

**Transient Motion Control of Passive and
Semiactive Damping for Vehicle Suspensions**

by

Angela K. Carter

Thesis submitted to the Faculty of the
Virginia Polytechnic Institute and State University
in partial fulfillment of the requirements for the degree of

Master of Science

in

Electrical Engineering

APPROVED:

Mehdi Ahmadian, Co-Chairman

William T. Baumann, Co-Chairman

Hugh F. Van Landingham

July 15, 1998

Blacksburg, Virginia

Keywords: Fuzzy Logic, Groundhook, Passive Damping,
Semiactive Damping, Skyhook, Vehicle Suspension

Transient Motion Control of Passive and Semiactive Damping for Vehicle Suspensions

by

Angela K. Carter

Mehdi Ahmadian, Co-Chairman

William T. Baumann, Co-Chairman

Electrical Engineering

This research will compare the transient response characteristics of a four-degree-of-freedom, roll-plane model, representing a class 8 truck, using passive and semiactive dampers. The semiactive damper control policies that are examined include the previously developed policies of on-off skyhook, continuous skyhook, and on-off groundhook control, along with a newly developed method of fuzzy logic semiactive control. The model input will include body forces and torques, as well as transient displacements at the tires. The model outputs include the vehicle body heave and roll displacements, the vertical displacement of the tire (wheel hop) and the vertical acceleration of the vehicle body. For each output, the maximum peak-to-peak and RMS values of the response are examined.

The results of the study show that semiactive dampers have minimal effect on improving the vehicle body and tire transients due to forces or torques applied to the body, as compared to passive dampers. For road inputs, however, semiactive dampers are able to provide a more favorable compromise between the body and axle transient dynamics, when compared to passive dampers. The fuzzy logic semiactive control policy that is proposed in this research is better able to balance the body and axle dynamics than the conventional semiactive damping control policies that are investigated. Further research on the application of fuzzy logic semiactive control concepts is suggested, in order to fully investigate the potential of such control schemes for vehicle suspensions.

Acknowledgments

I would like to thank Dr. Mehdi Ahmadian for his assistance and advice regarding this research, and for the use of the Advanced Vehicle Dynamics Laboratory (AVDL). The support provided by the Department of Mechanical Engineering, Center for Transportation Research (CTR) at Virginia Tech, and other corporate sponsors of AVDL, including Lord Corporation and Volvo Heavy Trucks, is also greatly acknowledged.

I would like to thank Dr. William T. Baumann for his recommendations concerning my program of study, Dr. Hugh F. Van Landingham for serving on my graduate committee, and Xubin Song, PhD. Candidate, for his assistance with some of the simulation programming techniques used in this research. I am also thankful to the Department of Electrical and Computer Engineering for the financial support of a graduate teaching assistantship.

Most importantly, I would like to thank my parents, William and Sheila Carter, along with Tony, Aaron, Janet, Staci, Andrew, Caroline, and Katie Carter, for their ever continuing support and understanding.

Contents

1.0 Introduction.....	1
1.1 Objectives.....	1
1.2 Approach.....	1
1.3 Outline.....	2
2.0 Background.....	4
2.1 Overview of Vehicle Suspension Damping.....	4
2.2 Passive Damping.....	6
2.3 Semiactive Damping.....	6
2.3.1 On-Off Skyhook Control.....	7
2.3.2 Continuous Skyhook Control.....	10
2.3.3 On-Off Groundhook Control.....	11
2.4 Fuzzy Logic Control.....	13
2.4.1 Step One: Fuzzification.....	13
2.4.2 Step Two: Execution of Rules.....	15
2.4.3 Step Three: Defuzzification.....	18
2.5 Literature Review.....	19
2.5.1 Skyhook-Based Control Literature.....	19
2.5.2 Fuzzy Logic Control Literature.....	22
3.0 Model Formulation	25
3.1 Roll-Plane Vehicle Model.....	25
3.2 Semiactive Damping Control Models.....	29
3.2.1 On-Off Skyhook Control.....	29
3.2.2 Continuous Skyhook Control.....	30
3.2.3 On-Off Groundhook Control.....	31
3.3 Fuzzy Logic Control.....	32
3.3.1 Step One: Fuzzification.....	32
3.3.2 Step Two: Execution of Rules.....	35
3.3.3 Step Three: Defuzzification.....	39

