

## Appendix A

### Grid independence and time step convergence:

#### Grid independence

The heat flux signal for a spatial step size of 0.25 mm can be seen below. See *figure A.1*. The flux does not change noticeably much with a change in the number of spatial steps.

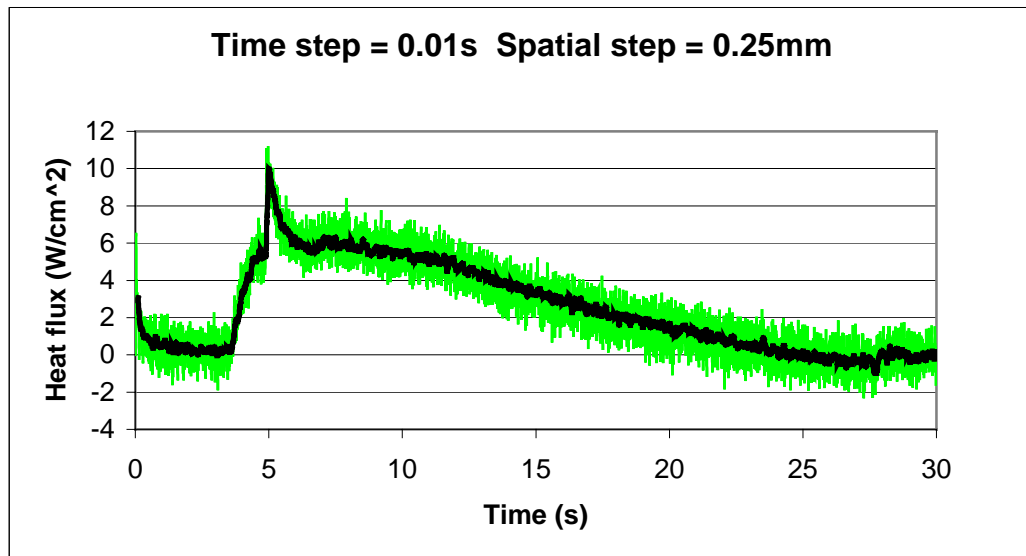


Figure A.1.

If the actual maximum of the flux is retrieved from the data and plotted against the number of spatial steps, the effect is obvious. See *figure A.2* for more detail.

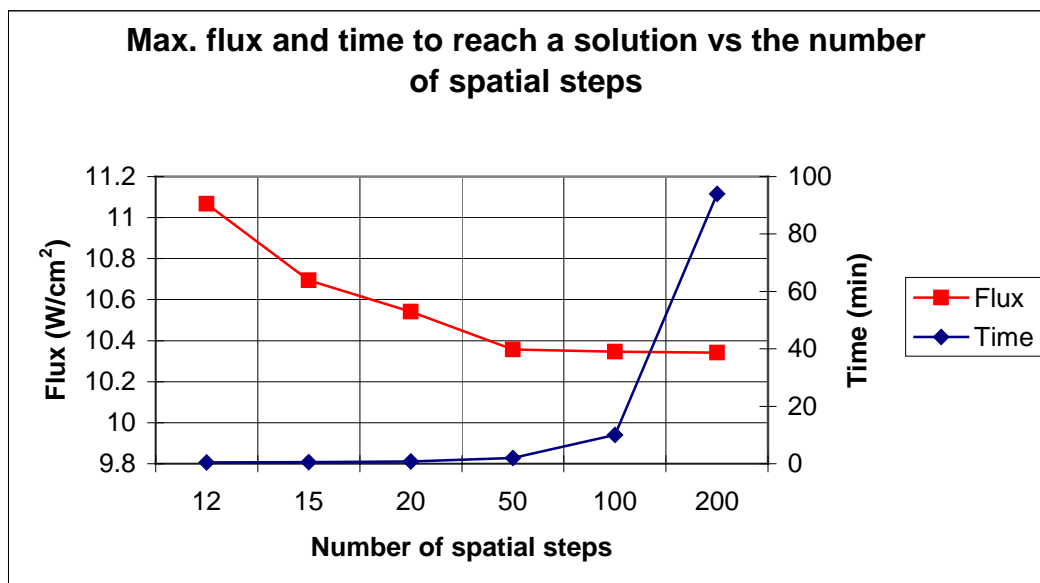
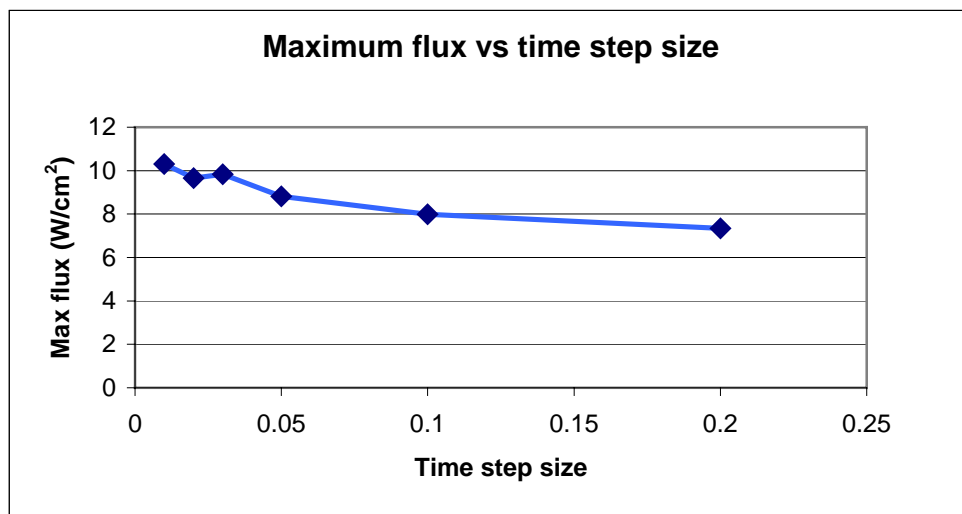


