	
PRESS. (kPa)	1378	2068	2757
FORWARD		\diamond	×
BACKWARD	+	Δ	▼

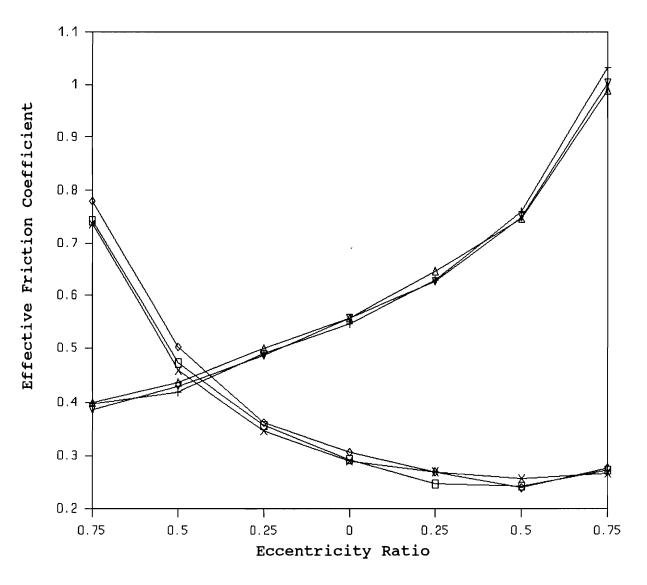


Figure 25: Average Effective Friction Coefficients for Ring 12 at 1378, 2068, and 2757 kPa.

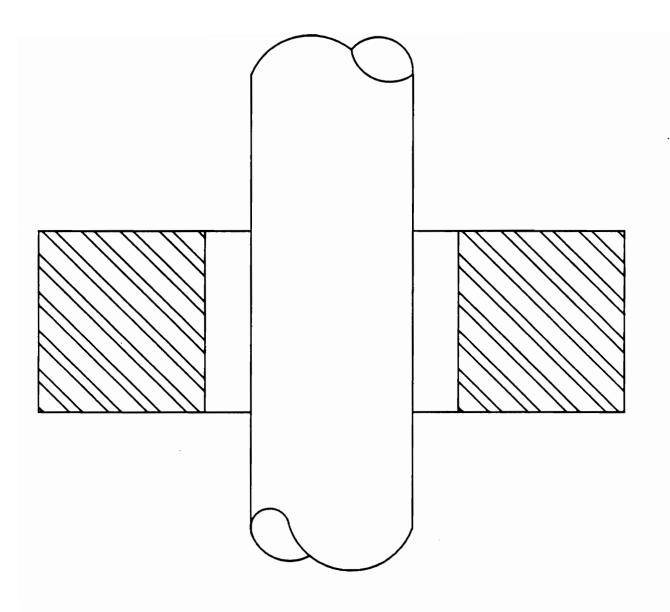


Figure 26: Ideal Seal Ring Orientation

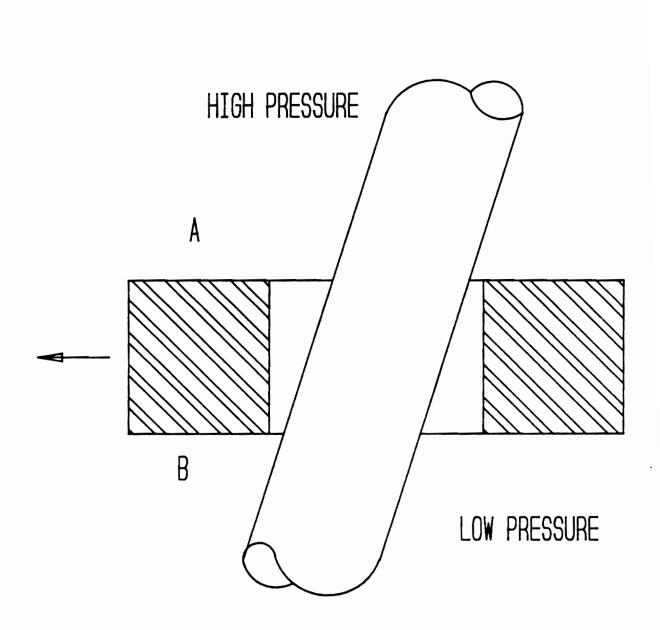
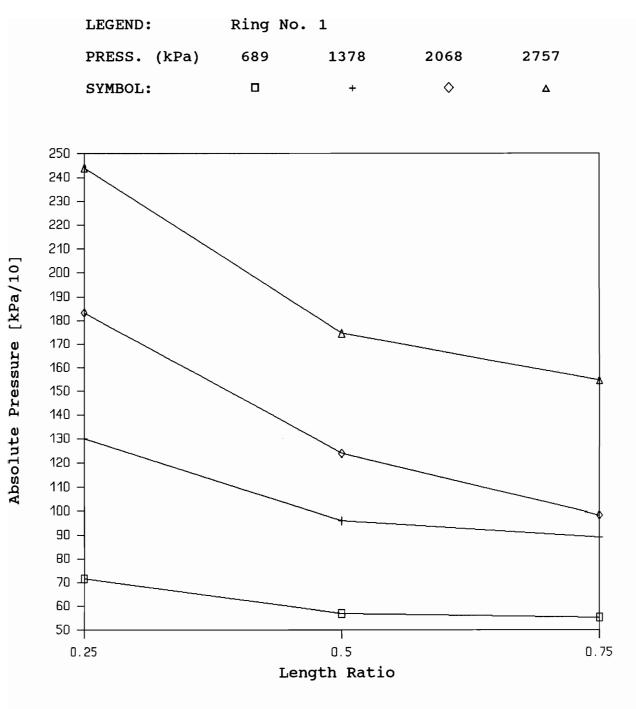
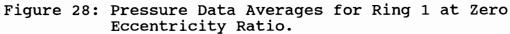
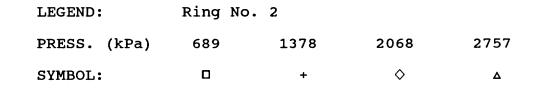


Figure 27: Seal Ring with Non-perpendicularity or Shaft Slope







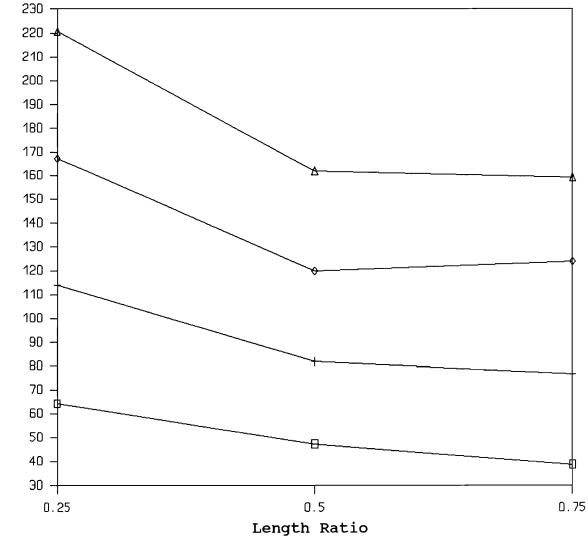


Figure 29: Pressure Data Averages for Ring 2 at Zero Eccentricity Ratio.

Absolute Pressure [kPa/10]

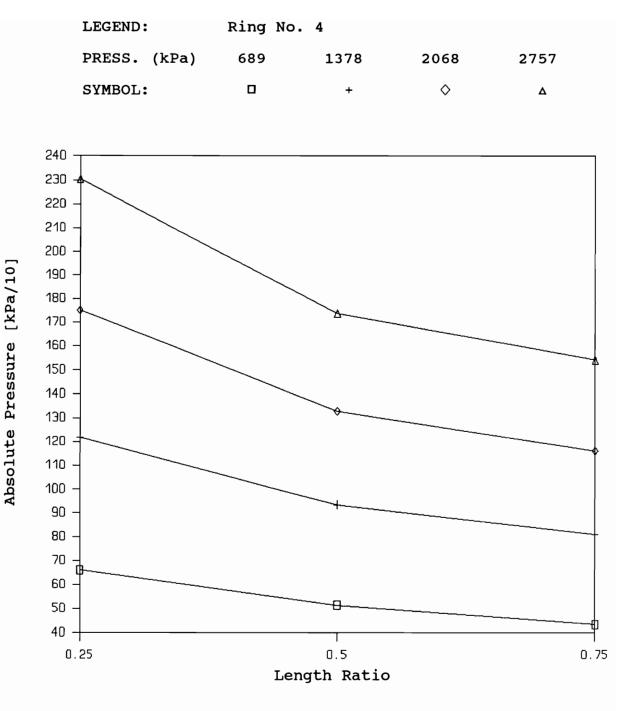
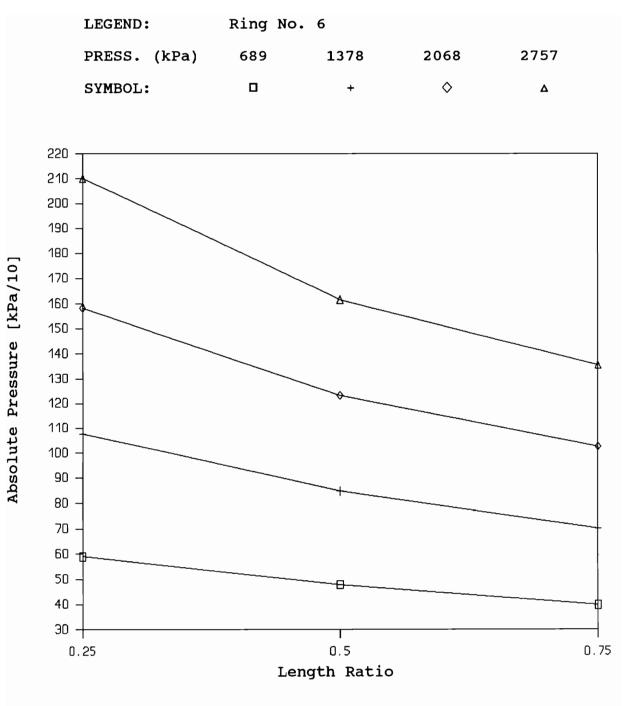
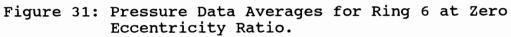
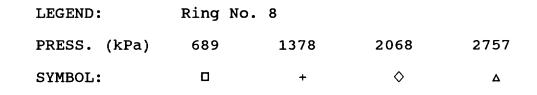


Figure 30: Pressure Data Averages for Ring 4 at Zero Eccentricity Ratio.







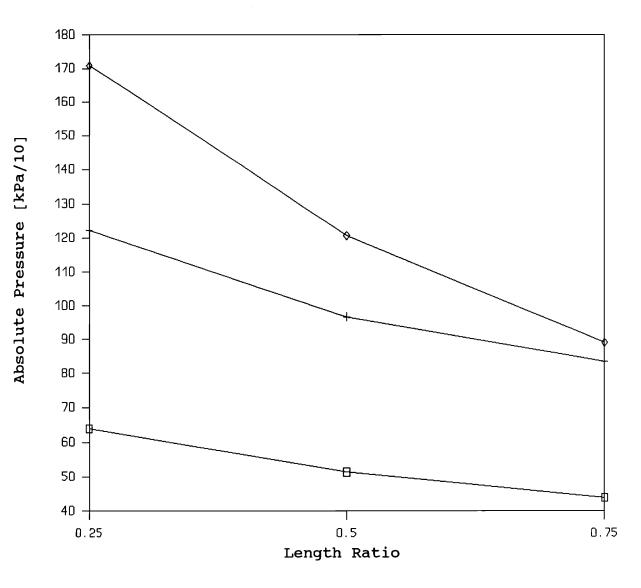


Figure 32: Pressure Data Averages for Ring 8 at Zero Eccentricity Ratio.

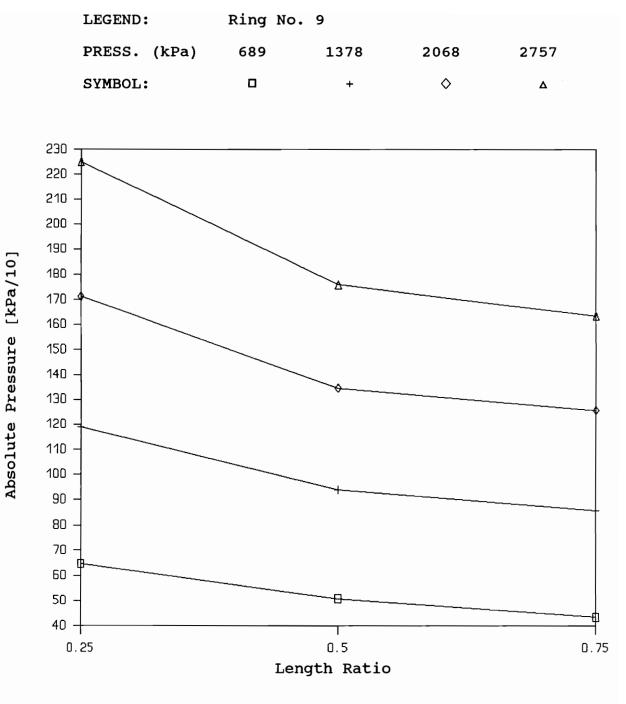


Figure 33: Pressure Data Averages for Ring 9 at Zero Eccentricity Ratio.