4.0 Computer Simulation.....	41
4.1 Main Program Body.....	41
4.2 Program Subroutines.....	44
4.2.1 On-Off Skyhook Control.....	45
4.2.2 Continuous Skyhook Control.....	46
4.2.3 On-Off Groundhook Control.....	48
4.2.4 Fuzzy Logic Control.....	50
4.3 Simulation Validation.....	53
4.3.1 Formulation of the Uncoupled System.....	53
4.3.2 Derivation of the Theoretical and Actual Response Parameters.....	55
4.3.3 Comparison of the Theoretical and Actual Response Parameters.....	57
5.0 Results.....	61
5.1 Program Parameter Constants.....	61
5.1.1 Program Main Body Constants.....	61
5.1.2 Program Subroutine Constants.....	62
5.2 Simulation Inputs.....	64
5.3 Simulation Outputs.....	66
5.4 Analysis of Simulation Results.....	68
5.4.1 Body Heave Input Results.....	69
5.4.2 Body Roll Input Results.....	72
5.4.3 Road Input Results.....	75
5.5 Additional Fuzzy Logic Control Results.....	92
6.0 Conclusions.....	100
6.1 Summary.....	100
6.2 Recommendations for Future Research.....	101
References.....	103
Vita.....	106

List of Figures

Figure 2.1	Passive and Semiactive Damper Curves.....	5
Figure 2.2	Passive Damping.....	6
Figure 2.3	Quarter-Car Model.....	7
Figure 2.4	Semiactive Damping - On-Off Control.....	8
Figure 2.5	On-Off Skyhook Control Illustration.....	9
Figure 2.6	Passive Damping Representation of Skyhook Control.....	10
Figure 2.7	Semiactive Damping - Continuous Control.....	10
Figure 2.8	On-Off Groundhook Control Illustration.....	12
Figure 2.9	Passive Damping Representation of Groundhook Control.....	13
Figure 2.10	Triangular-Shaped Input Membership Function Example.....	14
Figure 2.11	Input Intersection with Memberships.....	14
Figure 2.12	Triangular-Shaped Output Membership Function Example.....	15
Figure 2.13	Rule Table Example.....	16
Figure 2.14	Models Used in Karnopp, Crosby, and Harwood Study [4].....	19
Figure 3.1	Roll-Plane Vehicle Model.....	25
Figure 3.2	Trapezoidal-Shaped Membership Function Definition.....	33
Figure 3.3	Left Damper - Controller Input Membership Functions.....	34
Figure 3.4	Right Damper - Controller Input Membership Functions.....	35
Figure 3.5	Controller Output Membership Functions.....	36
Figure 3.6	Left Damper - Controller Rule-Base.....	37
Figure 3.7	Right Damper - Controller Rule-Base.....	38
Figure 4.1	Flow Chart for the Roll-Plane Simulation Model.....	42
Figure 4.2	Flow Chart for On-Off Skyhook Control Policy Subroutine.....	45
Figure 4.3	Flow Chart for Continuous Skyhook Control Policy Subroutine.....	47
Figure 4.4	Flow Chart for On-Off Groundhook Control Policy Subroutine.....	49
Figure 4.5	Flow Chart for Fuzzy Logic Control Policy Subroutine.....	51
Figure 4.6	Similar Triangle Solution Example.....	52
Figure 4.7	Step Input Response of a Damped System.....	56