Figure A.2

In *figure A.2* the effect of the number of spatial steps on the computer time can also be seen. The time raise steeply to 94 min for this particular case when the number of spatial steps is increased from 100 to 200. Looking at the flux on the red line, it seems that the flux reached grid independence at about 50 spatial steps. That means 0.5 mm per step. To be a little more conservative a value of 100 steps were used. The value of 100 seemed realistic in the sense that convergence has been reached and the time for solving has not yet increased dramatically.

### Time step convergence



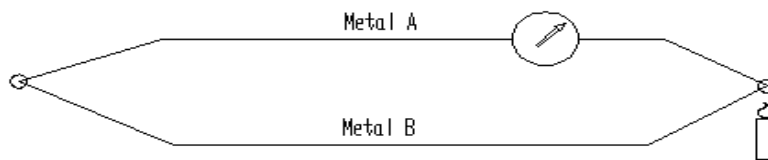
**Figure A.3**

Looking at the variance of the maximum flux with the change in the time step size the graph shown in *figure A.3* indicates that the maximum flux levels out in the region of 0.03s and lower. A realistic and conservative value of 0.01s was used in the test cases throughout the text. The flux will thus stay the same for smaller time steps and will not increase further as is the case when one moves from a time step size of 0.2s to 0.05s. The conclusion on this simple test is that a time step convergence was achieved with a step size of 0.01s.

## Appendix B

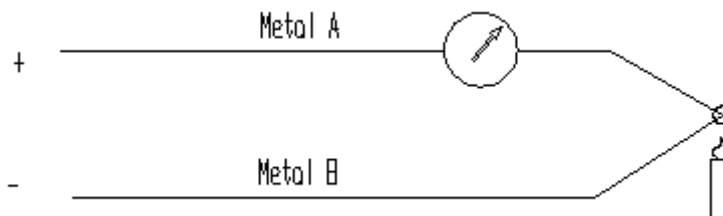
### Thermocouples:

Thomas Seebeck discovered in 1821 that if two wires composed of dissimilar metals are joined at both ends and one of the ends is heated, there is a continuous current that flows in the thermoelectric circuit.



**Figure B.1 Thermoelectric circuit**

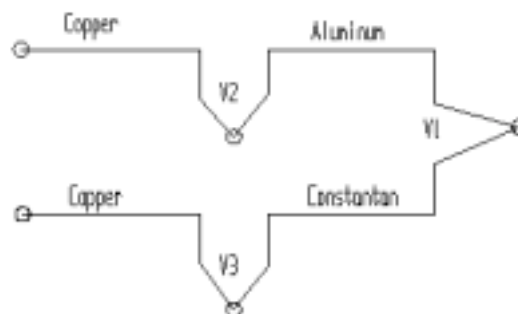
If the circuit is broken in the middle, then the open circuit voltage will be a function of the temperature at the junction and the properties of the two metals.



**Figure B.2 Open ended thermoelectric circuit**

If the voltage is to be measured, care should be taken. Connecting the voltmeter across the ends of the circuit creates two additional junctions which in turn are also thermocouples giving a current proportional to the temperature and the composition of the junctions.