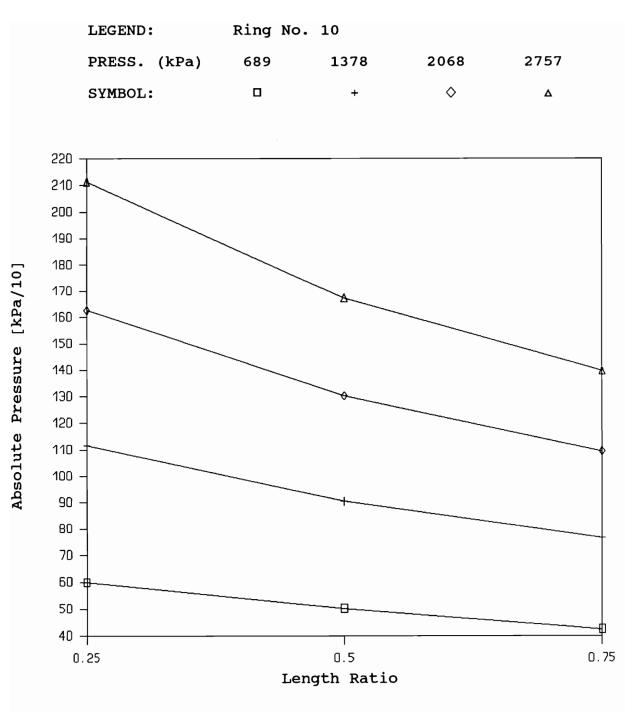
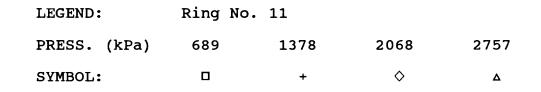


Figure 34: Pressure Data Averages for Ring 10 at Zero Eccentricity Ratio.



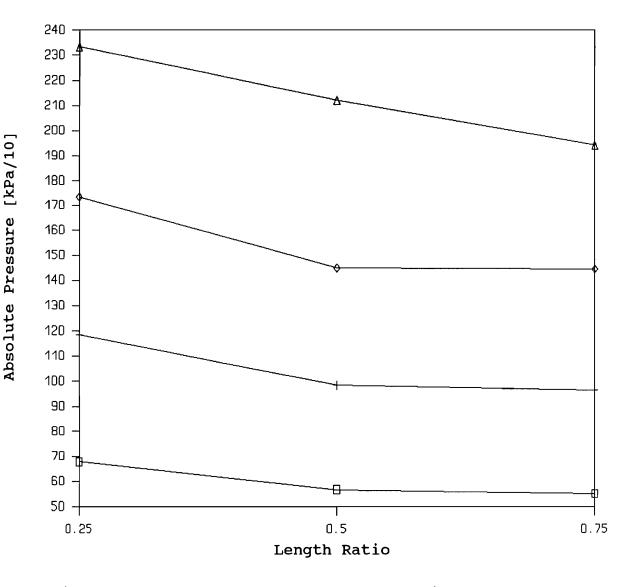


Figure 35: Pressure Data Averages for Ring 11 at Zero Eccentricity Ratio.

CONCLUSIONS

Based on the results of this research, the following conclusions have been made:

- Minor shaft slope or seal ring bore-to-sealing face non-perpendicularity can have significant influence on shaft centering forces. Conventional analysis of these rings to determine dynamic characteristics neglects axial slope. The results indicate that these factors should be considered.
- Experimental values from these tests indicate higher effective friction coefficients than is currently used in industry for design purposes.
- 3) It may be possible to optimize the centering capacity of a seal ring for use in turbomachinery with a sloped shaft due to gravity sag by incorporating bore-tosealing face non-perpendicularity into the design of a ring to cancel the effects of shaft slope.
- 4) The results indicate a lower effective coefficient of friction when the seal ring is at high eccentricity and moving in a direction to center the ring, compared to a direction which further decenters the ring.

RECOMMENDATIONS

- Further testing of the influence of seal ring bore-to-sealing face nonperpendicularity should be conducted to determine the possibility of optimizing a design considering shaft slope due to gravity sag.
- A dynamic seal test rig is necessary to determine the effective friction coefficients under operating conditions including the shaft rotational effects.
- 3) Designers should consider higher friction coefficients than those currently used for analysis. Static test results indicate a predominance of effective friction coefficients in the range of 0.25 to 0.30 as lower limits for rings in the range of 0.50 to 0.75 eccentricity ratio, which is where many rings are predicted to operate.
- 4) Further testing on seal rings is needed to confirm that a circumferential groove added to a ring bore will reduce the effects of shaft slope or ring nonperpendicularity on radial centering forces.

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APPENDIX A

Calculation of $\beta_{\rm F}$

Calculation of β_F for Assumed Linear Lapped Face Pressure Profile

The factor $\beta_{\rm F}$ can be calculated for a linear pressure drop on the lapped face by integrating the profile pressure difference across the face width.

$$F_{pressure} = \int_{R_i}^{R_o} (P_s + \Delta P - P_o) \left[\frac{R_o - R}{R_o - R_i}\right] 2\pi R dR \qquad (A-1)$$

This results in the following expression,

$$F_{pressure} = \frac{\tilde{p}2\pi}{\Delta R} \left[\frac{R_o^3 - R_o R_i^2}{2} - \frac{R_o^3 - R_i^3}{3} \right]$$

= $\tilde{p} \frac{\pi}{4} (D_o^2 - D_i^2) \beta_F$ (A-2)

where

$$\tilde{p} = P_a + \Delta P - P_a \tag{A-3}$$

$$\beta_F = \frac{1 - 3\overline{D}^2 + 2\overline{D}^3}{3(1 - \overline{D})(1 - \overline{D}^2)}$$
(A-4)

and

$$\overline{D} = \frac{D_i}{D_o} \tag{A-5}$$

D_{o}	D _i	\overline{D}	$eta_{ extsf{F}}$
(in)	(in)	(dim)	(dim)
6.5	6.0	0.923	0.492
7.0	6.0	0.857	0.487
5.0	4.0	0.800	0.481

For typical values of \overline{D} the following results were obtained:

These typical results give some assurance that using an average value of $\beta_F \sim 0.5$ is a good engineering approximation for a linear pressure drop on the lapped sealing face. The operating β_F can vary due to ring distortion or end plate distortion and increase or decrease the pressure loading for a given design. The actual β_F during operation is required for accurate seal operating eccentricity evaluation.

APPENDIX B

Friction Coefficient Data for Individual Rings

Table 4: Data for Ring 1 at 689 kPa SEAL POSITIONS: 0.750 0.500 0.250 0.000 0.250 0.500 0.750 FRICTION COEFFICIENT AVERAGES: FORWARD: 1.187 0.623 0.303 0.230 0.207 0.190 0.140 BACKWARD: 0.350 0.460 0.540 0.617 0.773 0.993 1.237 STANDARD DEVIATIONS: FORWARD: 0.031 0.034 0.012 0.000 0.005 0.008 0.000 BACKWARD: 0.016 0.014 0.008 0.009 0.021 0.031 0.025 MAXIMUM COEFF AT EACH ECCENTRICITY: 1.230 0.670 0.550 0.630 0.800 1.020 1.270 MINIMUM COEFF AT EACH ECCENTRICITY: 0.330 0.450 0.290 0.230 0.200 0.180 0.140 The OVERALL MAXIMUM coefficient is: 1.270 The OVERALL MINIMUM coefficient is: 0.140

Table 5: Data for Ring 1 at 1378 kPa

SEAL POSITIONS:	0.750	0.500	0.250	0.000	0.250	0.500	0.750
FRICTION COEFFICIENT AV	ERAGES:						
FORWARD: BACKWARD:	1.163 0.090	0.550 0.117	0.367 0.137	0.323 0.167	0.300 0.203		0.213 0.287
STANDARD DEVIATIONS:							
FORWARD: BACKWARD:	0.012 0.016	0.062 0.025	0.012 0.025	0.024 0.025	0.028 0.029		0.017 0.039
MAXIMUM COEFF AT EACH E	CCENTRI	CITY:					
	1.180	0.620	0.380	0.340	0.320	0.300	0.340
MINIMUM COEFF AT EACH E	CCENTRI	CITY:					
	0.070	0.090	0.110	0.140	0.170	0.210	0.190
The OVERALL MAXIMUM coe	fficien	t is: 1	.180				
The OVERALL MINIMUM coe	fficien	t is: O	.070				

Table 6: Data for Ring 1 at 2068 kPa

Table 7: Data for Ring 1 at 2757 kPa

SEAL POSITIONS:	0.750	0.500	0.250	0.000	0.250	0.500	0.750	
FRICTION COEFFICIENT AV	ERAGES:							
FORWARD: BACKWARD:	0.603 0.293	0.447 0.360	0.410 0.383	0.383 0.430	0.350 0.487	0.323 0.557	0.250 0.667	
STANDARD DEVIATIONS:								
FORWARD: BACKWARD:	0.172 0.076	0.151 0.065	0.170 0.068	0.175 0.016	0.163 0.026	0.147 0.048	0.099 0.063	
MAXIMUM COEFF AT EACH E	CCENTRI	CITY:						
	0.730	0.660	0.650	0.630	0.580	0.600	0.730	
MINIMUM COEFF AT EACH E	CCENTRI	CITY:						
	0.190	0.270	0.270	0.240	0.220	0.200	0.170	
The OVERALL MAXIMUM coe	fficien	t is: O	.730					
The OVERALL MINIMUM coe	fficien	t is: 0	.170					

Table 8: Data for Ring 2 at 689 kPa SEAL POSITIONS: 0.750 0.500 0.250 0.000 0.250 0.500 0.750 FRICTION COEFFICIENT AVERAGES: FORWARD: 0.733 0.473 0.400 0.377 0.370 0.353 0.307 BACKWARD: 0.400 0.423 0.447 0.503 0.633 0.820 1.027 STANDARD DEVIATIONS: FORWARD: 0.021 0.063 0.029 0.026 0.024 0.021 0.031 BACKWARD: 0.008 0.009 0.005 0.029 0.021 0.008 0.005 MAXIMUM COEFF AT EACH ECCENTRICITY: 0.760 0.560 0.450 0.540 0.660 0.830 1.030 MINIMUM COEFF AT EACH ECCENTRICITY: 0.390 0.410 0.360 0.340 0.340 0.330 0.280 The OVERALL MAXIMUM coefficient is: 1.030 The OVERALL MINIMUM coefficient is: 0.280 Table 9: Data for Ring 2 at 1378 kPa 0.750 0.500 0.250 0.000 0.250 0.500 SEAL POSITIONS: 0.750 FRICTION COEFFICIENT AVERAGES: FORWARD: 0.770 0.543 0.477 0.440 0.423 0.397 0.330 BACKWARD: 0.390 0.417 0.467 0.540 0.640 0.787 0.983 STANDARD DEVIATIONS: FORWARD: 0.028 0.057 0.025 0.029 0.041 0.034 0.022 BACKWARD: 0.008 0.012 0.012 0.014 0.022 0.021 0.021

MAXIMUM COEFF AT EACH ECCENTRICITY:

0.810 0.610 0.510 0.560 0.660 0.810 1.010

MINIMUM COEFF AT EACH ECCENTRICITY:

0.380 0.400 0.450 0.400 0.370 0.350 0.300

The OVERALL MAXIMUM coefficient is: 1.010

The OVERALL MINIMUM coefficient is: 0.300

SEAL POSITIONS: 0.750 0.500 0.250 0.000 0.250 0.500 0.750 FRICTION COEFFICIENT AVERAGES: FORWARD:0.8730.6630.5830.5430.4900.447BACKWARD:0.3870.4670.5270.5900.6930.827 0.323 1.013 STANDARD DEVIATIONS: FORWARD: 0.045 0.060 0.033 0.025 0.024 0.029 0.058 BACKWARD: 0.009 0.005 0.012 0.016 0.017 0.017 0.012 MAXIMUM COEFF AT EACH ECCENTRICITY: 0.930 0.720 0.620 0.610 0.710 0.850 1.030 MINIMUM COEFF AT EACH ECCENTRICITY: 0.380 0.460 0.510 0.510 0.460 0.410 0.260 The OVERALL MAXIMUM coefficient is: 1.030 The OVERALL MINIMUM coefficient is: 0.260

Table 11: Data for Ring 2 at 2757 kPa

SEAL POSITIONS:	0.750	0.500	0.250	0.000	0.250	0.500	0.750
FRICTION COEFFICIENT AV	ERAGES:						
FORWARD: BACKWARD:	0.753 0.350	0.597 0.470		0.480 0.597	0.417 0.697		0.247 0.997
STANDARD DEVIATIONS:							
FORWARD: BACKWARD:		0.012 0.014		0.022 0.005	0.019 0.009		0.025 0.005
MAXIMUM COEFF AT EACH E	CCENTRI	CITY:					
	0.770	0.610	0.530	0.600	0.710	0.840	1.000
MINIMUM COEFF AT EACH E	CCENTRI	CITY:					
	0.310	0.460	0.480	0.450	0.390	0.330	0.220
The OVERALL MAXIMUM coe	fficien	t is: 1	.000				
The OVERALL MINIMUM coe	fficien	t is: O	.220				

Table 10: Data for Ring 2 at 2068 kPa

Table 12: Data for Ring 3 at 689 kPa 0.750 0.500 0.250 0.000 0.250 0.500 0.750 SEAL POSITIONS: FRICTION COEFFICIENT AVERAGES: FORWARD: 0.720 0.380 0.263 0.203 0.190 0.157 0.120 BACKWARD: 0.097 0.243 0.393 0.663 0.793 0.873 1.010 STANDARD DEVIATIONS: FORWARD: 0.024 0.022 0.009 0.019 0.008 0.026 0.008 BACKWARD: 0.012 0.005 0.012 0.005 0.005 0.009 0.014 MAXIMUM COEFF AT EACH ECCENTRICITY: 0.750 0.410 0.410 0.670 0.800 0.880 1.030 MINIMUM COEFF AT EACH ECCENTRICITY: 0.080 0.240 0.250 0.190 0.180 0.120 0.110 The OVERALL MAXIMUM coefficient is: 1.030 The OVERALL MINIMUM coefficient is: 0.080