Figure 4.8	Theoretical and Actual Response to a Step Input.....	58
Figure 5.1	Impulse Profile for a Force Input.....	65
Figure 5.2	Transient Response Example.....	66
Figure 5.3	Maximum Peak-to-Peak Value of a Transient Response.....	68
Figure 5.4	Effect of High-State Damping on Controlling the Vehicle Response Due to a Body Heave Input.....	70
Figure 5.5	Effect of Low-State Damping on Controlling the Vehicle Response Due to a Body Heave Input.....	71
Figure 5.6	Effect of High-State Damping on Controlling the Vehicle Response Due to a Body Roll Input.....	73
Figure 5.7	Effect of Low-State Damping on Controlling the Vehicle Response Due to a Body Roll Input.....	74
Figure 5.8	Effect of High-State Damping on Controlling the Vehicle Response Due to a Road Input at the Left Tire.....	76
Figure 5.9	Effect of Low-State Damping on Controlling the Vehicle Response Due to a Road Input at the Left Tire.....	77
Figure 5.10	Normalized Semiactive Control Response Results for Values Equivalent to the Maximum Peak-to-Peak, Body Heave Displacement, Passive Damping Result.....	81
Figure 5.11	Normalized Semiactive Control Response Results for Values Equivalent to the Maximum Peak-to-Peak, Body Heave Acceleration, Passive Damping Result.....	82
Figure 5.12	Normalized Semiactive Control Response Results for Values Equivalent to the Maximum Peak-to-Peak, Body Roll Displacement, Passive Damping Result.....	83
Figure 5.13	Normalized Semiactive Control Response Results for Values Equivalent to the Maximum Peak-to-Peak, Left Tire Heave Displacement, Passive Damping Result.....	84
Figure 5.14	Normalized Semiactive Control Response Results for Values Equivalent	

	to the RMS, Body Heave Displacement, Passive Damping Result.....	85
Figure 5.15	Normalized Semiactive Control Response Results for Values Equivalent to the RMS, Body Heave Acceleration, Passive Damping Result.....	85
Figure 5.16	Normalized Semiactive Control Response Results for Values Equivalent to the RMS, Body Roll Displacement, Passive Damping Result.....	86
Figure 5.17	Normalized Semiactive Control Response Results for Values Equivalent to the RMS, Left Tire Heave Displacement, Passive Damping Result...	86
Figure 5.18	Normalized Semiactive Control Response Results for Minimum Values of the Maximum Peak-to-Peak, Body Heave Displacement.....	88
Figure 5.19	Normalized Semiactive Control Response Results for Minimum Values of the Maximum Peak-to-Peak, Body Heave Acceleration.....	88
Figure 5.20	Normalized Semiactive Control Response Results for Minimum Values of the Maximum Peak-to-Peak, Body Roll Displacement.....	89
Figure 5.21	Normalized Semiactive Control Response Results for Minimum Values of the Maximum Peak-to-Peak, Left Tire Heave Displacement.....	89
Figure 5.22	Normalized Semiactive Control Response Results for Minimum Values of the RMS, Body Heave Displacement.....	90
Figure 5.23	Normalized Semiactive Control Response Results for Minimum Values of the RMS, Body Heave Acceleration.....	90
Figure 5.24	Normalized Semiactive Control Response Results for Minimum Values of the RMS, Body Roll Displacement.....	91
Figure 5.25	Normalized Semiactive Control Response Results for Minimum Values of the RMS, Left Tire Heave Displacement.....	91
Figure 5.26	Effect of On-Off Skyhook and Fuzzy Logic Semiactive High-State Damping on Controlling the Vehicle Response Due to a Road Input at the Left Tire.....	94
Figure 5.27	Effect of On-Off Skyhook and Fuzzy Logic Semiactive Low-State Damping on Controlling the Vehicle Response Due to a Road Input at the Left Tire.....	95
Figure 5.28	Effect of On-Off Groundhook and Fuzzy Logic Semiactive	

	High-State Damping on Controlling the Vehicle Response Due to a Road Input at the Left Tire.....	97
Figure 5.29	Effect of On-Off Groundhook and Fuzzy Logic Semiactive Low-State Damping on Controlling the Vehicle Response Due to a Road Input at the Left Tire.....	98

List of Tables

Table 3.1	Roll-Plane Vehicle Model Parameter Definition.....	26
Table 3.2	Roll-Plane Vehicle Model States.....	27
Table 4.1	Vehicle Parameters Used for Response Comparison.....	57
Table 4.2	Comparison of the Theoretical and Actual Response Characteristics.....	59
Table 5.1	Roll-Plane Vehicle Model Parameter Constants.....	62
Table 5.2	Fuzzy Logic Input Membership Constants.....	62
Table 5.3	Minimum Characteristics for Passive Damping.....	80