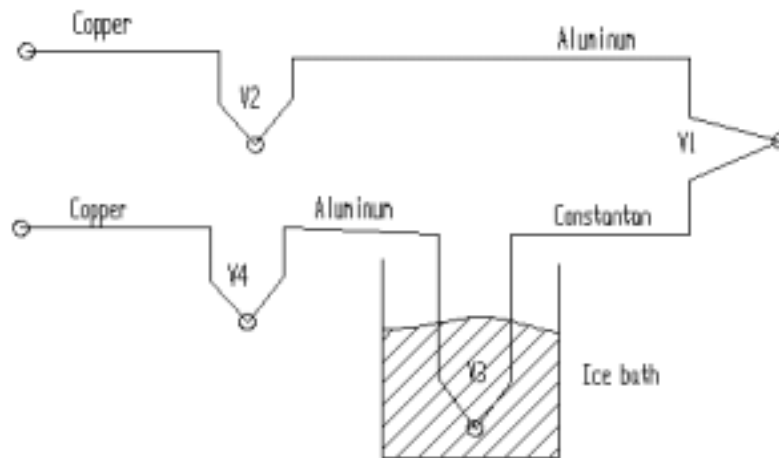
If the two metals are Aluminum and Constantan (as is the case in this study) and the voltage read is done with a voltmeter with copper internal wires, the circuit will look like this:



**Figure B.3 Circuit with three different metals**

There will thus be a voltage  $V_1$ ,  $V_2$  and  $V_3$ . The voltage read by the voltmeter will be a combination of the three and not just that of the junction at  $V_1$ . To overcome this problem a

reference point is used. In this case it is a  $0^{\circ}\text{C}$  ice bath— a mixture of ice and water. See the figure.



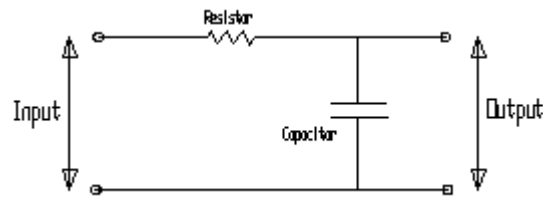
**Figure B.4 Circuit with an ice-bath**

If the temperatures at V2 and V4 are kept the same, the voltage created by these two junctions will cancel each other. The measured voltage by the voltmeter will be a function of the difference in the temperature between the junctions at V1 and V3. The fact that the reference temperature is  $0^{\circ}\text{C}$  makes it simpler in the sense that the voltage at V1 (in degrees Celsius) is now proportional to the temperature at junction V1.

## Appendix C

### **Resistor-capacitor low-pass filters:**

A combination of a resistor-capacitor in series can be connected as a low-pass filter. This will allow only low frequencies to pass the filter, while high frequency noise is bypassed to ground. See the figure for a schematic layout of a resistor-capacitor low-pass filter.

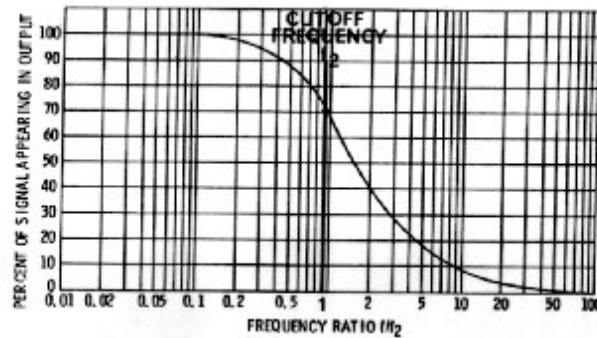


**Figure C.1 A simple filter circuit**

If the cutoff frequency is to be 50 Hz, which is a good choice, because most electrical noise is at 60 Hz, one needs to establish the sizes of the resistor and the capacitor needed for the circuit. Choosing a resistor of 30 k $\Omega$  the capacitance of the capacitor can be calculated with the following equation:

$$C = \frac{1}{2\pi fR}$$

The frequency  $f$  is called the high-frequency cutoff. Frequencies below the cutoff point are said to be in the passband of the filter, and frequencies above are in the rejection band of the filter. The characteristics of a resistor-capacitor filter are shown below. The vertical axis is the relative voltage or percent of the input voltage. The frequency  $f$  corresponds to 70.7 percent.



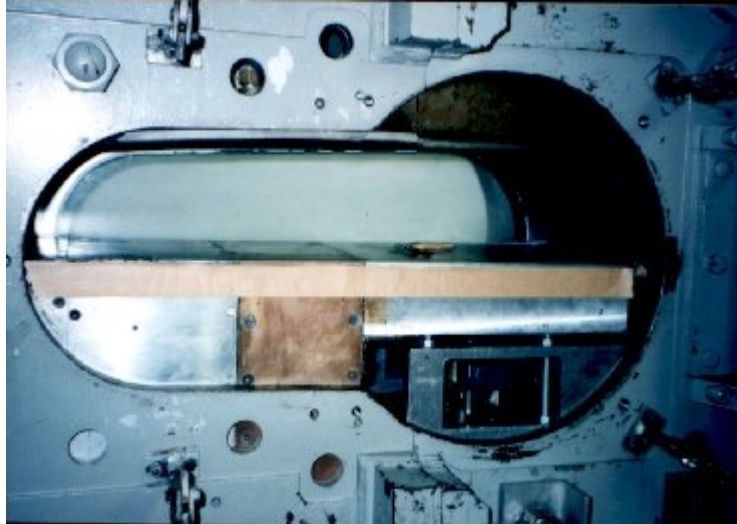
**Figure C.2 Cut-off frequency chart**

Using the above equation gives a capacitance of 106 pF. This is unfortunately not a standard capacitor capacitance. The closest will be a 100 pF capacitor. Using the same equation as above the cutoff frequency can be calculated to be 53.05 Hz. This is close to the desired value of 50 Hz.