Table 13: Data for Ring 3 at 1378 kPa

SEAL POSITIONS:	0.750	0.500	0.250	0.000	0.250	0.500	0.750		
FRICTION COEFFICIENT AV	ERAGES:								
FORWARD: BACKWARD:	0.900 0.000	0.463 0.140	0.293 0.387	0.190 0.607	0.157 0.663	0.150 0.717	0.117 0.827		
STANDARD DEVIATIONS:									
FORWARD: BACKWARD:	0.037 0.000	0.017 0.008	0.009 0.045	0.008 0.009	0.005 0.012	0.014 0.019	0.034 0.005		
MAXIMUM COEFF AT EACH E	CCENTRI	CITY:							
	0.940	0.480	0.450	0.620	0.680	0.730	0.830		
MINIMUM COEFF AT EACH E	CCENTRI	CITY:							
	0.000	0.130	0.280	0.180	0.150	0.130	0.070		
The OVERALL MAXIMUM coe	fficien	t is: 0	.940						
The OVERALL MINIMUM coe	fficien	t is: 0	.000						

	14010 111 04	04 101 1	ting o t	10 2000			
SEAL POSITIO	ONS: 0.750	0.500	0.250	0.000	0.250	0.500	0.750
FRICTION COEFFICI	ENT AVERAGES:						
	WARD: 0.890 KWARD: 0.000						
STANDARD DEVIATIO	DNS:						
	WARD: 0.029 KWARD: 0.000			0.057 0.009			
MAXIMUM COEFF AT	EACH ECCENTRI	CITY:					
	0.930	0.480	0.340	0.490	0.620	0.750	0.890
MINIMUM COEFF AT	EACH ECCENTRI	CITY:					
	0.000	0.170	0.230	0.130	0.120	0.130	0.110
The OVERALL MAXIM	UM coefficier	it is: 0	.930				
The OVERALL MINIM	IUM coefficier	nt is: O	.000				

Table 14: Data for Ring 3 at 2068 kPa

Table 15: Data for Ring 3 at 2757 kPa

SEAL POSITIONS:	0.750	0.500	0.250	0.000	0.250	0.500	0.750	
FRICTION COEFFICIENT AV	TERAGES:							
FORWARD: BACKWARD:	0.970 0.000	0.573 0.210	0.420 0.370	0.323 0.500	0.250 0.617	0.197 0.757	0.167 0.907	
STANDARD DEVIATIONS:								
FORWARD: BACKWARD:	0.029 0.000	0.005 0.014		0.017 0.014	0.014 0.005	0.012 0.009	0.012 0.005	
MAXIMUM COEFF AT EACH E	CCENTRI	CITY:						
	1.010	0.580	0.430	0.510	0.620	0.770	0.910	
MINIMUM COEFF AT EACH E	CCENTRI	CITY:						
	0.000	0.190	0.370	0.300	0.230	0.180	0.150	
The OVERALL MAXIMUM coe	fficien	t is: 1	.010					
The OVERALL MINIMUM coe	fficien	t is: 0	.000					

Table 16: Data for Ring 4 at 689 kPa SEAL POSITIONS: 0.750 0.500 0.250 0.000 0.250 0.500 0.750 FRICTION COEFFICIENT AVERAGES: FORWARD: 0.717 0.473 0.337 0.233 0.257 0.287 0.290 BACKWARD: 0.317 0.313 0.357 0.470 0.600 0.757 0.803 STANDARD DEVIATIONS: FORWARD: 0.012 0.009 0.019 0.017 0.005 0.012 0.014 BACKWARD: 0.005 0.005 0.021 0.014 0.016 0.005 0.005 MAXIMUM COEFF AT EACH ECCENTRICITY: 0.730 0.480 0.380 0.490 0.620 0.760 0.810 MINIMUM COEFF AT EACH ECCENTRICITY: 0.310 0.310 0.310 0.210 0.250 0.270 0.270 The OVERALL MAXIMUM coefficient is: 0.810 The OVERALL MINIMUM coefficient is: 0.210 Table 17: Data for Ring 4 at 1378 kPa SEAL POSITIONS: 0.750 0.500 0.250 0.000 0.250 0.500 0.750

SEAL POSITIONS	: 0.750	0.500	0.250	0.000	0.250	0.500	0.750
FRICTION COEFFICIEN	T AVERAGES:						
FORWA BACKW	RD: 0.643 ARD: 0.340	0.443 0.380		0.287 0.547	0.243 0.637	0.243 0.800	0.250 1.037
STANDARD DEVIATIONS	:						
FORWA BACKW	RD: 0.009 ARD: 0.000	0.009 0.014	0.008 0.009		0.005 0.009		
MAXIMUM COEFF AT EA	CH ECCENTRI	CITY:					
	0.650	0.450	0.480	0.570	0.650	0.820	1.060
MINIMUM COEFF AT EA	CH ECCENTRI	CITY:					
	0.340	0.370	0.320	0.270	0.240	0.230	0.230
The OVERALL MAXIMUM	coefficien	t is: 1	.060				
The OVERALL MINIMUM	coefficien	t is: O	.230				

Table 18: Data for Ring 4 at 2068 kPa SEAL POSITIONS: 0.750 0.500 0.250 0.000 0.250 0.500 0.750 FRICTION COEFFICIENT AVERAGES: FORWARD: 0.637 0.443 0.347 0.317 0.270 0.273 0.260 BACKWARD: 0.353 0.393 0.467 0.533 0.983 0.627 0.767 STANDARD DEVIATIONS: FORWARD: 0.009 0.005 0.005 0.005 0.008 0.009 0.000 BACKWARD: 0.012 0.024 0.021 0.025 0.029 0.042 0.046 MAXIMUM COEFF AT EACH ECCENTRICITY: 0.650 0.450 0.490 0.560 0.660 0.810 1.030 MINIMUM COEFF AT EACH ECCENTRICITY: 0.340 0.360 0.340 0.310 0.260 0.260 0.260 The OVERALL MAXIMUM coefficient is: 1.030 The OVERALL MINIMUM coefficient is: 0.260 Table 19: Data for Ring 4 at 2757 kPa 0.750 0.500 0.250 0.000 0.250 0.500 0.750 SEAL POSITIONS: FRICTION COEFFICIENT AVERAGES: FORWARD: 0.673 0.470 0.377 0.307 0.290 0.243 0.337 BACKWARD: 0.400 0.467 0.530 0.610 0.710 0.853 1.057 STANDARD DEVIATIONS: 0.008 0.012 0.005 0.005 0.008 0.021 FORWARD: 0.017 BACKWARD: 0.022 0.021 0.022 0.016 0.016 0.012 0.017 MAXIMUM COEFF AT EACH ECCENTRICITY: 0.690 0.490 0.560 0.630 0.730 0.870 1.080 MINIMUM COEFF AT EACH ECCENTRICITY: 0.380 0.440 0.360 0.330 0.300 0.280 0.220 The OVERALL MAXIMUM coefficient is: 1.080 The OVERALL MINIMUM coefficient is: 0.220

 Table 20: Data for Ring 5 at 689 kPa

 SEAL POSITIONS: 0.750 0.500 0.250 0.000 0.250 0.500 0.750

 FRICTION COEFFICIENT AVERAGES:

 FORWARD: 0.893 0.547 0.353 0.267 0.297 0.323 0.310

 BACKWARD: 0.387 0.400 0.400 0.397 0.393 0.430 0.970

 STANDARD DEVIATIONS:

 FORWARD: 0.005 0.017 0.086 0.012 0.012 0.009 0.029

 BACKWARD: 0.005 0.017 0.086 0.012 0.012 0.009 0.029

 BACKWARD: 0.021 0.022 0.014 0.021 0.009 0.008 0.008

 MAXIMUM COEFF AT EACH ECCENTRICITY:

 0.900 0.570 0.460 0.420 0.400 0.440 0.980

 MINIMUM COEFF AT EACH ECCENTRICITY:

 0.360 0.370 0.250 0.250 0.280 0.310 0.270

 The OVERALL MAXIMUM coefficient is: 0.980

 The OVERALL MINIMUM coefficient is: 0.250

Table 21: Data for Ring 5 at 1378 kPa

SEAL POSITIONS:	0.750	0.500	0.250	0.000	0.250	0.500	0.750	
FRICTION COEFFICIENT AV	ERAGES:							
FORWARD: BACKWARD:	0.790 0.457	0.550 0.450	0.403 0.453	0.300 0.470	0.303 0.560	0.330 0.760	0.347 1.057	
STANDARD DEVIATIONS:								
FORWARD: BACKWARD:	0.042 0.012	0.036 0.022	0.066 0.024	0.036 0.022	0.029 0.008	0.022 0.022	0.017 0.049	
MAXIMUM COEFF AT EACH E	CCENTRI	CITY:						
	0.820	0.580	0.470	0.490	0.570	0.780	1.120	
MINIMUM COEFF AT EACH E	CCENTRI	CITY:						
	0.440	0.420	0.310	0.250	0.270	0.310	0.330	
The OVERALL MAXIMUM coe	fficien	t is: 1	.120					
The OVERALL MINIMUM coe	fficien	t is: 0	.250					

Table 22: Data for Ring 5 at 2068 kPa SEAL POSITIONS: 0.750 0.500 0.250 0.000 0.250 0.500 0.750 FRICTION COEFFICIENT AVERAGES: FORWARD: 0.737 0.507 0.383 0.327 0.323 0.337 0.330 BACKWARD: 0.413 0.487 0.487 0.517 0.583 0.723 1.020 STANDARD DEVIATIONS: FORWARD:0.0370.0340.0520.0240.0310.025BACKWARD:0.0450.0190.0120.0190.0450.045 0.036 0.050 MAXIMUM COEFF AT EACH ECCENTRICITY: 0.780 0.540 0.500 0.530 0.620 0.760 1.060 MINIMUM COEFF AT EACH ECCENTRICITY: 0.360 0.460 0.310 0.310 0.280 0.310 0.280 The OVERALL MAXIMUM coefficient is: 1.060 The OVERALL MINIMUM coefficient is: 0.280

Table 23: Data for Ring 5 at 2757 kPa

SEAL POSITIONS:	0.750	0.500	0.250	0.000	0.250	0.500	0.750	
FRICTION COEFFICIENT AV	ERAGES:							
FORWARD: BACKWARD:	0.707 0.507	0.530 0.493	0.410 0.523	0.343 0.520	0.317 0.560	0.317 0.697	0.280 0.953	
STANDARD DEVIATIONS:								
FORWARD: BACKWARD:	0.019 0.026	0.016 0.026	0.000 0.005		0.005 0.008	0.024 0.012	0.037 0.026	
MAXIMUM COEFF AT EACH E	CCENTRI	CITY:						
	0.720	0.550	0.530	0.560	0.570	0.710	0.990	
MINIMUM COEFF AT EACH E	CCENTRI	CITY:						
	0.470	0.470	0.410	0.320	0.310	0.300	0.230	
The OVERALL MAXIMUM coe	fficien	t is: O	.990					
The OVERALL MINIMUM coe	fficien	t is: 0	.230					

 Table 24: Data for Ring 6 at 689 kPa

 SEAL POSITIONS: 0.750 0.500 0.250 0.000 0.250 0.500 0.750

 FRICTION COEFFICIENT AVERAGES:

 FORWARD: 0.747 0.477 0.377 0.353 0.357 0.343 0.333

 BACKWARD: 0.747 0.477 0.377 0.353 0.357 0.343 0.333

 BACKWARD: 0.747 0.477 0.377 0.353 0.357 0.343 0.333

 STANDARD DEVIATIONS:

 FORWARD: 0.068 0.074 0.045 0.017 0.045 0.063 0.037

 BACKWARD: 0.014 0.017 0.009 0.008 0.012 0.034 0.177

 MAXIMUM COEFF AT EACH ECCENTRICITY:

 0.800 0.570 0.430 0.370 0.420 0.430 0.800

 MINIMUM COEFF AT EACH ECCENTRICITY:

 0.300 0.290 0.300 0.290 0.280 0.280 0.290

 The OVERALL MAXIMUM coefficient is: 0.800

 The OVERALL MINIMUM coefficient is: 0.280

Table 25: Data for Ring 6 at 1378 kPa

SEAL POSITIONS:	0.750	0.500	0.250	0.000	0.250	0.500	0.750
FRICTION COEFFICIENT AV	ERAGES:						
FORWARD: BACKWARD:	0.783 0.317	0.550 0.293	0.410 0.277	0.367 0.300	0.323 0.427	0.307 0.573	0.320 0.783
STANDARD DEVIATIONS:							
FORWARD: BACKWARD:	0.005 0.005	0.008 0.005	0.008 0.009	0.012 0.024	0.009 0.034		0.008 0.009
MAXIMUM COEFF AT EACH E	CCENTRI	CITY:					
	0.790	0.560	0.420	0.380	0.460	0.580	0.790
MINIMUM COEFF AT EACH E	CCENTRI	CITY:					
	0.310	0.290	0.270	0.270	0.310	0.290	0.310
The OVERALL MAXIMUM coe	fficien	t is: O	.790				
The OVERALL MINIMUM coe	fficien	t is: 0	.270				

 Table 26: Data for Ring 6 at 2068 kPa

 SEAL POSITIONS: 0.750 0.500 0.250 0.000 0.250 0.500 0.750

 FRICTION COEFFICIENT AVERAGES:

 FORWARD: 0.760 0.563 0.430 0.383 0.347 0.327 0.307

 BACKWARD: 0.317 0.337 0.337 0.403 0.490 0.627 0.820

 STANDARD DEVIATIONS:

 FORWARD: 0.008 0.005 0.008 0.005 0.009 0.009 0.012

 BACKWARD: 0.008 0.005 0.008 0.005 0.009 0.009 0.012

 BACKWARD: 0.005 0.012 0.017 0.014 0.005 0.008

 MAXIMUM COEFF AT EACH ECCENTRICITY:

 0.310 0.320 0.320 0.380 0.340 0.320 0.290

 The OVERALL MAXIMUM coefficient is: 0.830

 The OVERALL MINIMUM coefficient is: 0.290

Table 27: Data for Ring 6 at 2757 kPa

SEAL POSITIONS:	0.750	0.500	0.250	0.000	0.250	0.500	0.750
FRICTION COEFFICIENT AV	VERAGES:						
FORWARD : BACKWARD :	0.740 0.223	0.577 0.383		0.417 0.460		0.360 0.670	0.280 0.853
STANDARD DEVIATIONS:							
FORWARD: BACKWARD:	0.022 0.017	0.017 0.017		0.021 0.024			0.008 0.066
MAXIMUM COEFF AT EACH E	CCENTRI	CITY:					
	0.760	0.600	0.480	0.490	0.570	0.720	0.940
MINIMUM COEFF AT EACH E	CCENTRI	CITY:					
	0.200	0.360	0.400	0.390	0.370	0.350	0.270
The OVERALL MAXIMUM coe	efficien	t is: 0	.940				
The OVERALL MINIMUM coe	fficien	t is: 0	.200				

 Table 28: Data for Ring 7 at 689 kPa

 SEAL POSITIONS: 0.750 0.500 0.250 0.000 0.250 0.500 0.750

 FRICTION COEFFICIENT AVERAGES:

 FORWARD: 1.073 0.693 0.607 0.567 0.560 0.553 0.523

 BACKWARD: 0.300 0.273 0.250 0.190 0.160 0.217 0.710

 STANDARD DEVIATIONS:

 FORWARD: 0.029 0.073 0.021 0.026 0.029 0.031 0.024

 BACKWARD: 0.029 0.073 0.021 0.026 0.029 0.031 0.024

 BACKWARD: 0.008 0.017 0.016 0.000 0.000 0.082 0.008

 MAXIMUM COEFF AT EACH ECCENTRICITY:

 1.110 0.750 0.630 0.590 0.590 0.580 0.720

 MINIMUM COEFF AT EACH ECCENTRICITY:

 0.290 0.250 0.230 0.190 0.160 0.140 0.490

 The OVERALL MAXIMUM coefficient is: 1.110

 The OVERALL MINIMUM coefficient is: 0.140

Table 29: Data for Ring 7 at 1378 kPa

SEAL POSITIONS:	0.750	0.500	0.250	0.000	0.250	0.500	0.750
FRICTION COEFFICIENT AV	ERAGES:					_	
FORWARD: BACKWARD:	0.800 0.300	0.587 0.267	0.503 0.217	0.450 0.173	0.403 0.200		0.340 0.857
STANDARD DEVIATIONS:							
FORWARD: BACKWARD:	0.028 0.008	0.009 0.012	0.019 0.009	0.014 0.017	0.012 0.042	0.014 0.005	0.016 0.005
MAXIMUM COEFF AT EACH E	CCENTRI	CITY:			_		
	0.840	0.600	0.530	0.470	0.420	0.530	0.860
MINIMUM COEFF AT EACH E	CCENTRI	CITY:					
	0.290	0.250	0.210	0.150	0.170	0.370	0.320
The OVERALL MAXIMUM coe	fficien	t is: O	.860			-	
The OVERALL MINIMUM coe	fficien	t is: O	.150				

 Table 30: Data for Ring 7 at 2068 kPa

 SEAL POSITIONS: 0.750 0.500 0.250 0.000 0.250 0.500 0.750

 FRICTION COEFFICIENT AVERAGES:

 FORWARD: 0.760 0.593 0.507 0.457 0.410 0.380 0.360

 BACKWARD: 0.760 0.593 0.507 0.457 0.410 0.380 0.360

 BACKWARD: 0.760 0.593 0.507 0.457 0.410 0.380 0.360

 STANDARD DEVIATIONS:

 FORWARD: 0.014 0.005 0.005 0.005 0.008 0.008 0.014

 BACKWARD: 0.014 0.012 0.025 0.024 0.029 0.062 0.025

 MAXIMUM COEFF AT EACH ECCENTRICITY:

 0.770 0.600 0.510 0.460 0.420 0.720 1.200

 MINIMUM COEFF AT EACH ECCENTRICITY:

 0.360 0.350 0.290 0.230 0.230 0.370 0.340

 The OVERALL MAXIMUM coefficient is: 1.200

Table 31: Data for Ring 7 at 2757 kPa

SEAL POSITIONS:	0.750	0.500	0.250	0.000	0.250	0.500	0.750		
FRICTION COEFFICIENT AV	VERAGES:								
FORWARD : BACKWARD :	0.687 0.400	0.527 0.363		0.423 0.420	0.393 0.493		0.317 1.393		
STANDARD DEVIATIONS:									
FORWARD: BACKWARD:	0.141 0.014	0.097 0.034	0.083 0.005	0.074 0.057	0.066 0.152		0.054 0.143		
MAXIMUM COEFF AT EACH E	ECCENTRI	CITY:							
	0.810	0.610	0.520	0.490	0.620	0.860	1.560		
MINIMUM COEFF AT EACH E	CCENTRI	CITY:							
	0.390	0.330	0.340	0.320	0.280	0.280	0.260		
The OVERALL MAXIMUM coe	efficien	t is: 1	.560						
The OVERALL MINIMUM coe	efficien	t is: O	.260						

 Table 32: Data for Ring 8 at 689 kPa

 SEAL POSITIONS: 0.750 0.500 0.250 0.000 0.250 0.500 0.750

 FRICTION COEFFICIENT AVERAGES:

 FORWARD: 0.817 0.553 0.477 0.430 0.427 0.430 0.427

 BACKWARD: 0.430 0.420 0.400 0.377 0.353 0.340 0.697

 STANDARD DEVIATIONS:

 FORWARD: 0.034 0.073 0.033 0.008 0.005 0.000 0.005

 BACKWARD: 0.034 0.073 0.033 0.008 0.005 0.000 0.005

 BACKWARD: 0.000 0.008 0.008 0.009 0.009 0.000 0.017

 MAXIMUM COEFF AT EACH ECCENTRICITY:

 0.850 0.610 0.500 0.440 0.430 0.430 0.720

 MINIMUM COEFF AT EACH ECCENTRICITY:

 0.430 0.410 0.390 0.370 0.340 0.340 0.420

 The OVERALL MAXIMUM coefficient is: 0.850

 The OVERALL MINIMUM coefficient is: 0.340

Table 33: Data for Ring 8 at 1378 kPa

SEAL POSITIONS:	0.750	0.500	0.250	0.000	0.250	0.500	0.750
FRICTION COEFFICIENT AV	ERAGES:						
FORWARD: BACKWARD:	0.773 0.357	0.600 0.357		0.460 0.320	0.430 0.313		0.363 0.623
STANDARD DEVIATIONS:							
FORWARD: BACKWARD:	0.012 0.012	0.022 0.012		0.008 0.008	0.016 0.009		0.025 0.059
MAXIMUM COEFF AT EACH E	CCENTRI	CITY:					
	0.790	0.630	0.530	0.470	0.450	0.440	0.670
MINIMUM COEFF AT EACH E	CCENTRI	CITY:					
	0.340	0.340	0.330	0.310	0.300	0.290	0.330
The OVERALL MAXIMUM coe	fficien	t is: O	.790				
The OVERALL MINIMUM coe	fficien	t is: 0	.290				

Table 34: Data for Ring 8 at 2068 kPa SEAL POSITIONS: 0.750 0.500 0.250 0.000 0.250 0.500 0.750 FRICTION COEFFICIENT AVERAGES: FORWARD: 0.633 0.597 0.573 0.587 0.557 0.563 0.543 BACKWARD: 0.443 0.453 0.417 0.407 0.413 0.387 0.397 STANDARD DEVIATIONS: FORWARD: 0.021 0.026 0.025 0.034 0.047 0.092 BACKWARD: 0.024 0.037 0.047 0.040 0.025 0.017 0.105 0.021 MAXIMUM COEFF AT EACH ECCENTRICITY: 0.660 0.620 0.600 0.620 0.590 0.660 0.650 MINIMUM COEFF AT EACH ECCENTRICITY: 0.410 0.410 0.350 0.350 0.380 0.370 0.370 The OVERALL MAXIMUM coefficient is: 0.660 The OVERALL MINIMUM coefficient is: 0.350

NO DATA FOR RING 8 AT 2757 kPa

 SEAL POSITIONS:
 0.750
 0.500
 0.250
 0.000
 0.250
 0.500
 0.750

 FRICTION COEFFICIENT AVERAGES:
 FORWARD:
 0.847
 0.410
 0.330
 0.347
 0.377
 0.390
 0.397

 BACKWARD:
 0.403
 0.390
 0.377
 0.370
 0.490
 0.650
 0.883

 STANDARD DEVIATIONS:
 FORWARD:
 0.017
 0.016
 0.022
 0.019
 0.021
 0.022
 0.034

 MAXIMUM COEFF AT EACH ECCENTRICITY:
 0.870
 0.430
 0.390
 0.380
 0.500
 0.660
 0.910

 MINIMUM COEFF AT EACH ECCENTRICITY:
 0.390
 0.380
 0.300
 0.320
 0.360
 0.350

 The OVERALL MAXIMUM coefficient is:
 0.910
 The OVERALL MINIMUM coefficient is:
 0.300

Table 36: Data for Ring 9 at 1378 kPa

SEAL POSITIONS:	0.750	0.500	0.250	0.000	0.250	0.500	0.750
FRICTION COEFFICIENT AV	ERAGES:						
FORWARD: BACKWARD:	0.780 0.410	0.480 0.390	0.353 0.403		0.353 0.550	0.367 0.667	0.360 0.850
STANDARD DEVIATIONS:			_				
FORWARD: BACKWARD:		0.128 0.024	0.049 0.031	0.022 0.036	0.005 0.014		0.000 0.028
MAXIMUM COEFF AT EACH E	CCENTRI	CITY:					
	0.820	0.580	0.430	0.490	0.560	0.700	0.890
MINIMUM COEFF AT EACH E	CCENTRI	CITY:					
	0.390	0.300	0.290	0.310	0.350	0.360	0.360
The OVERALL MAXIMUM coe	fficien	t is: O	.890				
The OVERALL MINIMUM coe	fficien	t is: 0	.290				

Table 35: Data for Ring 9 at 689 kPa

Table 37: Data for Ring 9 at 2068 kPa SEAL POSITIONS: 0.750 0.500 0.250 0.000 0.250 0.500 0.750 FRICTION COEFFICIENT AVERAGES: FORWARD: 0.767 0.533 0.393 0.350 0.350 0.360 0.337 BACKWARD: 0.407 0.443 0.457 0.503 0.590 0.690 0.877 STANDARD DEVIATIONS: FORWARD: 0.009 0.012 0.012 0.014 0.014 0.028 0.019 BACKWARD: 0.012 0.009 0.009 0.017 0.016 0.016 0.026 MAXIMUM COEFF AT EACH ECCENTRICITY: 0.780 0.550 0.470 0.520 0.610 0.710 0.900 MINIMUM COEFF AT EACH ECCENTRICITY: 0.390 0.430 0.380 0.340 0.330 0.320 0.310 The OVERALL MAXIMUM coefficient is: 0.900 The OVERALL MINIMUM coefficient is: 0.310 Table 38: Data for Ring 9 at 2757 kPa SEAL POSITIONS: 0.750 0.500 0.250 0.000 0.250 0.500 0.750 FRICTION COEFFICIENT AVERAGES: FORWARD: 0.757 0.550 0.420 0.413 0.387 0.260 0.397 BACKWARD: 0.390 0.470 0.503 0.530 0.617 0.723 0.917 STANDARD DEVIATIONS: 0.014 0.012 FORWARD: 0.009 0.008 0.009 0.005 0.024 BACKWARD: 0.029 0.014 0.009 0.016 0.029 0.024 0.042 MAXIMUM COEFF AT EACH ECCENTRICITY: 0.770 0.560 0.510 0.550 0.650 0.740 0.960 MINIMUM COEFF AT EACH ECCENTRICITY: 0.360 0.460 0.400 0.400 0.390 0.380 0.230 The OVERALL MAXIMUM coefficient is: 0.960 The OVERALL MINIMUM coefficient is: 0.230

Table 39: Data for Ring 10 at 689 kPa SEAL POSITIONS: 0.750 0.500 0.250 0.000 0.250 0.500 0.750 FRICTION COEFFICIENT AVERAGES: FORWARD: 0.887 0.617 0.473 0.390 0.380 0.393 0.387 BACKWARD: 0.327 0.303 0.267 0.257 0.333 0.423 0.720 STANDARD DEVIATIONS: FORWARD: 0.037 0.033 0.066 0.000 0.008 0.012 0.026 BACKWARD: 0.012 0.039 0.041 0.048 0.068 0.172 0.008 MAXIMUM COEFF AT EACH ECCENTRICITY: 0.930 0.640 0.520 0.390 0.400 0.550 0.730 MINIMUM COEFF AT EACH ECCENTRICITY: 0.310 0.250 0.220 0.190 0.240 0.180 0.350 The OVERALL MAXIMUM coefficient is: 0.930 The OVERALL MINIMUM coefficient is: 0.180 Table 40: Data for Ring 10 at 1378 kPa SEAL POSITIONS: 0.750 0.500 0.250 0.000 0.250 0.500 0.750 FRICTION COEFFICIENT AVERAGES: FORWARD: 0.853 0.650 0.547 0.460 0.407 0.417 0.417 BACKWARD: 0.297 0.270 0.283 0.340 0.440 0.580 0.780 STANDARD DEVIATIONS: FORWARD: 0.040 0.036 0.031 0.054 0.017 0.012 0.005 BACKWARD: 0.005 0.000 0.017 0.008 0.008 0.016 0.014 MAXIMUM COEFF AT EACH ECCENTRICITY: 0.910 0.700 0.590 0.520 0.450 0.600 0.800 MINIMUM COEFF AT EACH ECCENTRICITY: 0.290 0.270 0.260 0.330 0.390 0.400 0.410 The OVERALL MAXIMUM coefficient is: 0.910 The OVERALL MINIMUM coefficient is: 0.260