## Appendix D

### **The supersonic wind tunnel**

The supersonic wind tunnel in which some of the experiments were conducted is shown for a little more clarity.



**Figure D.1 Supersonic tunnel test section**

In the first picture, the tunnel is shown with the side door off. The skin-friction gage was fixed in the door (not shown) at a position at a certain position not impinged by the shock.



**Figure D.2 Supersonic tunnel at Virginia Tech**

Next is an overview of the supersonic tunnel. Seen on the left top corner is the control valve. The air is fed into the settling-chamber and then through the nozzle and the exhaust to the outside of the laboratory.

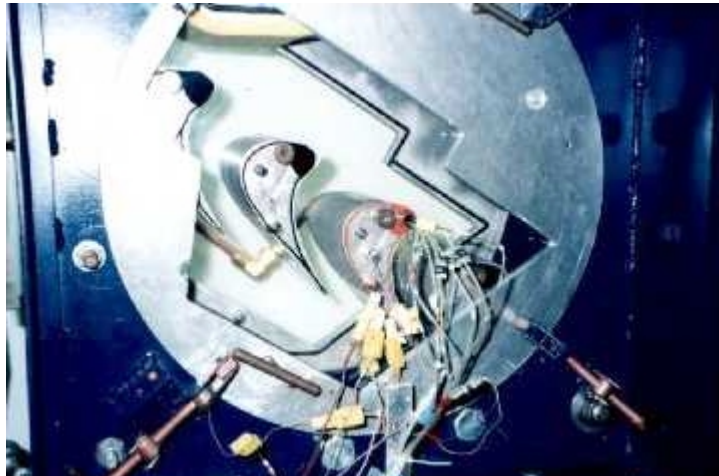
## The cascade tunnel

The most important experiments were done in the cascade tunnel. The next picture shows the tunnel and the “heater loop”. Looking at figure 4.1 will give more clarity on the picture angle.



**Figure D.3 Cascade tunnel**

The next picture shows the blades in the cascade.



**Figure D.4 Turbine blades in the cascade tunnel**

Here, the instrumented blade can be seen with the thermocouples, pressure sensors and the HFS heat flux gages in. Only one of the blades is instrumented at a time, and the others are just there to complete the cascade. The additional blades help retain the true flow of gas around the instrumented blade.

The picture below shows the blade from the other side with the coolant flow coming in through the copper pipe. This cool air ( $\pm -90^{\circ}\text{C}$ ) is then blown through the small holes on in



the blade surface. This cold air forms a thin film of cold air around the blade, protecting it from the hot air coming from the settling-chamber. From there the name film cooling.



**Figure D.5 The pipe with cooling air blowing into the blade.**

See web page: <http://www.aoe.vt.edu/aoe/physical/cascade.html>

## Appendix E

```

PROGRAM Implicitinverse
*   This program calculates the heat flux at the surface
*   of an object, given the surface temperature history.
*   It uses a implicit solver to solve the inverse of the
*   matrix of temperatures and incorporates this into the
*   inverse calculation of the surface flux.

INTEGER x, t, r
PARAMETER(x=100,t=2500,r=3)

*   x = the number of nodes in the region looked at.
*   t = the number of time steps for the run.
*   r = an index.

REAL*8 dx(x), dt, xt, v(r)
REAL*8 temp(x), temp1(x), tsurf(t)
REAL*8 bft(t), bft0(x), bft1(x)
REAL*8 nodetemp(r), nodetemp1(r), xx1(r,r), time(t), h(r,r)
REAL*8 a(r,r), c(r), p(t)
REAL*8 qguess(t), qsave, qb(t), qi(0:r)
REAL*8 penetra
REAL*8 h0(r,r), h1(r,r), h2(r,r)
REAL*8 w, w0, w1, w2
REAL*8 error1, error2
REAL*8 relax, sensativity, tollerance, one, alpha, d
INTEGER indx(r)
INTEGER nodes
INTEGER count, tpos, dqct, i, j

*   dx(x) is the spacial grid spacing between nodes. In
*   this case it will be a constant, but it can be a
*   function of the possition relative to the surface.
*   dt is the timestep size.
*   bft stands for the back face temperature, measured inside
*   the object.
*   time is the time
*   qsave saves the heat flux of every iteration for the next.
*   qguess is the guessed flux used as input to the code. The
*   initial guess is usually zero.
*   penetra is the penetration into the object. Also the distance
*   from the surface to the second/interior thermocouple.
*   w and relax is the relaxation factor.
*   error1 and error2 is for interation purposes.
*   relax is the relaxation factor.
*   count is just a counter.
*   i is a index.
*   j is a index.

c 'Enter the input file name'
c 'Enter the output file name '
      OPEN(5, FILE='Aprldata.prn')
      OPEN(12, FILE='Testflux.dat')

c   Read the time, the guessed flux, the surface temperature and the
c   back face temperature.

```

```

count = 1
1  READ(5,*,END=2) time(count), qguess(count), tsurf(count),
+   bft(count)
count = count + 1
GOTO 1

```

c Nodes is the number of nodes.  
2 nodes = count-1

```

alpha      = 1.0d-4
tolerance  = 1.0d-4
one        = 1.0
sensitivity = 0.00001
xt         = 0.0
w0         = 0.0
w1         = 1.0
w2         = 0.0
relax      = 1.0

```

c Assume that the penetration depth is 1 inch (25 mm)

```
penetra = 0.025
```

c Calculate the distance between the nodes. dfloat(x) is the  
c number of nodes

```

DO count = 1,x
dx(count) = penetra/DFLOAT(x)
ENDDO

```

c Save the measured temp in array "t"

```

DO count = 1,x
temp(count) = tsurf(1)
bft0(count) = bft(1)
ENDDO

```

```

DO i = 0,r
qi(i) = 0.0d0
ENDDO

```

```

DO count = 1,t
p(count) = one
ENDDO
p(t) = 1.0d-2

```

C---- Inverse parameters

```
INCLUDE 'Regularization.f'
```

```

DO i = 1,r
qi(i) = 1.0d-3
ENDDO

```

C---- Begin inverse loop section

```

WRITE(*,6) time(1),qguess(1),tsurf(1), bft(1), 0.0
WRITE(12,6) time(1),qguess(1),tsurf(1), bft(1), 0.0

```

```
DO 3 tpos = 2,nodes-r+1
```

```

c      dt is the time step. In this case it will stay the same throughout
c      the program.
      dt = time(tpos) - time(tpos-1)
      dqct = 0
      w = relax

```

#### C---- Iteration loop

```

4      CONTINUE
      dqct = dqct + 1

      DO count = 1,x
        temp1(count) = temp(count)
        bft1(count) = bft0(count)
      ENDDO

      CALL Implicit1(qb(tpos),bft(tpos),x,r,temp1,qi,dx,dt,nodetemp1,xt)

      DO j = 1,r
        DO count = 1,x
          temp1(count) = temp(count)
          bft1(count) = bft0(count)
        ENDDO
        qsave = qi(j)
        qi(j) = qsave*(one+sensitivity)+sensitivity

        CALL Implicit1(qb(tpos),bft(tpos),x,r,temp1,qi,dx,dt,nodetemp,xt)

        DO i = 1,r
          xx1(i,j) = (nodetemp(i) - nodetemp1(i))/(qi(j)-qsave)
        ENDDO

        qi(j) = qsave
      ENDDO

```

#### C---- Calculate left hand side (A matrix)

```

      DO i = 1,r
        DO j = 1,r
          a(i,j) = 0.0d0
          DO count = 1,r
            a(i,j) = a(i,j) + xx1(count,i)*p(tpos+count-1)
          +      *xx1(count,j)

          ENDDO
          a(i,j) = a(i,j) + h(i,j)
        ENDDO
      ENDDO

```

#### C---- Calculate temperature difference

```

      DO i = 1,r
        v(i) = tsurf(tpos+i-1) - nodetemp1(i)
      ENDDO

```

#### C---- Calculate right hand side (c vector)

```

      DO i = 1,r
        c(i) = 0.0d0

```

```

      DO count = 1,r
          c(i) = c(i) + xx1(count,i)*p(tpos+count-1)*v(count)
+      - h(i,count)*qi(count)

      ENDDO
ENDDO

```

C---- Solve the matrix equations ( $A*dq = c$ )

```

      CALL ludcmp(a,r,r,indx,d)
      CALL lubksb(a,r,r,indx,c)

```

C---- Calculate err (dq)

```

      error1 = 0.0
      error2 = 0.0
      DO i = 1,r
          error1 = error1 + ABS(c(i)/qi(i))
          error2 = error2 + ABS(c(i))
      ENDDO