 SEAL POSITIONS:
 0.750
 0.500
 0.250
 0.000
 0.250
 0.500
 0.750

 FRICTION COEFFICIENT AVERAGES:
 FORWARD:
 0.957
 0.737
 0.643
 0.523
 0.517
 0.533
 0.410

 BACKWARD:
 0.253
 0.317
 0.347
 0.380
 0.477
 0.607
 0.800

 STANDARD DEVIATIONS:
 FORWARD:
 0.009
 0.021
 0.029
 0.009
 0.026
 0.021
 0.080

 MAXIMUM COEFF AT EACH ECCENTRICITY:
 0.970
 0.760
 0.680
 0.530
 0.540
 0.620
 0.830

 MINIMUM COEFF AT EACH ECCENTRICITY:
 0.240
 0.300
 0.340
 0.380
 0.470
 0.510
 0.290

 The OVERALL MAXIMUM coefficient is:
 0.970
 1.340
 0.380
 0.470
 0.510
 0.290

Table 42: Data for Ring 10 at 2757 kPa

SEAL POSITIONS:	0.750	0.500	0.250	0.000	0.250	0.500	0.750
FRICTION COEFFICIENT AV	ERAGES:						
FORWARD: BACKWARD:	0.883 0.203	0.690 0.337	0.560 0.383	0.460 0.403	0.490 0.487	0.510 0.620	0.350 0.827
STANDARD DEVIATIONS:							
FORWARD: BACKWARD:	0.031 0.021	0.014 0.009	0.022 0.009	0.028 0.012	0.024 0.005	0.022 0.008	0.000 0.017
MAXIMUM COEFF AT EACH E	CCENTRI	CITY:					
	0.910	0.710	0.580	0.480	0.520	0.630	0.850
MINIMUM COEFF AT EACH E	CCENTRI	CITY:					
	0.180	0.330	0.370	0.390	0.460	0.490	0.350
The OVERALL MAXIMUM coe	fficien	t is: O	.910				
The OVERALL MINIMUM coe	fficien	t is: 0	.180				

Table 41: Data for Ring 10 at 2068 kPa

 Table 43: Data for Ring 11 at 689 kPa

 SEAL POSITIONS: 0.750 0.500 0.250 0.000 0.250 0.500 0.750

 FRICTION COEFFICIENT AVERAGES:

 FORWARD: 0.850 0.520 0.323 0.340 0.353 0.353 0.337

 BACKWARD: 0.423 0.433 0.423 0.390 0.380 0.807 1.187

 STANDARD DEVIATIONS:

 FORWARD: 0.022 0.042 0.019 0.022 0.021 0.021 0.025

 BACKWARD: 0.009 0.005 0.005 0.000 0.000 0.009 0.025

 MAXIMUM COEFF AT EACH ECCENTRICITY:

 0.410 0.430 0.310 0.320 0.330 0.330 0.310

 The OVERALL MAXIMUM coefficient is: 1.220

 The OVERALL MINIMUM coefficient is: 0.310

Table 44: Data for Ring 11 at 1378 kPa

SEAL POSITIONS:	0.750	0.500	0.250	0.000	0.250	0.500	0.750
FRICTION COEFFICIENT AV	ERAGES:						
FORWARD: BACKWARD:	0.757 0.437	0.460 0.413		0.333 0.370	0.340 0.377		
STANDARD DEVIATIONS:							
FORWARD: BACKWARD:	0.033 0.012	0.106 0.005		0.005 0.016			
MAXIMUM COEFF AT EACH E	CCENTRI	CITY:					
	0.800	0.540	0.430	0.390	0.390	0.810	1.170
MINIMUM COEFF AT EACH E	CCENTRI	CITY:					
	0.420	0.310	0.300	0.330	0.340	0.340	0.340
The OVERALL MAXIMUM coe	fficien	t is: 1	.170				
The OVERALL MINIMUM coe	fficien	t is: O	.300				

Table 45: Data for Ring 11 at 2068 kPa SEAL POSITIONS: 0.750 0.500 0.250 0.000 0.250 0.500 0.750 FRICTION COEFFICIENT AVERAGES: FORWARD: 0.737 0.520 0.407 0.327 0.323 0.340 0.330 BACKWARD: 0.453 0.427 0.413 0.393 0.477 0.797 1.130 STANDARD DEVIATIONS: FORWARD: 0.012 0.014 0.005 0.024 0.017 0.014 0.014 BACKWARD: 0.005 0.017 0.012 0.012 0.061 0.025 0.028 MAXIMUM COEFF AT EACH ECCENTRICITY: 0.750 0.530 0.430 0.410 0.550 0.830 1.150 MINIMUM COEFF AT EACH ECCENTRICITY: 0.450 0.410 0.400 0.310 0.300 0.320 0.320 The OVERALL MAXIMUM coefficient is: 1.150 The OVERALL MINIMUM coefficient is: 0.300 Table 46: Data for Ring 11 at 2757 kPa 0.750 0.500 0.250 0.000 0.250 0.500 0.750 SEAL POSITIONS: FRICTION COEFFICIENT AVERAGES: FORWARD: 0.763 0.547 0.427 0.353 0.330 0.343 0.337 0.797 0.493 0.647 BACKWARD: 0.437 0.397 0.387 1.050 STANDARD DEVIATIONS: FORWARD: 0.024 0.021 0.021 0.041 0.022 0.017 0.019 BACKWARD: 0.009 0.021 0.009 0.019 0.021 0.041 0.100 MAXIMUM COEFF AT EACH ECCENTRICITY: 0.780 0.570 0.450 0.520 0.670 0.850 1.140 MINIMUM COEFF AT EACH ECCENTRICITY: 0.430 0.370 0.380 0.300 0.300 0.320 0.310

The OVERALL MAXIMUM coefficient is: 1.140

The OVERALL MINIMUM coefficient is: 0.300

Table 47: Data for Ring 12 at 689 kPa 0.750 0.500 0.250 0.000 0.250 0.500 0.750 SEAL POSITIONS: FRICTION COEFFICIENT AVERAGES: 0.303 FORWARD: 0.773 0.500 0.357 0.320 0.297 0.297 BACKWARD: 0.470 0.437 0.507 0.607 0.680 0.797 1.107 **STANDARD DEVIATIONS:** FORWARD: 0.042 0.051 BACKWARD: 0.014 0.005 0.104 0.037 0.040 0.038 0.019 0.068 0.021 0.037 0.021 0.040 MAXIMUM COEFF AT EACH ECCENTRICITY: 0.830 0.550 0.560 0.630 0.720 0.820 1.140 MINIMUM COEFF AT EACH ECCENTRICITY: 0.450 0.430 0.210 0.270 0.240 0.270 0.290 The OVERALL MAXIMUM coefficient is: 1.140 The OVERALL MINIMUM coefficient is: 0.210 Table 48: Data for Ring 12 at 1378 kPa SEAL POSITIONS: 0.750 0.500 0.250 0.000 0.250 0.500 0.750 FRICTION COEFFICIENT AVERAGES:

FORWARD: 0.743 0.473 0.357 0.243 0.273 0.293 0.247 BACKWARD: 0.397 0.420 0.490 0.547 0.630 0.760 1.033 STANDARD DEVIATIONS: FORWARD: 0.009 0.021 0.012 0.012 0.026 0.005 0.005 BACKWARD: 0.009 0.000 0.008 0.012 0.008 0.008 0.019 MAXIMUM COEFF AT EACH ECCENTRICITY: 0.750 0.500 0.500 0.560 0.640 0.770 1.060 MINIMUM COEFF AT EACH ECCENTRICITY: 0.390 0.420 0.340 0.280 0.210 0.240 0.270

The OVERALL MAXIMUM coefficient is: 1.060

The OVERALL MINIMUM coefficient is: 0.210

 Table 49: Data for Ring 12 at 2068 kPa

 SEAL POSITIONS: 0.750 0.500 0.250 0.000 0.250 0.500 0.750

 FRICTION COEFFICIENT AVERAGES:

 FORWARD: 0.780 0.503 0.363 0.307 0.270 0.240 0.277

 BACKWARD: 0.780 0.503 0.363 0.307 0.270 0.240 0.277

 BACKWARD: 0.780 0.503 0.363 0.307 0.647 0.747 0.990

 STANDARD DEVIATIONS:

 FORWARD: 0.029 0.009 0.012 0.009 0.008 0.008 0.012

 BACKWARD: 0.029 0.009 0.012 0.009 0.021 0.009 0.012 0.008

 MAXIMUM COEFF AT EACH ECCENTRICITY:

 0.810 0.510 0.510 0.580 0.660 0.760 1.000

 MINIMUM COEFF AT EACH ECCENTRICITY:

 0.390 0.420 0.350 0.300 0.260 0.230 0.260

 The OVERALL MAXIMUM coefficient is: 1.000

 The OVERALL MINIMUM coefficient is: 0.230

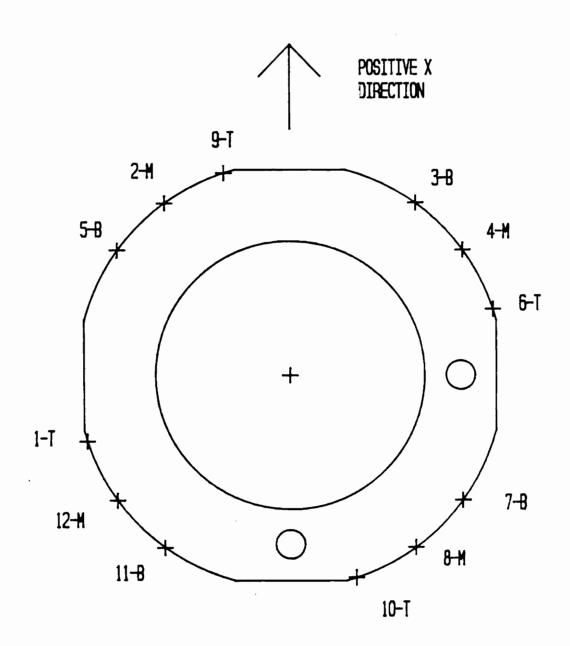
Table 50: Data for Ring 12 at 2757 kPa

SEAL POSITIONS:	0.750	0.500	0.250	0.000	0.250	0.500	0.750
FRICTION COEFFICIENT AV	ERAGES:						
FORWARD: BACKWARD:		0.460 0.430		0.290 0.557	0.270 0.627		0.267 1.003
STANDARD DEVIATIONS:							
FORWARD: BACKWARD:	0.017 0.025	0.008 0.008	0.009 0.005	0.000 0.009	0.000 0.005	0.012 0.000	0.021 0.005
MAXIMUM COEFF AT EACH E	CCENTRI	CITY:					
	0.760	0.470	0.490	0.570	0.630	0.750	1.010
MINIMUM COEFF AT EACH E	CCENTRI	CITY:					
	0.360	0.420	0.340	0.290	0.270	0.240	0.240
The OVERALL MAXIMUM coe	fficien	t is: 1	.010				
The OVERALL MINIMUM coe	fficien	t is: 0	.240				

APPENDIX C

Pressure Data for Individual Rings

Note: The pressures presented in the following tables are off by a factor of 2.26 due to incorrect voltage supplied to the probes which was dicovered after the creation of these tables. The correct pressure can be obtained by multiplying the given pressures by this factor.



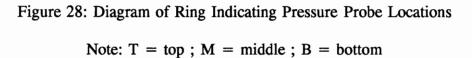


Table	51:
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	ECCENTRICITY RATIO									
probe	0.75	0.50	0.25	0.00	0.25	0.50	0.75			
1	0.00	0.00	0.00	0.00	0.00	0.00	0.00			
2	22.83	23.72	24.81	26.06	27.28	26.26	27.02			
3	25.25	24.56	23.78	22.60	21.58	20.48	20.48			
4	0.00	0.00	0.00	0.00	0.00	0.00	0.00			
5	0.00	0.00	0.00	0.00	0.00	0.00	0.00			
6	32.20	32.26	32.14	31.83	31.53	31.22	31.13			
7	25.91	26.95	27.16	26.53	25.87	24.34	23.82			
8	25.02	25.10	24.94	24.47	24.11	23.56	23.32			
9	0.00	0.00	0.00	0.00	0.00	0.00	0.00			
10	32.48	32.22	32.00	31.57	31.37	31.23	31.08			
11	16.94	14.85	13.66	12.86	12.33	12.07	11.90			
12	0.00	0.00	0.00	0.00	0.00	0.00	0.00			

Pressure Data (N/sq cm) for Ring #1 at 689 kPa

	ECCENTRICITY RATIO									
probe	0.75	0.50	0.25	0.00	0.25	0.50	0.75			
1	0.00	0.00	0.00	0.00	0.00	0.00	0.00			
2	37.44	38.89	39.98	40.87	41.80	42.75	44.30			
3	44.39	42.31	39.94	37.33	34.92	33.90	34.56			
4	0.00	0.00	0.00	0.00	0.00	0.00	0.00			
5	0.00	0.00	0.00	0.00	0.00	0.00	0.00			
6	58.59	58.74	58.50	57.89	57.04	56.43	56.34			
7	41.96	43.81	42.94	41.47	39.70	38.34	37.36			
8	45.31	45.58	44.99	44.13	43.06	42.35	41.84			
9	0.00	0.00	0.00	0.00	0.00	0.00	0.00			
10	58.21	57.81	57.49	57.01	56.10	55.64	55.22			
11	30.13	26.29	23.44	21.68	20.49	19.66	19.29			
12	0.00	0.00	0.00	0.00	0.00	0.00	0.00			

Table 52:

Pressure Data (N/sq cm) for Ring #1 at 1378 kPa

Note: Zero pressure indicates no data available for probe

Table	53:
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			ECC	ENTRICITY	RATIO		
probe	0.75	0.50	0.25	0.00	0.25	0.50	0.75
1	0.00	0.00	0.00	0.00	0.00	0.00	0.00
2	48.79	48.99	49.06	49.38	50.28	51.20	52.52
3	57.16	52.26	47.45	44.67	43.49	44.39	48.67
4	0.00	0.00	0.00	0.00	0.00	0.00	0.00
5	0.00	0.00	0.00	0.00	0.00	0.00	0.00
6	83.34	82.52	81.28	80.55	80.64	81.13	82.04
7	49.38	46.94	44.22	42.34	40.99	40.32	40.11
8	65.68	64.22	62.13	60.36	59.06	58.23	57.44
9	0.00	0.00	0.00	0.00	0.00	0.00	0.00
10	86.24	85.07	83.45	81.68	80.69	80.20	79.29
11	42.83	34.14	30.26	28.67	27.58	26.95	26.62
12	0.00	0.00	0.00	0.00	0.00	0.00	0.00

Pressure	Data	(N)	/sa	Cm)	for	Ring	#1	at	2068	kPa

Table 54:

Pressure Data (N/sq cm) for Ring #1 at 2757 kPa

		ECCENTRICITY RATIO									
probe	0.75	0.50	0.25	0.00	0.25	0.50	0.75				
1	0.00	0.00	0.00	0.00	0.00	0.00	0.00				
2	68.12	68.75	71.12	73.17	75.81	78.05	83.20				
3	86.12	78.70	73.39	67.27	63.03	61.40	62.62				
4	0.00	0.00	0.00	0.00	0.00	0.00	0.00				
5	0.00	0.00	0.00	0.00	0.00	0.00	0.00				
6	112.02	111.77	111.14	109.50	107.95	107.07	106.88				
7	72.43	73.37	71.87	69.61	67.55	66.09	65.54				
8	84.12	84.27	83.17	81.48	79.98	78.99	78.76				
9	0.00	0.00	0.00	0.00	0.00	0.00	0.00				
10	108.98	108.69	107.81	106.53	104.90	103.99	103.68				
11	59.87	50.75	42.63	38.55	36.36	35.34	34.41				
12	0.00	0.00	0.00	0.00	0.00	0.00	0.00				

Note: Zero pressure indicates no data available for probe

Table	55:	
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	Flessule	Data (N/S	q cm) 101	KING #2	at 689 KP	a 	
			ECC	ENTRICITY	RATIO		
probe	0.75	0.50	0.25	0.00	0.25	0.50	0.75
1	0.00	0.00	0.00	0.00	0.00	0.00	0.00
2	19.83	19.99	20.22	20.72	21.38	22.70	24.54
3	21.05	19.99	18.93	18.03	17.46	17.34	17.79
4	0.00	0.00	0.00	0.00	0.00	0.00	0.00
5	9.57	9.99	10.40	11.06	11.50	12.10	12.39
6	28.79	28.67	28.49	28.34	28.22	28.10	28.00
7	22.81	22.95	22.67	22.36	22.11	22.01	21.97
8	21.87	21.71	21.47	21.28	21.12	21.00	21.00
9	0.00	0.00	0.00	0.00	0.00	0.00	0.00
10	29.03	28.86	28.69	28.60	28.49	28.35	28.29
11	10.31	11.20	11.24	10.91	10.77	10.71	10.71
12	0.00	0.00	0.00	0.00	0.00	0.00	0.00

Pressure	Data	(N/	sq	Cm)	for	Ring	# 2	at	689	kPa	
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Table 56:

Pressure Data (N/sq cm) for Ring #2 at 1378 kPa

	ECCENTRICITY RATIO								
probe	0.75	0.50	0.25	0.00	0.25	0.50	0.75		
1	0.00	0.00	0.00	0.00	0.00	0.00	0.00		
2	31.70	32.03	32.86	34.21	36.29	40.28	43.78		
3	38.51	36.06	33.49	31.74	30.52	29.90	30.19		
4	0.00	0.00	0.00	0.00	0.00	0.00	0.00		
5	17.71	19.50	24.64	33.46	41.81	49.13	42.11		
6	52.00	51.73	51.27	50.94	50.72	50.51	50.18		
7	35.87	36.63	36.88	36.70	36.42	36.21	35.45		
8	38.65	38.57	38.37	38.22	38.06	38.02	37.67		
9	0.00	0.00	0.00	0.00	0.00	0.00	0.00		
10	50.69	50.34	50.03	49.94	49.74	49.72	49.46		
11	14.42	17.10	17.60	17.34	17.14	17.00	16.91		
12	0.00	0.00	0.00	0.00	0.00	0.00	0.00		

Note: Zero pressure indicates no data available for probe

Table	57:	
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	Pressure	Data (N/s	q cm) for	Ring #2	at 2068 k	:Pa	
			ECC	ENTRICITY	RATIO		
probe	0.75	0.50	0.25	0.00	0.25	0.50	0.75
1	0.00	0.00	0.00	0.00	0.00	0.00	0.00
2	44.54	44.17	47.08	49.68	52.52	59.48	66.97
3	55.12	49.98	50.02	47.08	45.08	43.90	43.57
4	0.00	0.00	0.00	0.00	0.00	0.00	0.00
5	26.48	27.37	41.70	64.52	80.12	82.41	80.36
6	76.42	75.48	76.09	75.51	75.23	75.11	74.78
7	48.75	48.68	52.55	52.93	53.07	53.45	54.78
8	55.75	55.36	56.38	56.42	56.62	56.93	58.11
9	0.00	0.00	0.00	0.00	0.00	0.00	0.00
10	73.42	73.05	72.71	72.48	72.59	72.54	72.88
11	20.58	24.60	24.13	23.83	23.73	23.60	23.77
12	0.00	0.00	0.00	0.00	0.00	0.00	0.00

Table 58:

Pressure Data (N/sq cm) for Ring #2 at 2757 kPa

	ECCENTRICITY RATIO								
probe	0.75	0.50	0.25	0.00	0.25	0.50	0.75		
1	0.00	0.00	0.00	0.00	0.00	0.00	0.00		
2	54.93	55.29	55.45	68.62	81.05	90.72	94.42		
3	60.14	55.08	52.22	61.48	58.10	56.38	55.48		
4	0.00	0.00	0.00	0.00	0.00	0.00	0.00		
5	30.67	30.79	30.73	80.54	108.12	108.68	105.86		
6	97.13	96.74	96.37	100.14	99.26	98.83	98.87		
7	51.88	50.46	49.48	69.47	69.89	70.55	73.09		
8	71.08	70.05	69.26	74.74	74.94	75.61	77.81		
9	0.00	0.00	0.00	0.00	0.00	0.00	0.00		
10	97.32	96.58	96.10	95.24	95.05	95.16	95.93		
11	33.02	31.19	30.56	30.43	30.00	30.00	30.73		
12	0.00	0.00	0.00	0.00	0.00	0.00	0.00		

Note: Zero pressure indicates no data available for probe

	Pressure	Data (N/s	q cm) for	Ring #3	at 689 kP	a 	
			ECC	ENTRICITY	RATIO		
probe	0.75	0.50	0.25	0.00	0.25	0.50	0.75
1	0.00	0.00	0.00	0.00	0.00	0.00	0.00
2	21.94	22.47	22.53	22.30	21.71	20.95	20.12
3	23.05	23.13	22.68	21.74	20.36	18.89	17.58
4	0.00	0.00	0.00	0.00	0.00	0.00	0.00
5	10.55	11.20	11.86	12.69	13.22	12.13	10.67
6	26.67	27.00	27.00	26.79	26.42	26.00	25.54
7	23.26	24.45	24.79	24.62	23.99	23.19	22.22
8	23.36	23.99	23.99	23.56	22.85	21.91	20.92
9	24.99	25.40	25.45	25.17	24.76	24.25	23.61
10	27.69	28.03	28.03	27.69	27.29	26.87	26.41
11	0.00	0.00	0.00	0.00	0.00	0.00	0.00
12	0.00	0.00	0.00	0.00	0.00	0.00	0.00

Table 60:

Pressure Data (N/sq cm) for Ring #3 at 1378 kPa

	ECCENTRICITY RATIO								
probe	0.75	0.50	0.25	0.00	0.25	0.50	0.75		
1	0.00	0.00	0.00	0.00	0.00	0.00	0.00		
2	36.68	37.41	37.84	39.29	38.47	36.78	34.77		
3	41.08	40.84	40.31	40.47	38.27	35.29	32.27		
4	0.00	0.00	0.00	0.00	0.00	0.00	0.00		
5	20.36	21.31	22.47	25.94	28.47	25.41	21.58		
6	48.05	48.38	48.54	48.75	50.24	49.24	47.90		
7	39.49	41.65	42.52	42.97	43.63	41.86	39.38		
8	41.88	42.87	43.14	43.06	43.65	41.72	39.20		
9	46.24	46.73	46.88	47.01	48.42	47.24	45.57		
10	49.09	49.52	49.55	49.57	50.91	49.92	48.49		
11	0.00	0.00	0.00	0.00	0.00	0.00	0.00		
12	0.00	0.00	0.00	0.00	0.00	0.00	0.00		

Note: Zero pressure indicates no data available for probe

Table 61:	:
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	Pressure	Data (N/s	q cm) for	Ring #3	at 2068 k	.Pa				
	ECCENTRICITY RATIO									
probe	0.75	0.50	0.25	0.00	0.25	0.50	0.75			
1	0.00	0.00	0.00	0.00	0.00	0.00	0.00			
2	51.53	52.95	53.84	53.67	52.45	49.78	47.01			
3	58.99	58.54	57.89	55.85	52.87	48.14	44.51			
4	0.00	0.00	0.00	0.00	0.00	0.00	0.00			
5	31.06	33.08	35.78	40.36	42.05	37.18	31.80			
6	69.25	70.22	70.92	70.92	70.04	68.19	66.52			
7	57.25	60.59	62.09	62.26	61.01	57.84	54.60			
8	60.44	62.13	62.76	62.37	60.71	57.21	54.02			
9	67.18	68.38	68.97	69.05	68.15	66.08	64.31			
10	70.51	71.57	71.91	71.74	70.77	68.55	66.73			
11	0.00	0.00	0.00	0.00	0.00	0.00	0.00			
12	0.00	0.00	0.00	0.00	0.00	0.00	0.00			

Data (N/sg cm) for Ring #3 at 2068 kPa

Note: Zero pressure indicates no data available for probe

Table 62:

Pressure Data (N/sq cm) for Ring #3 at 2757 kPa

	ECCENTRICITY RATIO									
probe	0.75	0.50	0.25	0.00	0.25	0.50	0.75			
1	0.00	0.00	0.00	0.00	0.00	0.00	0.00			
2	69.28	70.00	70.86	70.83	69.21	65.78	61.39			
3	80.53	77.68	76.21	73.56	69.48	63.60	57.77			
4	0.00	0.00	0.00	0.00	0.00	0.00	0.00			
5	43.24	45.59	49.18	56.20	55.66	49.13	41.64			
6	93.98	94.19	94.73	95.01	94.31	92.37	89.27			
7	76.40	80.16	81.73	81.90	80.23	76.30	71.21			
8	82.34	83.45	83.92	83.29	81.28	76.98	71.71			
9	91.58	92.25	92.76	92.86	92.27	89.91	86.63			
10	95.05	95.76	95.90	95.59	94.82	92.28	88.75			
11	0.00	0.00	0.00	0.00	0.00	0.00	0.00			
12	0.00	0.00	0.00	0.00	0.00	0.00	0.00			

Note: Zero pressure indicates no data available for probe

	Pressure	Data (N/s	q cm) for	Ring #4	at 689 kP	a	
			ECC	ENTRICITY	RATIO		
probe	0.75	0.50	0.25	0.00	0.25	0.50	0.75
1	0.00	0.00	0.00	0.00	0.00	0.00	0.00
2	23.16	23.22	23.06	22.56	21.94	21.31	20.78
3	23.34	22.81	21.91	20.56	19.17	18.07	17.58
4	0.00	0.00	0.00	0.00	0.00	0.00	0.00
5	13.58	14.09	14.74	14.59	13.79	13.08	12.60
6	28.98	28.95	28.82	28.58	28.34	28.06	27.79
7	22.98	23.33	23.33	22.67	21.83	21.10	20.61
8	23.99	23.95	23.68	23.05	22.30	21.51	20.92
9	0.00	0.00	0.00	0.00	0.00	0.00	0.00
10	30.54	30.34	30.17	29.94	29.60	29.29	28.98
11	14.22	13.99	13.82	13.42	12.99	12.60	12.33
12	0.00	0.00	0.00	0.00	0.00	0.00	0.00

Table 63:

Table 64:

Pressure Data (N/sq cm) for Ring #4 at 1378 kPa _____

			ECC	ENTRICITY	RATIO		
probe	0.75	0.50	0.25	0.00	0.25	0.50	0.75
1	0.00	0.00	0.00	0.00	0.00	0.00	0.00
2	39.55	39.88	39.69	39.16	38.04	36.22	34.74
3	43.12	42.14	40.35	38.31	31.66	32.88	31.21
4	0.00	0.00	0.00	0.00	0.00	0.00	0.00
5	25.94	27.07	28.71	30.55	33.14	29.06	25.89
6	53.27	53.49	53.49	53.27	52.64	51.48	50.48
7	37.61	38.55	38.90	38.51	37.29	35.27	33.46
8	44.44	44.64	44.36	43.54	42.08	40.03	38.06
9	0.00	0.00	0.00	0.00	0.00	0.00	0.00
10	56.10	55.19	55.13	54.70	53.93	52.94	51.60
11	23.37	23.80	23.83	23.53	22.81	21.68	20.82
12	0.00	0.00	0.00	0.00	0.00	0.00	0.00