```

C---- Update count and flux "guess"

```

      IF (dqct.gt.5) w = relax/10.0d0
      IF (dqct.gt.15) w = relax/30.0d0

      DO i = 1,r
          qi(i) = qi(i) + w*c(i)
      ENDDO

      IF (dqct.lt.40) THEN
      IF ((error1.gt.tolerance).and.(error2.gt.tolerance)) GOTO 4
      ENDIF

```

C---- Direct solution for correct flux and write out results

```

      CALL Implicit2(qb(tpos),bft(tpos),x,temp,qi(0),qi(1),dx,dt)

      WRITE(*,6) time(tpos),qguess(tpos),tsurf(tpos), bft(tpos), qi(1)
      WRITE(12,6) time(tpos),qguess(tpos), tsurf(tpos),bft(tpos),qi(1)

```

C---- Bump up flux "guess" for next time step

```

      DO i = 0,r-1
          qi(i) = qi(i+1)
      ENDDO

3      CONTINUE

5      FORMAT(1X,F10.7,F10.7,F12.4,I3,F12.4)
6      FORMAT(5(3X,E13.6))

      END

```

C Include impliter subroutine and property functions

```

      INCLUDE 'LUdecomp.f'
      INCLUDE 'Nodetemperatures.f'
      INCLUDE 'Alum.f'

```

**File Regularization**

```

DO i = 1,r
  DO j = 1,r
    h0(i,j) = 0.0d0
    h1(i,j) = 0.0d0
    h2(i,j) = 0.0d0
  ENDDO
ENDDO

DO count = 1,r
  h0(count,count) = 1.0d0
ENDDO
DO count = 1,r-1
  h1(count,count) = -1.0d0
  h1(count,count+1) = 1.0d0
ENDDO
DO count = 1,r-2
  h2(count,count) = 1.0d0
  h2(count,count+1) = -2.0d0
  h2(count,count+2) = 1.0d0
ENDDO

DO i = 1,r
  DO j = 1,r
    h(i,j) = 0.0d0
    DO count = 1,r
      h(i,j) = h(i,j) + w0*h0(count,i)*h0(count,j)
      h(i,j) = h(i,j) + w1*h1(count,i)*h1(count,j)
      h(i,j) = h(i,j) + w2*h2(count,i)*h2(count,j)
    ENDDO
    h(i,j) = alpha*h(i,j)
  ENDDO
ENDDO

```

**File LUdecomp**

```

SUBROUTINE ludcmp(a,n,np,indx,d)
  INTEGER n,np,indx(n),NMAX
  REAL*8 d,a(np,np),TINY
  PARAMETER (NMAX=500,TINY=1.0e-20)
  INTEGER i,imax,j,k
  REAL*8 aamax,dum,sum,vv(NMAX)
  d=1.0d0
  DO 12 i=1,n
    aamax=0.0d0
    DO 11 j=1,n
      IF (ABS(a(i,j)).gt.aamax) aamax=ABS(a(i,j))
11    CONTINUE
    IF (aamax.eq.0.0d0) PAUSE 'singular matrix in ludcmp'
    vv(i)=1.0d0/aamax
12  CONTINUE
  DO 19 j=1,n
    DO 14 i=1,j-1
      sum=a(i,j)
      DO 13 k=1,i-1
        sum=sum-a(i,k)*a(k,j)
13    CONTINUE
      a(i,j)=sum

```

```

14  CONTINUE
    aamax=0.0d0
    DO 16 i=j,n
      sum=a(i,j)
      DO 15 k=1,j-1
        sum=sum-a(i,k)*a(k,j)
15  CONTINUE
    a(i,j)=sum
    dum=vv(i)*ABS(sum)
    IF (dum.ge.aamax) THEN
      imax=i
      aamax=dum
    ENDIF
16  CONTINUE
    IF (j.ne.imax) THEN
      DO 17 k=1,n
        dum=a(imax,k)
        a(imax,k)=a(j,k)
        a(j,k)=dum
17  CONTINUE
      d=-d
      vv(imax)=vv(j)
    ENDIF
    indx(j)=imax
    IF(a(j,j).eq.0.0d0)a(j,j)=TINY
    IF(j.ne.n) THEN
      dum=1./a(j,j)
      DO 18 i=j+1,n
        a(i,j)=a(i,j)*dum
18  CONTINUE
    ENDIF
19  CONTINUE
    RETURN
    END

```

C (C) Copr. 1986-92 Numerical Recipes Software J!V%03#1y.

```

SUBROUTINE lubksb(a,n,np,indx,b)
INTEGER n,np,indx(n)
REAL*8 a(np,np),b(n)
INTEGER i,ii,j,ll
REAL*8 sum
ii=0
DO 12 i=1,n
  ll=indx(i)
  sum=b(ll)
  b(ll)=b(i)
  IF (ii.ne.0) THEN
    DO 11 j=ii,i-1
      sum=sum-a(i,j)*b(j)
11  CONTINUE
  ELSE IF (sum.ne.0.0d0) THEN
    ii=i
  ENDIF
  b(i)=sum
12 CONTINUE
DO 14 i=n,1,-1
  sum=b(i)
  DO 13 j=i+1,n
    sum=sum-a(i,j)*b(j)

```

```

13 CONTINUE
   b(i)=sum/a(i,i)
14 CONTINUE
   RETURN
   END
C (C) Copr. 1986-92 Numerical Recipes Software J!V%03#1y.