	Pressure	Data (N/s	q cm) for	Ring #4	at 2068 k	Pa	
			ECC	ENTRICITY	RATIO		
probe	0.75	0.50	0.25	0.00	0.25	0.50	0.75
1	0.00	0.00	0.00	0.00	0.00	0.00	0.00
2	55.72	56.25	55.92	54.86	52.82	50.74	48.43
3	62.79	61.28	58.38	54.75	50.59	47.08	44.35
4	0.00	0.00	0.00	0.00	0.00	0.00	0.00
5	38.22	40.48	42.77	46.27	51.26	52.39	42.44
6	76.81	77.33	77.33	76.75	75.20	73.69	72.47
7	51.68	53.59	53.80	53.14	51.47	48.99	46.42
8	64.30	64.93	64.18	62.72	60.48	57.60	54.88
9	0.00	0.00	0.00	0.00	0.00	0.00	0.00
10	79.00	79.43	81.85	78.21	76.98	75.02	73.53
11	32.58	33.45	33.55	32.98	31.79	30.53	29.10
12	0.00	0.00	0.00	0.00	0.00	0.00	0.00

Tab	le	65	:
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Table 66:

Pressure Data (N/sq cm) for Ring #4 at 2757 kPa

	ECCENTRICITY RATIO							
probe	0.75	0.50	0.25	0.00	0.25	0.50	0.75	
1	0.00	0.00	0.00	0.00	0.00	0.00	0.00	
2	74.36	74.62	73.50	71.06	68.85	66.31	63.83	
3	85.43	82.78	78.05	71.72	66.38	61.60	58.79	
4	0.00	0.00	0.00	0.00	0.00	0.00	0.00	
5	52.93	55.57	58.01	64.85	73.91	77.80	61.16	
6	103.94	104.03	102.90	101.05	99.75	98.56	97.41	
7	68.18	70.55	70.06	68.39	66.30	63.58	61.53	
8	86.99	87.58	85.49	82.74	79.90	76.55	74.11	
9	0.00	0.00	0.00	0.00	0.00	0.00	0.00	
10	106.56	106.44	104.59	103.02	101.37	99.60	98.63	
11	42.40	44.09	44.29	42.63	41.34	39.74	38.25	
12	0.00	0.00	0.00	0.00	0.00	0.00	0.00	

Note: Zero pressure indicates no data available for probe

Table	67:
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	TICODUIC		q cm) 101	ning #0		u 	
			ECC	ENTRICITY	RATIO		
probe	0.75	0.50	0.25	0.00	0.25	0.50	0.75
1	0.00	0.00	0.00	0.00	0.00	0.00	0.00
2	20.45	20.78	20.95	20.92	20.72	20.16	19.66
3	18.07	17.95	17.62	17.09	16.48	15.71	15.38
4	0.00	0.00	0.00	0.00	0.00	0.00	0.00
5	10.10	10.67	11.44	12.81	14.29	11.71	10.64
6	26.03	26.15	26.27	26.27	26.12	25.88	25.54
7	21.69	22.53	23.02	23.09	22.67	21.87	21.28
8	21.04	21.39	21.55	21.51	21.16	20.49	19.94
9	24.25	24.40	24.53	24.68	24.81	25.35	26.61
10	27.64	27.64	27.66	27.66	27.47	27.15	26.84
11	0.00	0.00	0.00	0.00	0.00	0.00	0.00
12	0.00	0.00	0.00	0.00	0.00	0.00	0.00

Pressure Data (N/sq cm) for 1	kina.	ŦO	aτ	689	кга
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Table 68:

Pressure Data (N/sq cm) for Ring #6 at 1378 kPa

			ECC	ENTRICITY	RATIO		
probe	0.75	0.50	0.25	0.00	0.25	0.50	0.75
1	0.00	0.00	0.00	0.00	0.00	0.00	0.00
2	34.41	34.90	35.23	35.36	34.80	34.01	32.82
3	32.56	32.03	31.25	30.27	28.88	27.78	26.89
4	0.00	0.00	0.00	0.00	0.00	0.00	0.00
5	19.08	19.94	21.25	24.28	28.26	28.71	21.58
6	46.74	47.08	47.41	47.56	46.96	46.50	45.86
7	36.28	37.61	38.51	38.79	38.06	36.91	35.31
8	38.57	39.12	39.56	39.75	38.69	37.70	36.40
9	44.75	45.14	45.65	46.08	46.06	47.37	51.06
10	48.75	48.89	49.23	49.26	48.46	47.89	47.29
11	0.00	0.00	0.00	0.00	0.00	0.00	0.00
12	0.00	0.00	0.00	0.00	0.00	0.00	0.00

Note: Zero pressure indicates no data available for probe

	Pressure	Data (N/s	q cm) for	Ring #6	at 2068 k	Pa	
			ECC	ENTRICITY	RATIO		
probe	0.75	0.50	0.25	0.00	0.25	0.50	0.75
1	0.00	0.00	0.00	0.00	0.00	0.00	0.00
2	49.42	50.01	50.64	50.61	49.85	48.59	47.31
3	47.49	46.06	45.04	43.57	41.82	40.02	39.25
4	0.00	0.00	0.00	0.00	0.00	0.00	0.00
5	28.89	30.61	33.20	37.48	42.94	42.47	32.96
6	68.86	69.37	69.86	70.01	69.56	68.52	67.85
7	52.16	54.08	55.12	55.26	54.36	52.62	51.19
8	57.40	58.23	58.63	58.55	57.60	55.79	54.29
9	66.69	67.49	68.00	68.56	69.20	71.18	75.82
10	71.26	71.77	71.88	71.88	71.43	70.06	69.15
11	0.00	0.00	0.00	0.00	0.00	0.00	0.00
12	0.00	0.00	0.00	0.00	0.00	0.00	0.00

Table 70:

Pressure Data (N/sq cm) for Ring #6 at 2757 kPa

	 		ECC	ENTRICITY	RATIO		
probe	0.75	0.50	0.25	0.00	0.25	0.50	0.75
1	0.00	0.00	0.00	0.00	0.00	0.00	0.00
2	64.66	65.38	65.75	65.75	64.99	63.37	62.81
3	61.89	59.97	58.01	56.01	54.02	52.02	51.49
4	0.00	0.00	0.00	0.00	0.00	0.00	0.00
5	39.11	41.16	45.02	52.69	56.61	52.96	44.99
6	91.12	91.73	92.52	92.61	91.97	90.91	90.30
7	68.39	70.10	71.39	71.39	70.41	69.23	69.57
8	76.55	77.10	77.85	77.38	76.08	74.42	73.64
9	88.99	89.58	90.94	91.63	92.32	96.58	101.60
10	94.08	94.22	95.13	94.90	93.79	92.68	60.37
11	0.00	0.00	0.00	0.00	0.00	0.00	0.00
12	0.00	0.00	0.00	0.00	0.00	0.00	0.00

Note: Zero pressure indicates no data available for probe

	Pressure	Data (N/s	q cm) for	Ring #8	at 689 kP	a	
			ECC	ENTRICITY	RATIO		
probe	0.75	0.50	0.25	0.00	0.25	0.50	0.75
1	0.00	0.00	0.00	0.00	0.00	0.00	0.00
2	21.34	21.84	22.20	22.47	22.50	22.50	22.47
3	18.44	18.52	18.48	18.36	18.11	18.03	18.20
4	0.00	0.00	0.00	0.00	0.00	0.00	0.00
5	11.71	12.39	13.37	14.68	15.19	15.19	14.65
6	26.70	27.09	27.37	27.55	27.55	27.52	27.55
7	23.05	24.24	25.07	25.39	25.45	25.45	25.39
8	21.83	22.42	22.93	23.13	23.17	23.13	23.09
9	0.00	0.00	0.00	0.00	0.00	0.00	0.00
10	28.49	28.69	29.00	29.09	29.09	28.95	28.89
11	14.78	14.59	14.72	14.88	15.02	15.81	15.51
12	0.00	0.00	0.00	0.00	0.00	0.00	0.00

Table 71:

Table 72:

Pressure Data (N/sq cm) for Ring #8 at 1378 kPa

	ECCENTRICITY RATIO								
probe	0.75	0.50	0.25	0.00	0.25	0.50	0.75		
1	0.00	0.00	0.00	0.00	0.00	0.00	0.00		
2	37.61	38.56	39.59	40.41	40.97	41.17	41.34		
3	33.90	33.94	34.23	34.43	34.43	34.43	35.00		
4	0.00	0.00	0.00	0.00	0.00	0.00	0.00		
5	22.88	24.31	26.72	30.46	36.44	38.55	31.68		
6	51.09	51.67	52.36	53.15	53.67	54.00	54.25		
7	40.15	42.48	44.57	45.97	46.56	46.73	47.15		
8	41.95	42.91	44.17	45.31	45.86	46.18	46.33		
9	0.00	0.00	0.00	0.00	0.00	0.00	0.00		
10	53.59	53.71	54.33	55.10	55.30	55.64	55.67		
11	0.00	0.00	0.00	0.00	0.00	0.00	0.00		
12	0.00	0.00	0.00	0.00	0.00	0.00	0.00		

Note: Zero pressure indicates no data available for probe

	Pressure	Data (N/S	q cm) for	Ring #8	at 2068 k	Pa	
			ECC	ENTRICITY	RATIO		
probe	0.75	0.50	0.25	0.00	0.25	0.50	0.75
1	0.00	0.00	0.00	0.00	0.00	0.00	0.00
2	46.25	47.70	49.06	49.98	51.03	52.09	53.28
3	38.80	39.70	40.67	41.33	42.59	44.31	47.49
4	0.00	0.00	0.00	0.00	0.00	0.00	0.00
5	27.37	28.53	29.63	30.25	31.35	32.45	33.97
6	70.56	72.68	74.29	74.72	75.17	75.60	76.12
7	40.08	41.93	44.43	46.73	50.04	53.00	55.89
8	52.24	54.29	56.02	56.85	58.43	59.81	61.11
9	0.00	0.00	0.00	0.00	0.00	0.00	0.00
10	76.53	76.44	76.95	76.58	76.90	77.04	76.92
11	0.00	0.00	0.00	0.00	0.00	0.00	0.00
12	0.00	0.00	0.00	0.00	0.00	0.00	0.00

Pressure Data (N/sq cm) for Ring #8 at 2068 kPa

Note: Zero pressure indicates no data available for probe

NO DATA FOR RING #8 AT 2757 kPa

Table	74:
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	Pressure	Data (N/s	q cm) for	Ring #9	at 689 kP	a					
		ECCENTRICITY RATIO									
probe	0.75	0.50	0.25	0.00	0.25	0.50	0.75				
1	0.00	0.00	0.00	0.00	0.00	0.00	0.00				
2	23.69	23.46	22.96	22.30	21.57	20.95	20.49				
3	26.11	25.05	23.62	21.99	20.48	19.34	18.73				
4	0.00	0.00	0.00	0.00	0.00	0.00	0.00				
5	13.49	13.58	13.28	12.84	12.36	12.07	11.89				
6	0.00	0.00	0.00	0.00	0.00	0.00	0.00				
7	24.20	24.31	23.71	23.02	22.25	21.69	21.35				
8	24.51	24.23	23.56	22.77	21.87	21.08	20.53				
9	0.00	0.00	0.00	0.00	0.00	0.00	0.00				
10	29.74	29.54	29.17	28.72	28.29	27.84	27.52				
11	0.00	0.00	0.00	0.00	0.00	0.00	0.00				
12	0.00	0.00	0.00	0.00	0.00	0.00	0.00				

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Note: Zero pressure indicates no data available for probe

Table 75:

Pressure Data (N/sq cm) for Ring #9 at 1378 kPa ______

	ECCENTRICITY RATIO									
probe	0.75	0.50	0.25	0.00	0.25	0.50	0.75			
1	0.00	0.00	0.00	0.00	0.00	0.00	0.00			
2	40.38	40.44	40.18	39.26	37.94	36.26	34.74			
3	48.59	47.32	45.53	42.92	39.82	36.64	34.56			
4	0.00	0.00	0.00	0.00	0.00	0.00	0.00			
5	26.27	26.95	28.41	29.78	30.22	28.23	27.28			
6	0.00	0.00	0.00	0.00	0.00	0.00	0.00			
7	39.87	41.37	41.96	41.19	39.77	37.89	36.53			
8	45.27	45.23	44.91	43.85	42.16	40.11	38.30			
9	0.00	0.00	0.00	0.00	0.00	0.00	0.00			
10	53.71	53.45	53.22	52.62	51.71	50.63	49.49			
11	0.00	0.00	0.00	0.00	0.00	0.00	0.00			
12	0.00	0.00	0.00	0.00	0.00	0.00	0.00			

	Pressure	Data (N/s	q cm) for	Ring #9	at 2068 k	Pa	
			ECC	ENTRICITY	RATIO		
probe	0.75	0.50	0.25	0.00	0.25	0.50	0.75
1	0.00	0.00	0.00	0.00	0.00	0.00	0.00
2	57.83	51.23	57.10	55.39	53.81	51.43	50.67
3	71.97	69.76	66.66	62.18	58.38	53.44	50.43
4	0.00	0.00	0.00	0.00	0.00	0.00	0.00
5	39.85	41.40	43.06	46.90	51.35	51.15	45.71
6	0.00	0.00	0.00	0.00	0.00	0.00	0.00
7	56.34	58.88	59.09	57.91	56.48	54.25	55.12
8	66.78	66.90	65.76	63.83	61.93	58.86	58.03
9	0.00	0.00	0.00	0.00	0.00	0.00	0.00
10	77.92	77.87	76.95	75.81	74.82	72.94	72.45
11	0.00	0.00	0.00	0.00	0.00	0.00	0.00
12	0.00	0.00	0.00	0.00	0.00	0.00	0.00

Table	7	6	:	
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Table 77:

Pressure Data (N/sq cm) for Ring #9 at 2757 kPa

	ECCENTRICITY RATIO								
probe	0.75	0.50	0.25	0.00	0.25	0.50	0.75		
1	0.00	0.00	0.00	0.00	0.00	0.00	0.00		
2	74.59	74.39	73.60	71.69	69.31	66.67	64.36		
3	95.06	91.10	86.98	81.19	75.43	70.05	64.79		
4	0.00	0.00	0.00	0.00	0.00	0.00	0.00		
5	52.78	54.56	56.32	61.93	70.08	68.77	59.14		
6	0.00	0.00	0.00	0.00	0.00	0.00	0.00		
7	71.73	74.94	74.97	74.00	72.01	69.78	69.47		
8	87.54	87.27	86.05	84.04	80.96	77.62	75.01		
9	0.00	0.00	0.00	0.00	0.00	0.00	0.00		
10	101.54	101.20	100.49	99.63	97.75	95.39	93.54		
11	0.00	0.00	0.00	0.00	0.00	0.00	0.00		
12	0.00	0.00	0.00	0.00	0.00	0.00	0.00		

Note: Zero pressure indicates no data available for probe

Ta	ble	78:
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	Pressure	Data (N/s	q cm) for	Ring #10	at 689	kPa	
			ECC	ENTRICITY	RATIO		
probe	0.75	0.50	0.25	0.00	0.25	0.50	0.75
1	0.00	0.00	0.00	0.00	0.00	0.00	0.00
2	20.22	20.95	21.57	21.84	21.81	21.51	21.01
3	16.97	17.34	17.67	17.71	17.50	17.26	17.09
4	0.00	0.00	0.00	0.00	0.00	0.00	0.00
5	10.16	10.88	11.65	12.21	12.07	11.65	11.09
6	6.35	6.29	6.29	6.32	6.32	6.44	6.50
7	23.68	25.11	26.08	26.50	26.40	25.94	25.32
8	20.84	21.75	22.46	22.73	22.69	22.26	21.67
9	24.14	24.63	24.99	25.17	25.20	25.12	25.73
10	27.09	27.49	27.78	27.89	27.86	27.58	27.27
11	0.00	0.00	0.00	0.00	0.00	0.00	0.00
12	0.00	0.00	0.00	0.00	0.00	0.00	0.00

Pressure Data (N/sq cm) for Ring #10 at 689 kPa

Note: Zero pressure indicates no data available for probe

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Table 79:

Pressure Data (N/sq cm) for Ring #10 at 1378 kPa

			ECC	ENTRICITY	RATIO		
probe	0.75	0.50	0.25	0.00	0.25	0.50	0.75
1	0.00	0.00	0.00	0.00	0.00	0.00	0.00
2	34.34	35.60	36.91	37.67	37.84	37.41	36.29
3	30.11	30.64	31.25	31.54	31.41	31.05	30.76
4	0.00	0.00	0.00	0.00	0.00	0.00	0.00
5	19.32	20.60	22.47	25.26	27.52	24.40	22.02
6	8.93	8.78	8.90	9.11	9.20	9.08	9.05
7	39.17	41.72	43.70	45.02	45.27	44.57	43.25
8	38.53	40.27	41.60	42.55	42.71	42.00	40.82
9	45.57	46.60	47.42	48.03	48.24	48.37	50.85
10	48.92	49.80	50.43	50.86	50.91	50.43	49.77
11	0.00	0.00	0.00	0.00	0.00	0.00	0.00
12	0.00	0.00	0.00	0.00	0.00	0.00	0.00

Note: Zero pressure indicates no data available for probe

Table	80:	
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	Pressure	Data (N/s	q cm) for	Ring #10	at 2068	kPa	
	* *		ECC	ENTRICITY	RATIO		
probe	0.75	0.50	0.25	0.00	0.25	0.50	0.75
<u> </u>	0.00	0.00	0.00	0.00	0.00	0.00	0.00
2	49.65	50.61	51.96	53.34	53.57	52.88	52.16
3	44.67	44.06	44.47	45.00	44.88	44.63	44.51
4	0.00	0.00	0.00	0.00	0.00	0.00	0.00
5	29.66	30.85	33.31	37.33	39.11	36.20	33.58
6	11.30	10.72	10.87	10.21	10.27	9.90	9.87
7	56.27	58.99	61.39	63.13	63.45	62.64	61.88
8	57.68	58.98	60.75	62.01	62.29	61.46	60.56
9	67.72	68.33	69.64	70.51	71.02	71.84	76.56
10	71.17	71.74	72.91	73.51	73.79	73.31	72.39
11	0.00	0.00	0.00	0.00	0.00	0.00	0.00
12	0.00	0.00	0.00	0.00	0.00	0.00	0.00

	Pressure	Data	(N/sc	(Cm)	for	Ring	#10	at	2068	kPa
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Table 81:

Pressure Data (N/sq cm) for Ring #10 at 2757 kPa

			ECC	ENTRICITY	RATIO		
probe	0.75	0.50	0.25	0.00	0.25	0.50	0.75
1	0.00	0.00	0.00	0.00	0.00	0.00	0.00
2 '	64.20	65.25	66.77	67.79	68.72	67.92	66.90
3	58.30	56.99	56.99	57.08	57.73	57.69	57.57
4	0.00	0.00	0.00	0.00	0.00	0.00	0.00
5	39.14	40.80	44.16	48.32	51.44	47.79	44.52
6	12.67	12.67	12.54	12.79	12.79	12.76	12.94
7	72.85	75.81	78.42	80.30	81.07	80.06	79.43
8	76.12	77.22	78.88	80.45	81.12	79.98	79.11
9	89.07	89.45	90.43	92.02	93.02	94.12	99.65
10	92.79	93.02	93.85	95.27	95.70	94.73	94.08
11	0.00	0.00	0.00	0.00	0.00	0.00	0.00
12	0.00	0.00	0.00	0.00	0.00	0.00	0.00

	Pressure	Data (N/s	q cm) for	Ring #11	at 689 J	kPa	
			ECC	ENTRICITY	RATIO		
probe	0.75	0.50	0.25	0.00	0.25	0.50	0.75
1	0.00	0.00	0.00	0.00	0.00	0.00	0.00
2	24.54	25.27	25.53	25.17	24.64	24.21	23.95
3	25.42	26.72	26.97	25.99	24.64	23.50	22.60
4	0.00	0.00	0.00	0.00	0.00	0.00	0.00
5	0.00	0.00	0.00	0.00	0.00	0.00	0.00
6	28.95	29.55	29.74	29.40	28.92	28.43	28.16
7	26.40	27.47	27.61	26.74	25.66	24.69	23.96
8	25.29	26.08	26.20	25.61	24.86	24.11	23.68
9	0.00	0.00	0.00	0.00	0.00	0.00	0.00
10	30.17	30.83	30.97	30.68	30.20	29.66	29.37
11	20.22	21.28	21.45	20.55	19.42	18.46	17.73
12	0.00	0.00	0.00	0.00	0.00	0.00	0.00

Pressure	Data	(N/	'sq	Cm)	for	Ring	#11	at	689	kPa	
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Table 83:

Pressure Data (N/sq cm) for Ring #11 at 1378 kPa

			ECC	ENTRICITY	RATIO		
probe	0.75	0.50	0.25	0.00	0.25	0.50	0.75
1	0.00	0.00	0.00	0.00	0.00	0.00	0.00
2	40.02	41.57	42.29	41.37	40.31	39.26	38.43
3	44.31	47.16	48.26	46.14	43.41	40.72	38.51
4	0.00	0.00	0.00	0.00	0.00	0.00	0.00
5	0.00	0.00	0.00	0.00	0.00	0.00	0.00
6	50.91	52.15	52.52	51.73	50.72	49.72	48.81
7	42.10	44.43	45.20	43.35	40.99	38.79	37.05
8	44.60	46.21	46.65	45.43	43.77	42.24	40.97
9	0.00	0.00	0.00	0.00	0.00	0.00	0.00
10	52.25	53.51	53.76	53.08	52.02	51.00	50.06
11	35.57	37.89	38.75	36.79	34.47	32.22	30.36
12	0.00	0.00	0.00	0.00	0.00	0.00	0.00

	Pressure	Data (N/s	q cm) for	Ring #11	at 2068	kPa	
			ECC	ENTRICITY	RATIO		
probe	0.75	0.50	0.25	0.00	0.25	0.50	0.75
1	0.00	0.00	0.00	0.00	0.00	0.00	0.00
2	55.55	57.93	59.58	60.44	59.91	57.93	56.41
3	63.36	67.60	70.17	71.07	69.15	63.81	59.16
4	0.00	0.00	0.00	0.00	0.00	0.00	0.00
5	0.00	0.00	0.00	0.00	0.00	0.00	0.00
6	72.56	74.63	75.93	76.21	75.23	73.05	71.29
7	57.94	61.39	63.24	63.69	62.16	57.67	53.94
8	63.75	66.35	67.77	67.92	66.78	63.51	60.79
9	0.00	0.00	0.00	0.00	0.00	0.00	0.00
10	73.82	76.10	77.27	77.41	76.70	74.36	72.51
11	51.38	54.76	56.72	57.18	55.26	50.82	46.87
12	0.00	0.00	0.00	0.00	0.00	0.00	0.00

Table 84:

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Note: Zero pressure indicates no data available for probe

Table 85:

Pressure Data (N/sq cm) for Ring #11 at 2757 kPa

			ECO	CENTRICITY	Y RATIO		
probe	0.75	0.50	0.25	0.00	0.25	0.50	0.75
1	0.00	0.00	0.00	0.00	0.00	0.00	0.00
2	75.02	77.69	79.21	80.56	80.66	78.51	76.24
3	87.51	92.85	95.30	96.28	93.87	86.08	79.43
4	0.00	0.00	0.00	0.00	0.00	0.00	0.00
5	0.00	0.00	0.00	0.00	0.00	0.00	0.00
6	99.23	101.30	102.02	102.72	102.18	99.41	96.31
7	77.93	82.42	83.89	84.27	82.15	75.74	70.51
8	87.19	90.38	91.17	91.56	90.38	85.77	81.71
9	0.00	0.00	0.00	0.00	0.00	0.00	0.00
10	100.34	102.94	103.39	103.91	103.68	100.52	97.35
11	71.30	75.44	77.20	77.53	75.05	68.45	62.91
12	0.00	0.00	0.00	0.00	0.00	0.00	0.00

APPENDIX D

Instrumentation Calibrations

PRESSURE (psi)	GAUGE READS (psi)
50	45
100	98
150	150
200	200
250	251
300	301
350	352
400	404
450	455
475	481

Table 86: Pressure Gauge Calibration

Table 87: Pressure Probe Calibrations

Probe No.	1	2	3	4	5	6
Sensitivity (mV/psi)	0.214	0.209	0.169	0.186	0.232	0.227
Probe No.	7	•		10	11	12
FIODE NO.	/	•	9	10	11	12

Zero Pressure Output: < ± 5% of Full Scale

LOAD (pounds)	READING (1b)	LOAD (pounds)	READING (1b)
10	10	450	428
20	20	500	474
30	30	550	520
40	40	600	566
50	50	650	612
100	98	700	658
150	146	750	704
200	194	800	750
250	241	850	796
300	289	900	842
350	335	950	889
400	381	993	928

Serial No. 191003

Table 89: Load Cell Calibration (-X Direction)

LOAD (pounds)	READING (1b)	LOAD (pounds)	READING (1b)
10	11	450	439
20	20	500	487
30	30	550	536
40	40	600	584
50	50	650	633
100	99	700	681
150	147	750	731
200	196	800	780
250	244	850	828
300	293	900	876
350	341	950	925
400	390	996	970

Serial No. 191010

APPENDIX E

Sample Calculation of Reynolds' Number

Sample Calculation of Reynolds' Numbers for Test Rig:

Dimensions:

Shaft Diameter = 6.995 cmBore Diameter = 7.005 cmAxial Length = 3.015 cmRadial Clearance (c) = 0.005 cmAbsolute Viscosity (μ) = $1.548 \text{ x } 10^{-6} \text{ N-s/cm}^2$ Kinematic Viscosity (ν) = $0.179 \text{ cm}^2/\text{s}$

$$Q = VA$$
; $Q = \frac{\Delta P D_s c^3}{12 \mu L}$; $A = \frac{\pi}{4} (D_{bore}^2 - D_s)$

Thus,

$$V = \frac{\Delta P D_s c^3}{3 \mu L (D_{bore}^2 - D_s^2)} = \frac{(689 \, kPa)(6.995 \, cm)(0.005 \, cm)^3}{3(1.548 x 10^{-6} \frac{N-s}{cm^2})(3.015 \, cm)(7.005^2 \, cm^2 - 6.995^2 \, cm^2)}$$

$$V = 30.7 \ \frac{cm}{s}$$

$$Re_{z} = \frac{2Vc}{v} = \frac{2(30.7 \frac{cm}{s})(0.005 cm)}{0.179 \frac{cm^{2}}{s}} = 1.72$$

For N = 12,000 RPM,

$$\omega = 12,000 \times \frac{\pi}{30} = 1257 \frac{rad}{s}$$

$$Re_{\theta} = \frac{Rc\omega}{v} = \frac{(\frac{7.005 \ cm}{2})(0.005 \ cm)(1257 \ \frac{rad}{s})}{0.179 \ \frac{cm^2}{s}} = 123$$

VITA

The author was born on January 9, 1967 in Alexandria, Virginia. He grew up in Woodbridge, Virginia and graduated from Potomac High School in 1985. That same year, the author began his studies at Virginia Tech, where he studied Mechanical Engineering. The author was a co-op student with IBM Corporation and graduated with his Bachelor of Science in Mechanical Engineering in 1990. He immediately began work on his Master's Degree as a research assistant under Dr. R. Gordon Kirk. He has currently accepted a position with Lord Corporation in Erie, Pennsylvania.

Aling W. Alat