```

### File Implicitinverse

```

C*****
C SUBROUTINE Implicit1(qback,back,nx,ni,t,qv,dx,dti,ts,xt)
C*****
C Implicit routine to determine temperature history
C Thomas Algorithm
C Properties can vary with temperature

INTEGER nx, ni
REAL*8 t(nx), dx(nx), dti, qv(0:ni), ts(ni), xt,back
INTEGER nmax
REAL*8 tol,qback
PARAMETER (nmax=500,tol=1.0d-4)

INTEGER ct, ctr, ctil, tl, n
REAL*8 ix, xc, q
REAL*8 t1, dt
REAL*8 to(nmax), a(nmax), b(nmax), c(nmax), r(nmax)

REAL*8 dxr,dxl,dxp,tw,te
REAL*8 rc,rcr,rcl,kr,kl,for,fol

REAL*8 rho,cp,k,rtc

tl = INT(8.0d0*dti*k(t(1))/dx(1)/dx(1)/rho(t(1))/cp(t(1)))
dt = dti / DFLOAT(tl)

ix = 0
xc = 0.0d0
81 ix = ix + 1
xc = xc + dx(ix)
IF (xc.lt.xt) GOTO 81

DO 10 ctr = 1,ni

   DO 22 ctil = 1,tl

      n = 0
      q = qv(ctr-1) + ctil*(qv(ctr)-qv(ctr-1))/DFLOAT(tl)

c Save the previous temperature values
   DO ct = 1,nx
      to(ct) = t(ct)
   ENDDO

4 t1 = t(1)

```



C---Here the matrix to be solved is built.

INCLUDE 'Implicitinverse.f'

CALL tridag(a,b,c,r,t,nx,nx)

IF (n.eq.200) STOP 'temp dist not converging – Implicit1'  
n = n + 1

IF (ABS(t1-t(1))/t(1).gt.tol) GOTO 4

22 CONTINUE

ts(ctr) = t(ix+1) - (xc-xt)\*(t(ix+1)-t(ix))/dx(ix)

10 CONTINUE

RETURN  
END

C\*\*\*\*\*

SUBROUTINE Implicit2(qback,back,nx,t,q1,q2,dx,dti)

C\*\*\*\*\*

c Implicit routine to determine temperature distribution  
c Thomas Algorithm  
c Specified Flux at surface  
c Properties can vary with temperature

INTEGER nx, tpos  
REAL\*8 t(nx), dx(nx), dti, q1, q2, back,qback

INTEGER nmax  
REAL\*8 tol  
PARAMETER (nmax=500,tol=1.0d-4)

INTEGER ct, ct1l, tl, n  
REAL\*8 t1, dt  
REAL\*8 to(nmax), a(nmax), b(nmax), c(nmax), r(nmax)

REAL\*8 dxr,dx1,dxp,tw,te  
REAL\*8 rc,rcr,rcl,kr,kl,for,fol

REAL\*8 rho,cp,k,rtc

tl = INT(8.0d0\*dti\*k(t(1))/dx(1)/dx(1)/rho(t(1))/cp(t(1)))  
dt = dti / DFLOAT(tl)

DO 22 ct1l = 1,tl

n=0  
q = q1 + ct1l\*(q2-q1)/DFLOAT(tl)

c Save the previous temperature values

DO ct = 1,nx  
to(ct) = t(ct)  
ENDDO

4 t1 = t(1)

```

c   Here the matrix to be solved is built
    INCLUDE 'Implicitinverse.f'

    CALL tridag(a,b,c,r,t,nx,nx)

    IF (n.eq.200) STOP 'temp dist not converging - imp2'
    n = n + 1

    IF (ABS(t1-t(1))/t(1).gt.tol) GOTO 4
    
```

22 CONTINUE

```

RETURN
END
    
```

```

c*****
  SUBROUTINE tridag(a,b,c,r,u,n,nx)
c*****
    
```

```

    INTEGER n,NMAX
    REAL*8 a(n),b(n),c(n),r(n),u(n)
    PARAMETER (NMAX=500)
c   Solves
    INTEGER j
    REAL*8 bet,gam(NMAX)
    IF(b(1).eq.0.0d0) PAUSE 'tridag: rewrite equations'
    bet=b(1)
    u(1)=r(1)/bet
    DO 11 j = 2,nx
      gam(j) = c(j-1)/bet
      bet = b(j)-a(j)*gam(j)
      IF(bet.eq.0.0d0) PAUSE 'tridag failed'
      u(j) = (r(j)-a(j)*u(j-1))/bet
11  CONTINUE
    DO 12 j = nx-1,1,-1
      u(j) = u(j)-gam(j+1)*u(j+1)
12  CONTINUE
    RETURN
    END
    
```

**File Alum**

```

C*****
  REAL*8 FUNCTION k(t)
  REAL*8 t
*   k is the thermalconductivity
*   t is the timestep.

  k = 180

  RETURN
  END
    
```

```
C*****
  REAL*8 FUNCTION rtc(t)
  REAL*8 t

  rtc = 2700*896

  RETURN
  END
```

```
C*****
  REAL*8 FUNCTION cp(t)
  REAL*8 t
  *   cp specific heat constant.

  cp = 896

  RETURN
  END
```

```
C*****
  REAL*8 FUNCTION rho(t)
  REAL*8 t
  *   rho is the density.

  rho = 2700

  RETURN
  END
```

### File Implicitinverse

```
c   This section of program builds the conduction matrix.
c   It is then solved for every point in time. A semi-infinite
c   approach was used in the past, but this is improved with
c   the use of the temperature at the back face of the object
c   as the boundary condition.
```

```
c   Variables declared in the main program.
c   t0(x),t1(x) = Temperatures
c   dx(x)= Distance between nodes.
c   rho,cp,rtc,k = Properties.
c   a(x),b(x),c(x),r(x) = Matrix coefficients
c   q,qf,dt
```

```
c   Local variables
c   dxr,dx1,dxp,tw,te
c   rc,rcr,rcl,kr,kl,for,fol
```

```
c   Node one is the flux boundary
```

```
dxr = dx(1)
dxp = dxr/2.0d0
```

```
rc = rtc(t(1))
```

```
te = (t(1) + t(2))/2.0d0
kr = k(te)
```

$$\text{for} = \text{kr} * \text{dt} / (\text{rc} * \text{d xp} * \text{dxr})$$

$$\text{a}(1) = 0.0\text{d}0$$

$$\text{b}(1) = 1.0\text{d}0 + \text{for}$$

$$\text{c}(1) = -\text{for}$$

$$\text{r}(1) = \text{to}(1) + \text{q} * \text{dt} / (\text{rc} * \text{d xp})$$

**C Interior boundary at a specific temperature**

$$\text{a}(\text{nx}) = -0.5 * \text{for}$$

$$\text{b}(\text{nx}) = 1 + \text{for}$$

$$\text{c}(\text{nx}) = 0.0\text{d}0$$

$$\text{r}(\text{nx}) = (0.5 * \text{for} * \text{back}) + \text{t}(\text{nx}-1)$$

**c Interior boundary is insulated/adiabatic**

\*  $\text{a}(\text{nx}) = -1.0\text{d}0$

\*  $\text{b}(\text{nx}) = 1.0\text{d}0$

\*  $\text{c}(\text{nx}) = 0.0\text{d}0$

\*  $\text{r}(\text{nx}) = 0.0\text{d}0$

**DO** 1 ct = 2, nx-1

$$\text{dxr} = \text{dx}(\text{ct})$$

$$\text{dxl} = \text{dx}(\text{ct}-1)$$

$$\text{d xp} = (\text{dxr} + \text{dxl}) / 2.0\text{d}0$$

$$\text{rcr} = \text{rtc}(\text{t}(\text{ct}))$$

$$\text{rc l} = \text{rtc}(\text{t}(\text{ct}))$$

$$\text{rc} = (\text{rc l} * \text{dxl} + \text{rcr} * \text{dxr}) / (\text{dxr} + \text{dxl})$$

$$\text{te} = (\text{t}(\text{ct}) + \text{t}(\text{ct}+1)) / 2.0\text{d}0$$

$$\text{kr} = \text{k}(\text{te})$$

$$\text{tw} = (\text{t}(\text{ct}) + \text{t}(\text{ct}-1)) / 2.0\text{d}0$$

$$\text{kl} = \text{k}(\text{tw})$$

$$\text{for} = \text{kr} * \text{dt} / (\text{rc} * \text{dxr} * \text{d xp})$$

$$\text{fol} = \text{kl} * \text{dt} / (\text{rc} * \text{dxl} * \text{d xp})$$

$$\text{a}(\text{ct}) = -\text{fol}$$

$$\text{b}(\text{ct}) = 1.0\text{d}0 + \text{fol} + \text{for}$$

$$\text{c}(\text{ct}) = -\text{for}$$

$$\text{r}(\text{ct}) = \text{to}(\text{ct})$$

1 **CONTINUE**